DRAFT FINAL REPORT

# EXPERIMENTAL AND ANALYTICAL EVALUATION OF FLEXIBLE PIPES FOR CULVERTS AND STORM SEWERS <br> VOLUME II - LABORATORY WORK 

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## SUMMARY

This volume is the second of the four-volume-report on the study entitled, "Experimental and Analytical Evaluation of Flexible Pipes for Culverts and Storm Sewers". It describes the laboratory work performed and presents results for ten different tests carried out in this study. The main objective of the laboratory work was to evaluate .and characterize, under laboratory conditions, the performance and properties of the different plastic and metal pipes considered in the study.

Visual Inspections of the different pipes indicated that HDPE, PVC, and metal pipes generally meet the requirements of AASHTO-M294, ASTM F949, and ASSHTO-T249. However, visible creasing at the surface of inside and outside walls, as well as irregular surface, at certain locations around, the circumference of the bell and spigot joint, were observed in ADS 48. Also the contact length of the seam lap in the case of, aluminum and its distance from the adjacent ribs for both types of metal pipes do not conform to AASHTO T249 requirements. These irregularities, even though they seem not to have an apparent incidence on structural performance, may require improvement.

Beam Test results show that for the plastic pipes the valley longitudinal bending strains were greater than the crown longitudinal bending strains. For the metal pipes, the longitudinal bending strains in the ribs were greater than the longitudinal bending strains in the wall (valley) between the ribs.

Parallel Plate. Test results indicated that for $5 \%$ vertical deflection and a loading rate of 0.5 in./min., all the pipes achieved a pipe stiffness, PS, greater than the minimum specified by the Standards. They also revealed no sign of distress or buckling in the pipes for vertical deflections less than $15 \%$. For a given vertical deflection, the HDPE pipe stiffness (PS) substantially decreased as the loading rate decreased and vice-versa.

Flattening Test results indicated that all the HDPE pipes passed the test, since no splitting, cracking, breaking, or separation of ribs or seams, or both, were observed under normal light with unaided eyes. The PVC specimens that could be flattened up to $60 \%$ vertical deflection without failure also passed the flattening test. However, a number of PVC pipe specimens ruptured before reaching the $60 \%$ limit.

Curved Beam Test results indicated that time-independent pipe stiffness is 2 to 3 times greater than the PS values determined by the parallel plate test for all the pipes and increase with the loading rate for HDPE pipes.

Joint Integrity Test results indicated that all the pipes exhibited no sign of cracks or excessive gaps up to $10 \%$ vertical deflection. The presence of a joint generally modified the PS of the pipe.

Type C tension tests (small dog bone with no welds) indicated that he tensile properties of the pipes, the modulus of elasticity and the tensile strength, are within the range of values specified by the AASHTO code. Type A tension tests (double wall Dumbbell shape), performed on ADS 48 only, underestimated the tensile strength of the D-wall-type pipes such as ADS 48. Type B tension tests (single wall Dumbbell shape) indicated that the seam behavior of the D-wall-type pipe under tensile stresses is satisfactory given the maximum strength achieved. Type D tension tests (split disk test) performed on all the pipes indicated that the apparent tensile properties under split disk tests are lower than those under Type $G$ tension tests on small dog bone specimen with no weld, but greater than those achieved on dumbbell shape specimens with welds for ADS 48.

ESCR Tests performed on HDPE pipes indicated that the 36 inch-diameter HDPE pipes behaved satisfactorily under ESCR tests. For the 48 in-diameter HDPE pipe however, one of the two specimens failed the ESCR test under the conditions described in this study.

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## TABLE OF CONVERSIONS

| To convert from | To | Multiply by |
| :---: | :---: | :---: |
| Length |  |  |
| Inch (in.) | Millimeter (mm) | 25.4 |
| Foot (ft) | Meter (m) | 0.3048 |
| Area |  |  |
| Square inch (sq. in.) | Square millimeter (sq. mm) | 645.2 |
| Square foot (sq. ft.) | Square meter (sq. m) | 0.0926 |
| Volume |  |  |
| Cubic inch (cu. in.) 0.00001639 | Cubic meter (cu. m) |  |
| Cubic foot (cu. ft.) | Cubic meter (cu. m) | 0.02832 |
| Cubic yard (cu. yd.) | Cubic meter (cu. m) | 0.7646 |
| Gallon (gal) | Liter | 3.785 |
| Force |  |  |
| Kip | Kilogram (kgf) | 453.6 |
| Kip | Newton (N) | 4448.0 |
| Pound (lb) | Newton (N) | 4.448 |
| Pressure or Stress |  |  |
| Kip/square inch (ksi) | Megapascal (MPa)** | 6.895 |
| Pound/square inch (psi) | Megapascal (MPa)** | 0.006895 |
| **One Pascal equals one newton/square meter |  |  |
| Mass |  |  |
| Pound | Kilogram (kg) | 0.4536 |
| Ton (short, 2000 lb ) | Kilogram (kg) | 907.2 |
| Mass (weight per length) |  |  |
| Kip/linear foot (klf) | Kilogram/meter (kg/m) | 0.001488 |
| Pound/linear foot (plf) | Kilogram/meter ( $\mathrm{kg} / \mathrm{m}$ ) | 1.488 |
| Pound/linear foot (plf) | Newton/meter ( $\mathrm{N} / \mathrm{m}$ ) | 4.593 |

## Chapter 1: Introduction

### 1.1 General

This volume is the second of a four-volume-report on an extensive experimental and analytical investigation of flexible pipes entitled: Experimental and Analytical Evaluation of Flexible Pipes for Culverts and Storm Sewers. This Volume II, presents results of the laboratory work carried out in this study on the six different types of pipe considered. It also describes the experimental results for the ten different tests carried out in this study. The related specimen preparations, testing procedures, and relevant ASTM and AASHTO Standards, are also presented.

### 1.2 Objective

The main objective of this part of the research study was to evaluate and characterize under laboratory conditions the performance and properties of the different plastic and metal pipes considered in the study.

### 1.3 Organization of Volume II

This report contains eleven chapters, in addition to the introductory chapter. Chapter 2 presents results of the visual inspection and measurements of the different pipes. Chapter 3 presents results of the simple beam tests performed on the pipes. Chapter 4 is dedicated to the parallel plate loading tests, while Chapter 5 presents results of flattening tests. The curved beam tests are presented in Chapter 6 and the joint integrity tests in Chapter 7. Results of tension tests are presented in Chapters 8, 9, and 10, respectively, for the dumbbell shape 28 inch-specimens with welds, for the 48 inch diameter D-wall-type full ring pipe specimens, and for the 10 inch-dog bone-shaped-specimens with no welds. Chapter 11 presents results of the environmental stress cracking tests performed on HDPE pipes. Concluding remarks related to the laboratory work undertaken in this part of the study are provided in Chapter 12.

## Chapter 2: Visual Inspection and Measurements

### 2.1 Objectives

The objectives of this chapter are: (a) to present the measurements and geometry of the different pipes considered in this study, and: (b) to presents the results of visual inspections carried out on the pipes and the joints according to relevant AASHTO and ASTM Standards.

### 2.2 Geometry of pipes

The measurements of the different; pipes used in this project are presented in Table 2.1 to Table 2.6 as follows:

ADS 48" (HDPE)
ADS 36" (HDPE)
HANCOR 36" (HDPE)
PVC 36" (PVC)
ALUMINUM 36"
STEEL 36"

Table 2.1
Table 2.2
Table 2.3
Table 2.4
Table 2.5
Table 2.6

In particular, the following dimensions are provided: the inside diameter (ID), the outer diameter (OD), the thickness of the walls, and the dimensions and thickness of the corrugations. For each dimension, the average of eight readings is given. In addition, for PVC and HDPE pipes the geometry of the joint (i.e. the bell and the spigot) is also provided in Fig 2.1 (ADS 48), Fig. 2.2 (ADS 36), Fig. 2.3 (RANCOR 36) and Fig. 2.4. (PVC 36).

### 2.3 Visual Inspection

Visual inspections were carried out on the different pipes according to the following AASHTO and ASTM standards as follows:

- AASHTO M294 for HDPE pipes (ADS 48, ADS 36 and RANCOR 36)
- ASTM F 949 for PVC pipes
- AASHTO T 249 for steel and aluminum pipes

Results of these visual inspections and observations are presented in Tables 2.7 to 2.12 and Figs. 2.5 to 2.15 as follows:
ADS 48" (HDPE)
ADS 36" (HDPE)
HANCOR 36" (HDPE) Table 2.9 and Figs. 2.10 to 2.11
PVC 36" (PVC)
ALUNQNUM 36"
STEEL 36"

Table 2.7 and Figs. 2.5 to 2.7
Table 2.8 and Figs. 2.8 to 2.9

Table 2.10 and Figs. 2.12 to 2.13
Table 2.11 and Fig. 2.14
Table 2.12 and Fig. 2.15

### 2.4 Conclusions

Visual inspections of the different pipes indicated the followings
(a) $H D P E$ ADS 48: Generally, the pipe meets the AASHTO-M294 requirements for visual inspection. However, the surfaces of the inside and the outside walls revealed visible creasing. Also, the bell and spigot joint showed irregular surfaces at certain locations around the circumference.
(b) HDPE ADS 36: The pipe meets the AASHTO-M294 requirements for visual inspection.
(c) HDPE HANCOR 36: The pipe meets the AASHTO-M294 requirements for visual inspection.
(d) PVC 36: The pipe meets the ASTM F949 requirements for visual inspection.
(e) Aluminum 36: Generally the pipe meets the requirements of the ASSHTO-T249 for visual inspection. However, the seam lap is smaller than the minimum required length and is not equidistant from adjacent ribs as required. $m$ addition, the lapped surfaces are not quite in tight contact as required.
(f) Steel 36: Generally the pipe meets the requirements of the AASHTO T249 for visual inspection. However, the seam lap is not equidistant from adjacent ribs as required.

Table 2.1-Geometry of ADS 48" Pipe

| Test <br> No. | OD <br> (in) | ID (in) | $\mathbf{t}_{\mathbf{a}}$ (in) | $\mathbf{t}_{\text {in }}$ (in) | $\mathbf{t}_{\text {out }}$ (in) | $\mathbf{d}_{\mathbf{i}}$ (in) | $\mathbf{t}_{\mathbf{b}}$ (in) | $\mathbf{t}_{\mathbf{c}}$ (in) | $\mathbf{t}_{\mathbf{d}}$ (in) |  |
| ---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | $521 / 4$ | 47 | $1 / 8$ | 2.6515 | 0.1430 | 0.1050 | 2.4420 | 0.1290 | 0.0930 | 0.3140 |
| 2 | 52 | 47 | 2.6670 | 0.1435 | 0.1260 | 2.5285 | 0.1345 | 0.0945 | 0.3090 |  |
| 3 | $527 / 16$ | 47 | 2.6800 | 0.1250 | 0.1665 | 2.5120 | 0.1265 | 0.1025 | 0.3075 |  |
| 4 | $523 / 16$ | 47 | 2.6535 | 0.1395 | 0.1495 | 2.4750 | 0.1105 | 0.0845 | 0.3040 |  |
| 5 | $52_{3 / 8}$ | $473 / 16$ | 2.6375 | 0.1395 | 0.1005 | 2.5365 | 0.1130 | 0.0970 | 0.2995 |  |
| 6 | $52_{1 / 2}$ | $4715 / 16$ | 2.6765 | 0.1315 | 0.1065 | 2.4460 | 0.1380 | 0.0865 | 0.3025 |  |
| 7 | $527 / 16$ | $4615 / 16$ | 2.6475 | 0.1345 | 0.0980 | 2.5375 | 0.1335 | 0.0965 | 0.3090 |  |
| 8 | $523 / 8$ | $4615 / 16$ | 2.6685 | 0.1245 | 0.1050 | 2.5380 | 0.1045 | 0.0810 | 0.2975 |  |
| Average | 52.32 | 47.016 | 2.6603 | 0.1351 | 0.1196 | 2.5019 | 0.1237 | 0.0919 | 0.3054 |  |



Table 2.2-Geometry of ADS $36^{\prime \prime}$ Pipe

| Test <br> No. | ID (in) | OD (in) | $t_{\text {a }}$ (in) | tin (in) | $\mathrm{t}_{\text {out }}$ (in) | $t_{b}$ (in) | $\mathrm{t}_{\mathrm{c}}$ (in) | $\mathrm{d}_{1}$ (in) | $\mathrm{d}_{2}$ (in) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 35.813 | 41.375 | 2.6895 | 0.1245 | 0.1650 | 0.2265 | 0.2525 | 2.5175 | 2.2835 |
| 2 | 36.0 | 41.563 | 2.6980 | 0.1220 | 0.1760 | 0.2065 | 0.2560 | 2.5070 | 2,2280 |
| 3 | 36.125 | 41.688 | 2.6845 | 0.1265 | 0.1760 | 0.2125 | 0.2585 | 2.4860 | 2.1830 |
| 4 | 36.0 | 41.688 | 2.6950 | 0.1225 | 0.1760 | 0.2050 | 0.2605 | 2.4750 | 2.2460 |
| 5 | 35.875 | 41.5 | 2.6705 | 0.1230 | 0.1780 | 0.2165 | 0.2440 | 2.5020 | 2.2105 |
| 6 | 35.938 | 41.375 | 2.6690 | 0.1285 | 0.1755 | 0.2090 | 0.2635 | 2.5500 | 2.2400 |
| 7 | 36.063 | 41.625 | 2.6575 | 0.1205 | 0.1780 | 0.2155 | 0.2570 | 2.4800 | 2.1995 |
| 8 | 36.188 | 41.75 | 2.6570 | 0.1325 | 0.1730 | 0.2070 | 0.2630 | 2.5015 | 2.2455 |
| Average | 36.00 | 41.5703 | 2.6776 | 0.1250 | 0.1744 | 0.2123 | 0.2569 | 2.5024 | 2.2285 |



Table 2.3 - Geometry of HANCOR 36" Pipe

| Test <br> No. | OD <br> (in) | ID (in) | $\mathbf{t}_{\mathbf{a}}$ (in) | $\mathbf{t}_{\mathbf{a}}{ }^{\prime}$ (in) | $\mathbf{t}_{\mathbf{b}}$ (in) | $\mathbf{t}_{\mathbf{c}}$ (in) | $\mathbf{t}_{\text {in }}$ (in) | $\mathbf{t}_{\text {out }}$ (in) | $\mathbf{d}_{\mathbf{1}}$ (in) | $\mathbf{d}_{\mathbf{2}}$ (in) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 41.5 | 36.063 | 2.6430 | 2.4815 | 0.1575 | 0.2685 | 0.1215 | 0.0905 | 2.3165 | 2.1860 |
| 2 | 41.375 | 35.875 | 2.6405 | 2.5035 | 0.1520 | 0.2635 | 0.1190 | 0.1165 | 2.3505 | 2.1505 |
| 3 | 41.375 | 35.938 | 2.6225 | 2.5040 | 0.1695 | 0.3030 | 0.1145 | 0.0945 | 2.3270 | 2.1855 |
| 4 | 41.5 | 36.063 | 2.6630 | 2.5060 | 0.1475 | 0.3080 | 0.1395 | 0.1010 | 2.2675 | 2.1705 |
| 5 | 41.563 | 36.063 | 2.6655 | 2.5370 | 0.1690 | 0.2590 | 0.1230 | 0.0900 | 2.3135 | 2.1425 |
| 6 | 41.625 | 36.0 | 2.6935 | 2.5210 | 0.1760 | 0.2560 | 0.1205 | 0.0985 | 2.2835 | 2.1465 |
| 7 | 41.438 | 35.938 | 2.6475 | 2.5320 | 0.1660 | 0.2350 | 0.1175 | 0.1080 | 2.2965 | 2.1565 |
| 8 | 41.438 | 35.875 | 2.6140 | 2.5440 | 0.1560 | 0.2420 | 0.1150 | 0.0905 | 2.3060 | 2.2065 |
| Average | 41.477 | 35.852 | 2.6487 | 2.5161 | 0.1614 | 0.2669 | 0.1213 | 0.0987 | 2.3076 | 2.1681 |



Table 2.4 (a) - Geometry of PVC 36" Pipe

| Test No. | OD (in) | ID (in) | $t_{\text {a }}($ in) | $t_{b}$ (in) | $\mathrm{t}_{\mathrm{c}}$ (in) | $\mathrm{tin}_{\text {(in) }}$ | $\mathrm{t}_{\text {out }}$ (in) | $\mathrm{d}_{1}$ (in) | $\mathrm{d}_{2}(\mathrm{in})$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 38.813 | 35.5 | 1.6320 | 0.1770 | 0.2385 | 0.1900 | 0.1415 | 1.8010 | 1.6755 |
| 2 | 38.813 | 35.5 | 1.6255 | 0.1585 | 0.2380 | 0.2010 | 0.1390 | 1.7895 | 1.6685 |
| 3 | 38.813 | 35.438 | 1.6390 | 0.1660 | 0.2435 | 0.1975 | 0.1325 | 1.7880 | 1.6630 |
| 4 | 38.813 | 35.438 | 1.6230 | 0.1635 | 0.2370 | 0.1890 | 0.1370 | 1.7735 | 1.6765 |
| 5 | 38.75 | 35.5 | 1.6260 | 0.1695 | 0.2340 | 0.1960 | 0.1395 | 1.8160 | 1.6755 |
| 6 | 38.813 | 35.563 | 1.6240 | 0.1830 | 0.2350 | 0.1930 | 0.1360 | 1.7850 | 1.6760 |
| 7 | 38.688 | 35.5 | 1.6245 | 0.1615 | 0.2335 | 0.1905 | 0.1370 | 1.7850 | 1.6740 |
| 8 | 38.75 | 35.438 | 1.6225 | 0.1725 | 0.2370 | 0.1900 | 0.1385 | 1.7970 | 1.6975 |
| Average | 38.7734 | 35.5078 | 1.6271 | 0.1689 | 0.2371 | 0.1934 | 0.1376 | 0.1719 | 1.6758 |

Table 2.4 (b) - Inside and Outer Diameters Using Perimeter Measurements

| Test <br> No. | Perimeter <br> P(in) | OD (in) <br> $(\mathrm{OD}=\mathrm{P} / \pi)$ | ID (in) <br> $(\mathrm{ID}=$ <br> $\mathrm{OD}-2$ <br> $\left.\mathrm{t}_{\mathrm{a}}\right)$ |
| :---: | :---: | :---: | :---: |
| 1 | 122.25 | 38.9134 | 35.6494 |
| 2 | 122.50 | 38.9930 | 35.7420 |
| 3 | 122.50 | 38.9930 | 35.7150 |
| 4 | 122.50 | 38.9930 | 35.7470 |
| 5 | 122.25 | 38.9134 | 35.6614 |
| 6 | 122.25 | 38.9134 | 35.6654 |
| 7 | 122.125 | 38.8736 | 35.6644 |
| 8 | 122.25 | 38.9134 | 35.6684 |
| Average | 122.3281 | 38.9383 | 35.6841 |


Table 2.5 - Geometry of Aluminum 36" Pipe

| Test <br> No. | $\begin{aligned} & \text { OD } \\ & \text { (in) } \end{aligned}$ | ID (in) | t (in) | $\mathrm{d}_{1}$ (in) | $\mathrm{d}_{2}$ (in) | $\mathrm{d}_{3}$ (in) | $\mathrm{d}_{4}$ (in) | $\mathrm{d}_{\mathrm{s}}$ (in) | $\mathrm{d}_{\text {a }}$ (in) | $\mathrm{d}_{l}$ (in) | $\begin{aligned} & \hline \mathbf{d}_{\mathrm{tap}} \\ & \text { (in) } \end{aligned}$ | $\mathrm{t}_{\text {tap }}$ (in) | $\alpha$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | $37_{1 / 4}$ | 35716 | 0.0865 | 0.9395 | 0.7625 | 6716 | 221516 | $43 / 8$ | $51 / 8$ | $87 / 8$ | 0.6885 | 0.1875 |  |
| 2 | $371 / 2$ | $357 / 8$ | 0.0835 | 0.9470 | 0.7935 | $63 / 8$ | $23^{1 / 16}$ | $45 / 16$ | 5 5/32 | 829132 | 0.7155 | 0.2205 |  |
| 3 | 37 | $357 / 8$ | 0.0820 | 0.9545 | 0.7825 | $63 / 8$ | 23 | $47 / 16$ | 53/16 | $87 / 8$ | 0.6790 | 0.1830 |  |
| 4 | 37 | $3511 / 16$ | 0.0830 | 0.9440 | 0.7570 | $63 / 8$ | $231 / 16$ | $41 / 2$ | $51 / 8$ | $87 / 8$ | 0.6980 | -- |  |
| 5 | 373/8 | 353/8 | 0.0865 | 0.9490 | 0.7805 | $63 / 8$ | $23_{1 / 16}$ | $41 / 2$ | $51 / 8$ | $87 / 8$ | 0.6965 | --- |  |
| 6 | 373/8 | $357 / 8$ | 0.0850 | 0.9475 | 0.7500 | $63 / 8$ | $2315 / 16$ | 47/16 | 5/32 | $87 / 8$ | 0.6805 | $\cdots$ |  |
| 7 | $371 / 2$ | $357 / 8$ | 0.0815 | 0.9445 | 0.8105 | $63 / 8$ | 23 | $47 / 16$ | $55 / 32$ | $813 / 16$ | 0.6855 | -- |  |
| 8 | $371 / 4$ | 35 11/16 | 0.0895 | 0.9410 | 0.7790 | $63 / 8$ | $23^{1 / 16}$ | $41 / 2$ | 55/32 | 829/32 | 0.6905 | --- |  |
| Average | 37.281 | 35.711 | 0.0847 | 0.9459 | 0.7769 | 6.383 | 23.016 | 4.4375 | 5.1484 | 8.875 | 0.6918 | 0.1970 | $78.5^{\circ}$ |


Table 2.6-Geometry of steel 36" Pipe

| Test No. | OD (in) | ID (in) | t (in) | $\mathrm{d}_{1}$ (in) | $\mathrm{d}_{2}$ (in) | $\mathrm{d}_{3}$ (in) | $\mathrm{d}_{4}$ (in) | $\mathrm{d}_{5}$ (in) | $\mathrm{d}_{\mathrm{a}}$ (in) | $\mathrm{d}_{l}$ (in) | $\mathrm{d}_{\text {tap }}$ (in) | $\mathrm{t}_{\text {tap }}$ (in) | $\alpha$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 36 | 34 11/16 | 0.0865 | 0.9395 | 0.7625 | $67 / 16$ | $22_{15 / 16}$ | 43/8 | $51 / 8$ | $87 / 8$ | 0.6885 | 0.3510 |  |
| 2 | $363 / 8$ | $347 / 8$ | 0.0835 | 0.9470 | 0.7935 | $63 / 8$ | $23_{1 / 16}$ | $45 / 16$ | 5 5/32 | $829 / 32$ | 0.7155 | 0.3495 |  |
| 3 | 36716 | $347 / 8$ | 0.0820 | 0.9545 | 0.7825 | $63 / 8$ | 23 | 47/16 | $53 / 16$ | $87 / 8$ | 0.6790 | 0.3295 |  |
| 4 | 36716 | $3415 / 16$ | 0.0830 | 0.9440 | 0.7570 | 6 3/8 | 23 1/16 | $41 / 2$ | $51 / 8$ | $87 / 8$ | 0.6980 | --- |  |
| 5 | $361 / 8$ | 34314 | 0.0865 | 0.9490 | 0.7805 | $63 / 8$ | $23_{1 / 16}$ | $41 / 2$ | $51 / 8$ | $87 / 8$ | 0.6965 | --- |  |
| 6 | $363 / 8$ | $347 / 8$ | 0.0850 | 0.9475 | 0.7500 | 6 3/8 | $23_{15 / 16}$ | 47/16 | 5 5/32 | $87 / 8$ | 0.6805 | --- |  |
| 7 | 367116 | $3415 / 16$ | 0.0815 | 0.9445 | 0.8105 | $63 / 8$ | 23 | $47 / 16$ | $5^{5 / 32}$ | $8_{13 / 16}$ | 0.6855 | --- |  |
| 8 | $365 / 16$ | 35 | 0.0895 | 0.9410 | 0.7790 | $63 / 8$ | $23^{1 / 16}$ | $41 / 2$ | $5_{\text {5/32 }}$ | $8^{29 / 32}$ | 0.6905 | --- |  |
| Average | 36.313 | 35.836 | 0.0847 | 0.9459 | 0.7769 | 6.383 | 23.016 | 4.4375 | 5.1484 | 8.875 | 0.6918 | 0.3433 | $78.5^{\circ}$ |


2-8
Table 2.7
Visual Inspection and Material Characterization: AASHTO : M 294M-98 for HDPE Pipe ADS $48{ }^{\prime \prime}$

| Article | Details and observations |
| :---: | :---: |
| Article 6.1 (MP7 or M294) <br> Pipes \& fittings shall be made of virgin PE compounds, which conforms with the requirements of cell class 335420 C . | Manufacturer's description: Letter of certification from ADS of Dec. 21, 2000 indicates compliance of the resin cell classification of 335420 C and the resin meets the requirements of AASHTO M294-01. (Letter provided in appendix). |
| Article 4.5 (MP7 or Article 4.1.3 M294-98) The prevailing specification requires that the inner and outer surfaces be essentially smooth. | The surfaces of the interior and exterior walls are rough and irregular (see photographs, Figs. 2.5 and 2.6). |
| Article 7.1 (MP7 or M294-98) <br> The prevailing specification requires that the pipe be free of visible defects, defined as: cracks, creases, unpigmented or non uniformly pigmented pipes. | The pipes' surface inside and outside walls, reveal visible deep creasing (see photographs, Figs. 2.5 and 2.6). |
| Article 7.8.1 (MP7 or M294-98) <br> Requires that the fittings not impair the overall integrity or function of the pipeline. | The bell and spigot joint shows irregular surfaces at certain locations around the circumference (see photograph, Fig. 2.7). |
| Article 7.2.2 (MP7 for $\phi \geq 1350 \mathrm{~mm}$ and M294-98 for $\boldsymbol{\phi} \leq \mathbf{1 2 0 0} \mathbf{~ m m}$ ) <br> The prevailing specification requires a minimum wall thickness of 1.8 mm for pipe diameter greater than 900 mm and 1.7 mm for 900 mm diameter pipe. | The minimum measured wall thickness is $2.06 \mathrm{~mm}(0.08 \mathrm{in})$, which is greater than the minimum required wall thickness of 1.8 mm . |
| Article 7.2.3 Inside Diameter <br> Tolerances (MP7 and M294-98). The tolerance on the specified ID shall be $4.5 \%$ oversize and $1.5 \%$ undersize, but not more than 30 mm oversize. | The average measured inside diameter is 1194 mm ( 47.02 in .); which represents $2.04 \%$ undersize based on the nominal diameter of $1219.2 \mathrm{~mm}(48 \mathrm{in})$. This undersize is smaller than 30 mm specified in MP7 and M294-98. |

Table 2.8
Visual Inspection and Material Characterization: AASHTO : M 294M-98 for HDPE Pipe ADS 36

| Article | Details and observations |
| :---: | :---: |
| Article 6.1 (MP7 or M294) <br> Pipes \& fittings shall be made of virgin PE compounds, which conforms with the requirements of cell class 335420 C . | Manufacturer's description: <br> As per letter (not dated) from ADS (Mr. James Park), pipes and fittings are manufactured according to AASHTO : M294M-98) with specified cell class 335420C. |
| Article 4.5 (MP7 or Article 4.1.3 M294-98) <br> The prevailing specification requires that the inner and outer surfaces be essentially smooth. | The inner surface is smooth with waviness in the longitudinal direction. The rise in the waviness is in the valley and the depressions in the corrugations (see photographs, Figs. 2.8 and 2.9). |
| Article 7.1 (MP7 or M294-98) <br> The prevailing specification requires that the pipe be free of visible defects, defined as: cracks, creases, unpigmented or non uniformly pigmented pipes. | No visible defects or creases were observed in the interior surface. |
| Article 7.8.1 (MP7 or M294-98) <br> Requires that the fittings not impair the overall integrity or function of the pipeline. | The bell and spigot show no irregularities. |
| Article 7.2.2 (MP7 for $\phi \geq 1350 \mathrm{~mm}$ and M294-98 for $\phi \leq \mathbf{1 2 0 0} \mathbf{~ m m}$ ) <br> The prevailing specification requires a minimum wall thickness of 1.8 mm for pipe diameter greater than 900 mm and 1.7 mm for 900 mm diameter pipe. | The minimum measured wall thickness is $3.06 \mathrm{~mm}(0.125 \mathrm{in})$, which is greater than the minimum required wall thickness of 1.7 mm . |
| Article 7.2.3 Inside Diameter <br> Tolerances (MP7 and M294-98). The tolerance on the specified DD shall be $4.5 \%$ oversize and $1.5 \%$ undersize, but not more than 30 mm oversize. | The average measured inside diameter is 914 mm ( 36.0 in ), which equals the nominal diameter. |

## Table 2.9

Visual Inspection and Material Characterization: AASHTO : M 294M-98 for HDPE Pipe HANCOR 36

| Article | Details and observations |
| :---: | :---: |
| Article 6.1 (MP7 or M294) <br> Pipes \& fittings shall be made of virgin PE compounds, which conforms with the requirements of cell class 335420 C . | Manufacturer's description: <br> As per letter dated December 4, 2000, the pipes and fittings are manufactured according to AASHTO : M294M-98 with specified cell class 335500C |
| Article 4.5 (MP7 or Article 4.1.3 M294-98) <br> The prevailing specification requires that the inner and outer surfaces be essentially smooth. | The inner surface is smooth with waviness in the longitudinal direction. The rise in the waviness is in the valley and the depressions in the corrugations (see photographs, Figs. 2.10 and 2.11). |
| Article 7.1 (MP7 or M294-98) <br> The prevailing specification requires that the pipe be free of visible defects, defined as: cracks, creases, unpigmented or non uniformly pigmented pipes. | No visible defects or creases were observed in the interior surface. |
| Article 7.8.1 (MP7 or M294-98) <br> Requires that the fittings not impair the overall integrity or function of the pipeline. | The bell and spigot show no irregularities. |
| Article 7.2.2 (MP7 for $\phi \geq 1350 \mathrm{~mm}$ and M294-98 for $\phi \leq \mathbf{1 2 0 0} \mathbf{~ m m}$ ) <br> The prevailing specification requires a minimum wall thickness of 1.8 mm for pipe diameter greater than 900 mm and 1.7 mm for 900 mm diameter pipe. | The minimum measured wall thickness is $2.91 \mathrm{~mm}(0.1145 \mathrm{in})$, which is greater than the minimum required wall thickness of 1.7 mm . |
| Article 7.2.3 Inside diameter <br> Tolerances (MP7 and M294-98). The tolerance on the specified ID shall be $4.5 \%$ oversize and $1.5 \%$ undersize, but not more than 30 mm oversize. | The average measured inside diameter is 910.6 mm ( 35.852 in ), which is smaller than the nominal diameter. This represents $0.4 \%$ undersize, which is smaller than $1.5 \%$ undersize specified by the code. |

Table 2.10
Visual Inspection and Material Characterization: ASTM F949-00 for PVC Pipe

| Article | Details and observations |
| :--- | :--- |
| Article 4.1 <br> The pipe shall be made of PVC compound having a <br> minimum cell classification of 12454B or 12454C in <br> accordance with specification ASTM D 1784. The <br> fittings shall be made of PVC compound having a cell <br> classification of 12454B, 12454C or 13343C. | Manufacturer's description: <br> As per letter dated January 17, 2001 from Contech, PVC A-2000 Pipe is manufactured as <br> detailed in Article 4.1 of ASTM F 949-00. |
| Article 4.3 <br> Pipe shall be manufactured by simultaneous extrusion <br> of the smooth inner wall fused to the outer corrugated <br> wall. | Manufacturer's description: <br> As per letter dated January 17, 2001 from Contech, PVC A-2000 Pipe is manufactured as <br> detailed in Article 4.3 of ASTM F 949-00. |
| Article 5.1 <br> The pipe and fittings shall be homogeneous <br> throughout and free from visible cracks, holes, | The pipe and fittings are homogeneous. No visible cracks, holes or foreign inclusions are <br> foreign inclusions or other injurious defects. |

Table 2.11
Visual Inspection and Material Characterization: AASHTO : T249-93 for Aluminum Pipe

| Article | Details and observations |
| :---: | :---: |
| 2. Lock seam quality (see Fig. below) <br> - Seam lap <br> - Contact of lapped surfaces <br> - Interior angularity and roller indentation | - Seam lap is smaller than the code specification average value of seam lap, which is 7.9 mm ( 0.311 in .). The seam lap is not equidistant from the two adjacent ribs, as required in the code. (Distance of seam lap from left rib $=$ 99 mm ( 3.89 in .), distance from right rib $=132 \mathrm{~mm}(5.18 \mathrm{in}$ ). <br> - The lapped surfaces are not in tight contact as required by the code. The physical separation between the lapped surfaces is quite evident (see photograph, Fig. 2.14). <br> - The roller indentation seems to be satisfactory, whereas the interior angularity is excessive. |

Table 2.12

| Article | Details and observations |
| :---: | :---: |
| 2. Lock seam quality (see Fig. below) <br> - Seam lap <br> - Contact of lapped surfaces <br> - Interior angularity and roller indentation | - Seam lap, which is $8.72 \mathrm{~mm}(0.343 \mathrm{in})$, is greater than the code specification minimum value of seam lap, which is 7.9 mm ( 0.311 in .). The seam lap is not equidistant from the two adjacent ribs as required in the code. (Distance of seam lap from left rib $=99 \mathrm{~mm}$ ( 3.89 in .), distance from right rib $=132 \mathrm{~mm}$ (5.18 in)). <br> - There is no apparent physical separation between the lapped surfaces (see photograph, Fig. 2.15). <br> - The roller indentation seems to be satisfactory. Although the interior angularity is better than that observed in the aluminum pipe, it does not comply fully with the code specifications. |

(a) Bell

(b) Spigot


Fig. 2.1 - Bell and Spigot Geometry of ADS 48 (Average Values)
(a) Bell

(b) Spigot


Fig. 2.2 - Bell and Spigot Geometry of ADS 36 (Average Values)
(a) Bell

(b) Spigot


Fig. 2.3 - Bell and Spigot Geometry of Hancor 36 (Average Values)
(a) Bell

$A=35.50$ in
$B=38.75$ in
$C=10.75$ in $D=2.25$ in $E=42.75$ in $F=39.50$ in
(b) Spigot

Spigot have the same dimensions as A and B above

Fig. 2.4 - Bell and Spigot Geometry of PVC 36 (Average Values)
(a) Inside View

(b) Outside View


Fig. 2.5 - Inside and Outside Wall Surfaces of ADS 48
(a) Wall Section

(b) Close-up View of Wall Section


Fig. 2.6 - Wall Section of ADS 48


Fig. 2.7 - Bell and Spigot Joint of ADS 48


Fig. 2.8 - Wall Section of ADS 36
(a) Inside View

(b) Outside View


Fig. 2.9- Inside and Outside Wall Surfaces of ADS 36
(a) Inside View

(b) Outside View


Fig. 2.10 - Inside and Outside Wall Surfaces of Hancor 36


Fig. 2.11 - Wall Section of Hancor 36


Fig. 2.12 - Wall Section of PVC 36
(a) Inside View

(b) Outside View


Fig. 2.13 - Inside and Outside Wall Surfaces of PVC 36

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Fig. 2.14 - Lock-Seam Section of Aluminum 36


Fig. 2.15-Lock-Seam Section of Steel 36

## Chapter 3: Simple Beam Tests

### 3.0 Objectives

The objective of this test is to evaluate the pipe performance when subjected to longitudinal bending. The strains experienced on outside walls versus inside walls as well as the longitudinal strains and stresses in relation to vertical deflections are of particular interest.

### 3.1 Experimental Program

## Specimens

Six test pipe specimens having approximately 20 feet in length (full length of pipe) were selected, one for each of the pipe types considered: HDPE ADS 48, HDPE ADS 36, HDPE HANCOR 36, PVC 36, Aluminum 36, and Steel 36.

## Test Setup

The test specimens were simply supported and subjected to four point bending. Fig. 3.1 presents photographs of typical setups for beam tests at the Structures Research Center, FDOT, Tallahassee. The widths of the end supports were made sufficiently large to prevent local failure and to permit end rotation (see Figs. 3.2 to 3.4).

## Instrumentation and Test Procedure

The test program included application of loads in predetermined increments until failure of the specimens. Each test specimen was instrumented with electrical resistance strain gages in the longitudinal and transverse directions, deflection gages, and crack gages (Figs. 3.2 to 3.4). The pipe response was. monitored and recorded after each load increment with a computer-controlled data acquisition system. Longitudinal strain gages were installed at the outer and inner surfaces at the top and bottom of the pipe at three transverse sections in one-half of the pipe specimen's sp an (see Fig. 3.5b). The transverse strain gages were installed (see Fig. 3.5b) at the third section around the circumference (Figs. 3.2a, 3.3a and 3.4a) to measure hoop strains. Vertical deflections were measured at the top and bottom of the specimens at $3-\mathrm{ft}(914.4 \mathrm{~mm})$ sections along the specimen's span (Figs. 3.2 to 3.4 ).

### 3.3 Presentation of Results

The experimental results are presented as follows:

- For ADS 48 (Figs. 3.6 to 3.11)
- Load versus deflections along the pipe in Fig. 3.6,
- Deflection versus distance along pipe in Fig. 3.7,
- Top and bottom outer strains versus distance along pipe in Fig. 3.8,
- Top and bottom inner strains versus distance along pipe in Fig. 3.9,
- Bottom deflection and load versus longitudinal strains at centerline in Fig. 3.10,
- Slope of load versus bottom deflection at centerline in Fig. 3.11.
- For ADS 36 (Figs. 3.12 to 3.18)
- Load versus deflection along the `top and bottom of the specimen in Fig. 3.12,
- Top and bottom deflection at sections along the specimen in Fig. 3.13,
- Top and bottom outer surface strains at sections along the specimen in Fig. 3.14,
- Top and bottom inner surface strains at sections along the specimen in Fig. 3.15,
- Bottom deflection and load versus longitudinal strains at centerline in Fig. 3.16,
- Slope of load versus bottom deflection at centerline in Fig. 3.17,
- Load versus valley longitudinal and transverse strains in Fig. 3.18.
- For HANCOR 36 (Fig. 3.19 to 3.25)
- Load versus deflections along, the top and bottom of the specimen in Fig. 3.19,
- Top and bottom deflections at 3-ft sections along the specimen in Fig. 3.20,
- Top and bottom outer surface strains at sections along the specimen in Fig. 3.21,
- Top and bottom inner surface strains at sections along the specimen in Fig. 3.22,
- Bottom deflection and load versus longitudinal strains at centerline in Fig. 3.23,
- Slope of load versus bottom deflection at centerline in Fig. 3.24,
- Load versus valley longitudinal and transverse strains in Fig. 3.25.
- For PVC 36 (Figs. 3.26 to 3.32)
- Load versus deflections along the top and bottom of the specimen in Fig. 3.26,
- Top and bottom deflections at 3-ft sections along the specimen in Fig. 3.27,
- Top and bottom outer surface strains at sections along the specimen in Fig. 3.28,
- TOP and bottom inner surface strains at sections along the specimen in Fig. 3.29,
- Bottom deflection and load versus longitudinal strains at centerline in Fig. 3.30,
- Slope of load versus bottom deflection at centerline in Fig. 3.31,
- Load versus valley longitudinal and transverse strains in Fig. 3.32.
- For Steel 36 (Figs. 3.33 to 3.39)
- Load versus deflections along the top and bottom of the specimen in Fig. 3.33,
- Top and bottom deflections at 3-ft sections along the specimen in Fig. 3.34,
- Top and bottom outer surface strains at sections along the specimen in Fig. 3.35,
- Bottom deflection and load versus longitudinal strains at centerline in Fig. 3.36,
- Slope of load versus bottom deflection at' centerline in Fig. 3.37,
- Load versus valley longitudinal strains in Fig. 3.38,
- View of lock seam lap near support and at centerline in Fig. 3.39.
- $\quad$ For Aluminum 36 (Figs. 3.40 to 3.46 )
- Load versus deflections along the top and bottom of the specimen in Fig. 3.40,
- Top and bottom deflections at 3- ft sections along the specimen in Fig. 3.41,
- Top and bottom outer surface strains at sections along the specimen in Fig. 3.42,
- Bottom deflections and load versus longitudinal strains at centerline in Fig. 3.43,
- Slope of load versus bottom deflection at centerline in Fig. 3.44,
- Load versus valley longitudinal strains in Fig. 3.45,
- View of lock seam lap near support and at centerline in Fig. 3.46.


## 34 Observations and Discussions

(a) ADS 36 and Hancor 36 achieved a similar bending stiffness (approximately 1100 Lbs/in j and properties (Table 3 1). The bending stiffness of PVC 36 is almost 3 times greater than that of ADS 36 or Hancor 36. This is mainly due to the higher modulus of elasticity of PVC compared to HDPE pipes. The aluminum pipe achieved the lowest bending stiffness ( $638 \mathrm{Lbs} / \mathrm{in}$.), while ADS 48 achieved the highest bending stiffness (5213 Lbs/in.).
(b) Given the load. and the section along the pipe, the bottom deflections (invert) are generally smaller than the top deflections. This is due to the ring and wall deflections which add up to the top deflection.
(c) For ADS 48, the inner wall longitudinal strains were compressive and outer wall strains tensile at both invert and crown.
(d) For ADS 36, both the top and bottom inner walls were in tension, whereas the top outer wall was in compression. The top inner wall experienced practically no strain (Fig. 3:16).
(e) For Hancor 36, the bottom inner and outer walls were in tension. (small-strains were recorded for the outer wall), while the top outer and inner walls were in compression (small strains were recorded for the inner wall); see. Fig. 3.23.
(f) For PVC 36, the top: and bottom outer wall strains were negligible. The top inner wall was in compression, whereas the bottom inner wall was in tension (see Fig. 3.30).
(g) For the steel and aluminum pipes, the top wall was in compression, while the bottom wall was in tension (Figs. 3.36 and 3.45).
(h) For the plastic pipes, the valley longitudinal bending strains were greater, than the crown longitudinal bending strains (Fig. 3.18, Fig. 3.32).. For the metal, pipes, the longitudinal bending, strains in the ribs were greater than the longitudinal bending strains in the wall (valley) between the ribs (Fig. 3.38: and-Fig. 3.45).
(i) The lock seams behaved satisfactorily. No separation or, loss of contact between seam laps was observed (see Figs. 3.39 and 3.46).
(j) For the aluminum pipe, the wall (valley) between the ribs experienced practically no longitudinal bending strains. The latter were concentrated in the ribs (Fig. 345).
(k) For a vertical bottom deflection of $1 \%$ and $2 \%$ of the span, the obtained maximum longitudinal tensile strains are as given in Table 3.2. It can be observed that for $1 \%$ deflection, which can be seen as the maximum grade slope during installation, the longitudinal bending strain ranged from $114 \mu \varepsilon$ (i.e., 12.5 psi ) (ADS 36) to $1000 \mu \varepsilon$ (i.e., 110 psi ) (Rancor 36) for HDRE, it reached $600 \mu \varepsilon$ (i.e., 240 , psi) for PVC and $200 \mu \varepsilon$ (i.e., 5800 psi for steel and 2000 psi for aluminum) for metal pipes.

Table 3.1 - Experimental Stiffness of Pipes in Bending

| Series | Stiffness, $\mathrm{K}^{(\mathrm{a})}$ | $E I^{(b)}$ | $\mathrm{I}^{(c)}$ |
| :---: | :---: | :---: | :---: |
|  | (Lbs/in.) | (kip-in. ${ }^{\text {2 }}$ ) | (in. ${ }^{4}$ ) |
| ADS 48 | 5213 | 931497 | 8468 |
| ADS 36 | 1061 | 189536 | 1723 |
| HANCOR 36 | 1221 | 218103 | 1983 |
| PVC36 | 3291 | 588010 | 1470 |
| Steel 36 | 1311 | 234256 | 8.08 |
| Aluminum 36 | 638 | 113999 | 11.40 |

Notes: ${ }^{(\mathrm{a})} \quad \mathrm{K}=$ Force/displacement; see curve fittings on Figs. 3.16, 3.22, 3.29, 3.36, 3.42 and 3.49 for ADS 48 , ADS 36, HANCOR 36, PVC 36, Steel 36 and Aluminum 36, respectively. $\mathrm{K}=\mathrm{F} \Delta$
(b) For 4 point loading, from beam theory $\mathrm{K}=56.4 \mathrm{EI} / \mathrm{L}^{3}$, then $\mathrm{EI}=\mathrm{KL}^{3} / 56.4$, with $\mathrm{L}=18^{\prime}=216$ in.
(c) Assuming $\mathrm{E}=29000 \mathrm{ksi}$ for Steel, $\mathrm{E}=10000 \mathrm{ksi}$ for aluminum, $\mathrm{E}=110 \mathrm{ksi}$ for HDPE and $\mathrm{E}=400$ ksi for PVC.

Table 3.2 - Longitudinal Bending Strains for Deflections of $1 \%$ and $2 \%$ Beam Span

| Series | Strains at 1\% | Strains at 2\% |
| :--- | :---: | :---: |
|  | $(\mu \varepsilon)$ | $(\mu \varepsilon)$ |
| ADS 48 $8^{(a)}$ | 672 | 1290 |
| ADS 36 ${ }^{(0)}$ | 114 | 221 |
| HANCOR 36 ${ }^{(\mathrm{c})}$ | 1000 | 2200 |
| PVC 36 ${ }^{(\mathrm{d})}$ | 600 | 1100 |
| Steel 36 |  |  |
| Aluminum 36 | $186(-185)$ | $350(-350)$ |

Notes: (a) Gage L34; (b) Gage VL34; (c) Gage L21; (d) Gage L33;
(e) Gages RL34 (-) and RL31 (+); (f) Gages L24 (+) and RL34(-)


Fig. 3.1 - Views of Pipes under Beam Test Setup
Deflection Gauges

Fig 3.2a - Location of Deflection and Transverse Strain Gages at Section 3 Around Circumference

Fig 3.2b - Location of Longitudinal Strain Gages on Top and Bottom of ADS 48 Pipe Specimen
Deflection Gauges

Fig 3.3a - Location of Deflection and Transverse Strain Gages at Section 3 Around Circumference of ADS 36, Hancor 36 and PVC 36 Pipe Specimens

Fig 3.3b - Location of Longitudinal Strain Gages on Top and Bottom of ADS 36, Hancor 36 and PVC 36 Pipe Specimens
Deflection Gauges

Fig 3.4a - Location of Deflection and Transverse Strain Gages at Section 3 Around Circumference of Steel 36 and Aluminum 36 Pipe Specimens

Fig 3.4b - Location of Longitudinal Strain Gages at Section 3 Around Circumference of Steel 36 and Aluminum 36 Pipe Specimens

(a) Transverse Strain Gages Inside Specimen

(b) Longitudinal Strain Gages Inside Specimen

Fig. 3.5 - Typical Transverse and Longitudinal Strain Gages Inside Specimen


Fig. 3.6-Load vs Deflection Measured Along Top and Bottom of ADS 48 Specimen


Fig. 3.7 - Measured Top and Bottom Deflections at Sections Along ADS 48 Specimen


Fig. 3.8 - Measured Top and Bottom Outer Surface Strains at Sections Along ADS 48


Fig. 3.9- Measured Top and Bottom Inner Surface Strains at Sections Along ADS 48


Fig. 3.10 Bottom Deflection and Load versus Longitudinal Strains at Centerline for ADS 48 Specimen


Fig. 3.11 - Slope of Load versus Bottom Deflection for ADS 48 Specimen at Centerline


Fig. 3.12 - Load vs Deflection Measured Along Top and Bottom of ADS 36 Specimen


Fig. 3.13 - Measured Top and Bottom Deflections at Sections Along ADS 36 Specimen


Fig. 3.14 - Measured Top and Bottom Outer Surface Strains at Sections Along ADS 36


Fig. 3.15 - Measured Top and Bottom Inner Surface Strains at Sections Along ADS 36


Fig. 3.16-Bottom Deflection and Load versus Longitudinal Strains at Centerline for ADS 36 Specimen


Fig. 3.17 - Slope of Load versus Bottom Deflection for ADS 36 Specimen at Centerline


Fig. 3.18-Load versus Valley Longitudinal and Transverse Strains for ADS 36


Fig. 3.19 - Load vs Deflection Measured Along Top and Bottom of Hancor 36


Fig. 3.20 - Measured Top and Bottom Deflections at Sections Along Hancor 36


Fig. 3.21 - Measured Top and Bottom Outer Surface Strains at Sections Along
Hancor 36

(a) Top Inner

(b) Bottom Inner

Fig. 3.22 - Measured Top and Bottom Inner Surface Strains at Sections Along Hancor 36 Specimen


Fig. 3.23 Bottom Deflection and Load versus Longitudinal Strains at Centerline for Hancor 36 Specimen


Fig. 3.24 - Slope of Load versus Bottom Deflection at Centerline for Hancor 36 Specimen


Fig. 3.25 - Load versus Valley Longitudinal and Transverse Strains for Hancor 36


Fig. 3.26 - Load vs Deflection Measured Along Top and Bottom of PVC 36


Fig. 3.27 - Measured Top and Bottom Deflections at Sections Along PVC 36


Fig. 3.28 - Measured Top and Bottom Outer Surface Strains at Sections Along PVC 36 Specimen

(a) Top Inner

(b) Bottom Inner

Fig. 3.29-Measured Top and Bottom Inner Surface Strains at Sections Along PVC 36 Specimen


Fig. 3.30 Bottom Deflection and Load versus Longitudinal Strains at Centerline for PVC 36 Specimen


Fig. 3.31 - Slope of Load versus Bottom Deflection for PVC 36 Specimen at Centerline


Fig. 3.32 - Load versus Valley Longitudinal and Transverse Strains for PVC 36


Fig. 3.33 - Load vs Deflection Measured Along Top and Bottom of Steel 36


Fig. 3.34 - Measured Top and Bottom Deflections at Sections Along Steel 36


Fig. 3.35 - Measured Top and Bottom Outer Surface Strains at Sections Along Steel 36 Specimen


Fig. 3.36 - Bottom Deflection and Load versus Longitudinal Strains at Centerline for Steel 36 Specimen


Fig. 3.37 - Slope of Load versus Bottom Deflection at Centerline for Steel 36


Fig. 3.38 - Load versus Valley Longitudinal Strains for Steel 36

(a) Near Support

(b) At Centerline

Fig. 3.39 - Lock Seam Sections of Steel Specimen


Fig. 3.40 - Load vs Deflection Measured Along Top and Bottom of Aluminum 36


Fig. 3.41 - Measured Top and Bottom Deflections at Sections Along Aluminum 36


Fig. 3.42 - Measured Top and Bottom Outer Surface Strains at Sections Along Aluminum 36 Specimen


Fig. 3.43 Bottom Deflection and Load versus Longitudinal Strains at Centerline for Aluminum 36 Specimen


Fig. 3.44 - Slope of Load versus Bottom Deflection at Centerline for Aluminum 36


Fig. 3.45 - Load versus Valley Longitudinal Strains for Aluminum 36


Fig. 3.46 - Lock Seam Sections of Aluminum Specimen

## Chapter 4: Parallel Plate Loading Tests

### 4.1 Objectives

The objective of this test is to determine the load-deflection characteristics of flexible pipes under parallel-plate loading. The pipe stiffness (PS), the stiffness factor (SF), and the percentage pipe deflection $(\mathrm{P})$ are determined from this test. The interrelations of dimensions and deflection properties for flexible pipes are: also evaluated in the study.

### 4.2 Experimental Program Apparatus

The hydraulic jack used in the testing has the capability of constant-rate-crosshead movement. The rate of head approach can be varied and was in the range of 0.05 to 150 in . per minute. The load could be applied to the flexible pipe through two parallel flat, smooth, and clean steel bearing plates. The steel plate at the top is welded to a NAT steel beam and the load applied to the center of the WF beam. The thickness of the plates is about 0.875 in , so as to minimize bending or deformation of the plate during testing. The plate length is slightly larger than the specimen length, and the plate width is approximately equal to the pipe contact width at maximum pipe deflection plus 6.0 in . The change in inside diameter was measured using LVDTs in three directions: parallel and perpendicular to the direction of loading, and $45^{\circ}$ to the direction of loading. The LVDTs were used to measure to the nearest 0.01 in . Fig. 4.1 shows a typical experimental set-up for the test.

## Test Specimens

The test specimens included two sizes: 36 in. and 48 in. diameter. The 36 in. diameter pipes were. of HDPE, PVC, aluminum and steel. One type of HDPE pipe had a 48-in. diameter. The 36-in. diameter pipe test specimens except PVC had a length equal to the pipe diameter, while the 48 - in. diameter pipe specimen had a length of 40 inches. The PVC pipe specimens were of 13 inches length. The ends of the specimens were cut square and free of burrs and jagged edges. At least three specimens were tested for each pipe sample.

The average measured outside diameter (OD), inside diameter (ID) and lengths of the test specimens are presented in Table 4.1, along with the minimum pipe stiffness values specified
by AASHTO M294 for HDPE and ASTM F679 for PVC pipes. Details of the, measured values of pipe diameters and geometries are given in Chapter 2.

## Test Procedure

The pipe specimen is positioned with its longitudinal axis parallel to the: bearing plates; and: centered laterally in the test set-up. The LVDTs were installed in place (Fig. 4.1). The load was applied by means of a hydraulic jack on the center of a VVF beam.
The specimens were loaded at rates of 0.05 in. per minute, 0.5 in. per minute, 10 in . per minute and 150 in. per minute. The load-deflection measurements were recorded continuously and observations were made to identify liner cracking, crazing, wall cracking, wall delamination, rupture and wall buckling.

The test continued until the load on the specimen failed to increase with increasing deflection or the specimen exhibited a deformation of $30 \%$ of the average inside diameter. The tests were performed according to the ASTM D2412 Standard.

## Test Program

Details of the parallel plate test program for pipe stiffness carried out in this study are presented in Table 4.2.

### 4.3 Description of Significant Pipe Events

Liner cracking or crazing --- the occurrence of a break or network of fine breaks in the liner visible to the unaided eye.
Wall cracking--- the occurrence of a break in the pipe wail visible to the unaided eye.
Wall delamination--- the occurrence' of any separation in the components of the pipe wall visible to the unaided eye.
Rupture--- a crack or break extending entirely or partly though the pipe wall.
Wall buckling---any reverse curvature or deformation in the pipe wall that reduces the load carrying capability of the pipe.

### 4.4 Calculations

The pipe stiffness, PS, for any given deflection is given by:

$$
\begin{equation*}
\mathrm{PS}=\frac{F}{\Delta y} \tag{4.1}
\end{equation*}
$$

The stiffness factor, SF , for any given deflection as follows:

$$
\begin{equation*}
\mathrm{SF}=0.149 \mathrm{r}^{3} \mathrm{PS} \tag{4.2}
\end{equation*}
$$

Where:
$\Delta y=$ measured change in.the inside diameter in the direction of load application (in.),
F = the load applied to the pipe to produce a given percentage deflection, and
r = the mid-wall radius determined by subtracting the average wall thickness from the average outside diameter and dividing the difference by two (in.).

### 4.5 Results and Discussion

## Overall Results

Table 4.3 summarizes the experimental results for a vertical deflection of $5 \%$ o and $10 \%$ of the diameter. Table 4.3 a gives the vertical and horizontal deflections, whereas Table 4.3 b provides the average PS values obtained from the tests. In the case of HDPE, PVC, and .metal flexible pipes, there was no evidence of wall buckling, rupture, cracking or delamination until the specimens exhibited a vertical deflection of $15 \%$ of the diameter.

## Pipe-stiffness

The LVDTs recorded the change in the inside diameter of the test specimens, whereas the MTS measured the deformation of the pipe wall plus the change in the inside diameter. Only the PS values based on LVDT measurements are presented in the report. The PS values based on the MTS measurement are slightly smaller than those based on LVDT measured deformations. The PS values for all pipes are calculated for both $5 \%$ and $10 \%$ of the inside vertical diameter for different loading rates and are presented in Tables 4.4 to 4.9 for ADS 48, ADS 36, Hancor 36, PVC 36, Steel 36, and Aluminum 36, respectively. As expected, the higher the loading rate, the greater the PS value. It is observed that the PS values corresponding to $5 \%$ of the inside vertical diameter for the loading rate of 0.5 in. per minute are greater than the minimum value suggested by AASHTO and ASTM. Standards for both the HDPE and PVC pipes, see Table 4.1. The PS values for all the HDPE, PVC and metal
pipes corresponding to the vertical deflection of $10 \%$ o of the inside diameter are smaller than those based on the vertical deformation of $5 \%$ of the inside diameter except one specimen for each PVC and aluminum pipe test series.

## Load versus Deflections

The vertical deformation of the test specimens increased with increasing load. The HDPE and PVC pipes maintained a perfectly symmetric deformed shape, even at a relatively large vertical deformation of $20 \%$ of the inside diameter. However, the metal flexible pipes did not show any symmetry in the deformed shape, and thus, exhibited distinctly different behavior than that of HDPE and PVC pipes. The deformed shapes of the specimens for various levels of vertical deflections are presented in Figs. 4.3 to 4.8 for ADS 48, ADS 36, Hancor-36, PVC 36, Steel 36, and Aluminum 36, respectively. The curves representing the load versus the vertical and horizontal deflections are presented in Fig. 4.9 to 4.11 for ADS 4.8, APS 36, and Hancor 36, respectively, and in Fig. 4.12 for PVC 36, Steel 36, and Aluminum 36.

## Vertical Deflection versus Horizontal Deflection Ratio

The vertical deflection-horizontal deflection ratios, $\Delta_{v} / \Delta_{\mathrm{x}}$ are summarized in Table 4.3a for 5\% and $10 \%$ vertical deflections. As can be observed, the ratio $\Delta_{\mathrm{V}} / \Delta_{\mathrm{x}}$ did not vary as the load rate increased. As the vertical deflection increased from $5 \%$ to $10 \%$, the $\Delta_{v} / \Delta_{x}$ ratio did not change for PVC pipes, it slightly increased for HDPE pipes, and it slightly decreased for metal pipes. For the load rate of $0.5 \mathrm{in} . / \mathrm{min}$. and $5 \%$ deflection, the average $\Delta_{\mathrm{v}} / \Delta_{\mathrm{x}}$ ranged from 1.25 to 1.46 for HDPE (highest average value achieved by Hancor 36), whereas it ranged from 1.49 to 1.64 for metal pipes and was equal to 1.49 for PVC.

### 4.6 Conclusions

The following conclusions are of interest:
(a) HDPE and PVC pipes tested according to ASTM and AASHTO Standards (vertical deflection $(5 \%)$ and loading rate ( $0.5 \mathrm{in} . / \mathrm{min}$.) achieved a pipe stiffness, PS, higher than the minimum specified by the standards (Figs. 4.13 and 4.14). The PS values for
all the HDPE, PVC and metal pipes corresponding to the vertical deflection of $10 \%$ inside diameter are smaller than those based on vertical deformation of 5\% inside diameter (Figs. 4.15 and 4.16).
(b) Tests confirmed that for a given vertical deflection, the HDPE pipe stiffness (PS) substantially decreases with decrease in the loading rate and vice-versa.
(c) Up to a $15 \%$ vertical deflection, no sign of distress in the pipes was observed.

Table 4.1-Geometry (Average values) and Minimum Specified PS of Specimens

| Pipe Type | OD <br> (in) | ID <br> (in) | L <br> (in.) | Minimum PS <br> $(\mathbf{p s i})$ |
| :--- | :---: | :---: | :---: | :---: |
| ADS 48 | 52.320 | 47.016 | 40 | 18.14 |
| ADS 36 | 36.00 | 41.5703 | 36 | 21.77 |
| HANCOR 36 | 41.477 | 35.852 | 36 | 21.77 |
| PVC 36 | 38.7734 | 35.5078 | 13 | 46.00 |
| STEEL 36 | 36.313 | 35.836 | 36 | -- |
| ALUMINUM 36 | 37.28 | 35.852 | 36 | -- |

Table 4.2 - Parallel Plate Test Program

| Type of pipe | Load Rate <br> (in/min.) | Number of Tests |
| :--- | :---: | :---: |
| ADS 48 | 0.05 | 1 |
|  | 0.5 | 2 |
|  | 10 | 1 |
| ADS 36 | 150 | 1 |
|  | 0.05 | 1 |
|  | 0.5 | 2 |
|  | 10 | 1 |
| Hancor 36 | 150 | 1 |
|  | 0.05 | 1 |
|  | 0.5 | 2 |
| PVC 36 | 10 | 1 |
|  | 150 | 1 |
| Steel 36 | 0.05 | 1 |
| Alumimun 36 | 0.5 | 2 |

Table 4.3a - Summary of Experimental Results

| Series | Load Rate/Specimen\# (in./min.) | 5\% Vertical Deflection |  |  |  |  | 10\% Vertical Deflection |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| : |  | Load (Kips) | Load <br> (Lbs/in.) | $\Delta_{\mathrm{v}}$ <br> (in.) | $\begin{aligned} & \Delta_{\mathbf{x}} \\ & \text { (in.) } \end{aligned}$ | $\Delta_{\mathrm{v}} / \Delta_{\mathrm{x}}$ | Load (Kips) | Load <br> (Lbs/in.) | $\begin{aligned} & \Delta_{\mathrm{v}} \\ & \text { (in.) } \end{aligned}$ | $\begin{aligned} & \Delta_{\mathrm{x}} \\ & \text { (in.) } \end{aligned}$ | $\Delta_{\mathrm{V}} / \Delta_{\mathrm{x}}$ |
| ADS 48 | 0.05 | 1.99 | 49.75 | 2.40 | 1.97 | 1.22 | 2.995 | 74.88 | 4.80 | 3.62 | 1.33 |
|  | 0.5-1 | 2.65 | 66.25 | 2.40 | 1.91 | 1.26 | 4.03 | 100.75 | 4.80 | 3.61 | 1.33 |
|  | 0.5-2 | 2,52 | 63.00 | 2.40 | 1.93 | 1.24 | 3.82 | 95.50 | 4.80 | 3.63 | 1.32 |
|  | 0.5 Average | 2.58 | 64.63 | 2.40 | 1.92 | 1.25 | 3.93 | 98.13 | 4.80 | 3.62 | 1.33 |
|  | 10 | 3.46 | 86.50 | 2.445 | 2.22 | 1.10 | 5.24 | 131.00 | 4.81 | 3.83 | 1.26 |
|  | 150 | 4.45 | 111.25 | 2.41 | 1.96 | 1.23 | 7.01 | 175.25 | 4.82 | 3.60 | 1.34 |
| ADS 36 | 0.05 | 1.74 | 48.33 | 1.80 | 1.45 | 1.24 | 2.71 | 75.28 | 3.60 | 2.74 | 1.31 |
|  | 0.5-1 | 2.35 | 65.28 | 1.80 | 1.45 | 1.24 | 3.68 | 102.22 | 3.60 | 2.69 | 1.34 |
|  | 0.5-2 | 2.46 | 68.33 | 1.80 | 1.38 | 1.30 | 3.74 | 103.89 | 3.61 | 2.58 | 1.40 |
|  | 0.5 Average | 2.40 | 66.80 | 1.80 | 1.42 | 1.27 | 3.71 | 103.05 | 3.61 | 2.64 | 1.37 |
|  | 10 | 3.17 | 88.06 | 1.83 | 1.48 | 1.24 | 4.86 | 135.00 | 3.60 | 2.62 | 1.37 |
|  | 150 | 3.75 | 104.17 | 1.81 | 1.37 | 1.32 | 5.96 | 165.55 | 3.62 | 2.58 | 1.40 |
| HANCOR 36 | 0.05 | 1.36 | 37.77 | 1.80 | 1.23 | 1.46 | 1.92 | 53.33 | 3.60 | 2.38 | 1.51 |
|  | 0.5-1 | 1.58 | 43.89 | 1.81 | 1.19 | 1.52 | 2.49 | 69.17 | 3.60 | 2.39 | 1.51 |
|  | 0.5-2 | 1.73 | 48.06 | 1.80 | 1.27 | 1.42 | 2.68 | 74.44 | 3.61 | 2.53 | 1.43 |
|  | 0.5 Average | 1.66 | 45.98 | 1.80 | 1.23 | 1.46 | 1.58 | 71.80 | 3.60 | 2.46 | 1.47 |
|  | 10 | 2.37 | 65.83 | 1.81 . | 1.17 | 1.55 | 3.62 | 100.56 | 3.60 | 2.41 | 1.49 |
| PVC 36 | 0.5-1 | 1.28 | 90.46 | 1.81 | 1.27 | 1.43 | 2.34 | 180.00 | 3.60 | 2.51 | 1.43 |
|  | 0.5-2 | 1.28 | 98.46 | 1.80 | 1.15 | 1.57 | 2.22 | 170.77 | 3.61 | 2.40 | 1.50 |
|  | 0.5 Average | 1.28 | 94.46 | 1.80 | 1.21 | 1.49 | 2.28 | 175.38 | 3.60 | 2.45 | 1.46 |
|  | 10 | 1.32 | 101.54 | 1.82 | 1.22 | 1.50 | 2.22 | 170.77 | 3.60 | 2.37 | 1.52 |
| Steel 36 | 0.5-1 | 2.19 | 60.83 | 1.80 | 1.42 | 1.27 | 2.75 | 76.39 | 3.60 | 2.70 | 1.33 |
|  | 0.5-2 | 2.07 | 57.50 | 1.80 | 0.98 | 1.84 | 2.98 | 82.78 | 3.61 | 2.40 | 1.50 |
|  | 0.5-3 | 2.19 | 60.83 | 1.80 | 1.00 | 1.80 | 3.05 | 84.72 | 3.60 | 2.38 | 1.51 |
|  | Average | 2.15 | 59.72 | 1.80 | 1.13 | 1.64 | 2.93 | 81.30 | 3.60 | 2.49 | 1.45 |
| Aluminium 36 | 0.5-1 | 0.94 | 26.11 | 1.80 | 1.40 | 1.29 | 1.43 | 39.72 | 3.60 | 2.82 | 1.28 |
|  | 0.5-2 | 0.94 | 26.11 | 1.81 | 0.96 | 1.88 | 1.43 | 39.72 | 3.60 | 2.54 | 1.42 |
|  | 0.5-3 | 0.94 | 26.11 | 1.80 | 1.38 | 1.30 | 1.39 | 38.61 | 3.60 | 2.74 | 1.31 |
|  | Average | 0.94 | 26.11 | 1.80 | 1.25 | 1.49 | 1.42 | 39.35 | 3.60 | 2.70 | 1.34 |

Table 4.3b - Experimental Pipe Stiffness PS (Average Values)
\(\left.$$
\begin{array}{lccc}\hline \text { Series } & \text { Load Rate } & \begin{array}{c}\text { PS for } \\
\text { 5\% Vert. Defl. } \\
\text { (psi) }\end{array} & \begin{array}{c}\text { PS for } \\
\text { (in./min) }\end{array}
$$ <br>
\hline (a) ADS 48 \& 0.05 \& 20.75 \& Vert. Defl. <br>
\& 0.5 \& 26.91 \& 15.60 <br>

\& 10 \& 35.34 \& 20.44\end{array}\right]\)|  |
| :--- |
|  |

Table 4.4 - Measured PS and SF of HDPE ADS $48 "$ pipes

| Pipe Stiffness (Parallel Plate Test) |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Pipe | ADS |  |  |  |  |  |  |  |  |  |
|  |  |  |  |  |  |  |  |  |  |  |
| ASTM Formula |  |  |  | (Load* 1000 )/(length ${ }^{\text {d }}$ defl $)$ |  |  |  |  |  |  |
| Actual Formula |  | PS | $=$ |  |  |  |  |  |  |  |
|  |  | SF | = | $0.149{ }^{\text {² }}{ }^{\text {3 }}$ * PS |  |  | $\mathrm{r}=24$. | 832 in . |  |  |
|  |  |  |  |  |  |  |  |  |  |  |
| vert defl (LVDT) |  |  |  |  |  |  |  |  |  |  |
| Head Rate | Test \# | vert defl (inch) |  | vert Load (Kips) |  | Length | PS | psi) | SF (lb | $-\mathrm{in}^{2}$ ) |
| in. per minute |  | 5\% | 10\% | 5\% | 10\% | (inch) | 5\% | 10\% | 5\% | 10\% |
| 0.05 |  | -2.4001 | -4.8006 | -1.99219 | -2.99531 | 40 | 20.75 | 15.60 | 47344.36 | 35588.11 |
| 0.5 | 1 | -2.404 | -4.8036 | -2.65469 | -4.02969 | 40 | 27.61 | 20.97 | 62985.26 | 47848.46 |
| 0.5 | 2 | -2.403 | -4.8026 | -2.52031 | -3.82344 | 40 | 26.22 | 19.90 | 59821.59 | 45408.72 |
| 10 |  | -2.4449 | -4.8129 | -3.45625 | -5.24375 | 40 | 35.34 | 27.24 | 80633.28 | 62143.23 |
| 150 |  | -2.4134 | -4.8174 | -4.45 | -7.0125 | 40 | 46.10 | 36.39 | 105171.79 | 83028.11 |

Table 4.5 - Measured PS and SF of HDPE ADS 36" pipes


Table 4.6 - Measured PS and SF of HDPE HANCOR 36" pipes

| Pipe Stiffness (Parallel Plate Test) |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Pipe | HAN 36 |  |  |  |  |  |  |  |  |  |
|  |  |  |  |  |  |  |  |  |  |  |
| ASTM Formula |  | PS | $=\quad 1$ | F/ $\Delta y$ | $\cdots$ |  | $r=19.332$ in. |  |  |  |
| Actual Formula |  | PS | $=\quad$. | (Load*1000)/(length*defl) |  |  |  |  |  |  |
|  |  | SF | $=0$ | $0.149^{*} \mathrm{r}^{\wedge} 3^{*} \mathrm{PS}$ |  |  |  |  |  |  |
| vert defl (LVDT) |  |  |  |  |  |  |  |  |  |  |
| Head Rate | Test\# | vert defl1 (inch) |  | vert Load (Kips) |  | Length | PS | (psi) | SF (lb | b-in ${ }^{2}$ ) |
| in. per minute |  | 5\% | 10\% | 5\% | 10\% | (inch) | 5\% | 10\% | 5\% | 10\% |
| 0.05 |  | -1.8012 | -3.6024 | 1.13086 | 1.92246 | 36 | -20.93 | -14.82 | -22529.36 | -15958.26 |
| 0.5 | 1 | -1.8086 | -3.6024 | 1.5832 | 2.48788 | 36 | -24.32 | -19.18 | -26176.83 | -20651.79 |
| 0.5 | 2 | -1.8012 | -3.6097 | 1.73398 | 2.67636 | 36 | -26.74 | -20.60 | -28787.47 | -22170.93 |
| 10 |  | -1.8086 | -3.6048 | 2.3748 | 3.61874 | 36 | -36.47 | -27.89 | -39265.24 | -30018.51 |

Table 4.7-Measured PS and SF of PVC 36" pipes

| Pipe Stiffness (Parallel Plate Test) |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |  |  |
| Pipe $\quad$ P | PVC 36 |  |  |  |  |  |  |  |  |  |
|  |  |  |  |  |  |  |  |  |  |  |
| ASTM Formula |  | PS | $=$ | F/ $\Delta \mathrm{y}$ |  |  |  |  |  |  |
| Actual Formula |  | PS | $=$ | (Load*1000)/(length*defl) |  |  |  |  |  |  |
|  |  | $\mathrm{SF} \div=$ | 0.1 | 49*ャ^3*PS |  |  | $r=18$. | 570 in. |  |  |
| vert defl ( LVDT ) |  |  |  |  |  |  |  |  |  |  |
| Head Rate | $\begin{gathered} \text { Test } \\ \# \end{gathered}$ | vert defl1 (inch) |  | vert Load (kips) |  | Length | PS (psi) |  | $\mathrm{SF}=\left(\mathrm{lb}-\mathrm{in}^{2}\right)$ |  |
| in. per minute |  | 5\% | 10\% | 5\% | 10\% | (inch) | 5\% | 10\% | 5\% | 10\% |
| 0.5 | 1 | -1.8061 | -3.60481 | 1.28164 | 2.3371 | 13 | 54.58 | 49.87 | 52.080 | 47.586 |
| 0.5 | 2 | -1.80363 | -3.60727 | 1.28164 | 2.22402 | 13 | 54.66 | 47.42 | 52.157 | 45:285 |
| 10 | 1 | -1.81594 | -3.6000 | 1.31933 | 2.2240 | 13 | 56.02 | 47.52 | 53.455 | 45.345 |

Table 4.8 - Measured PS and SF of Steel 36 " pipes

| Pipe Stiffness (Parallel Plate Test) |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |  |  |
| Pipe | STEEL 36 |  |  |  |  |  |  |  |  |  |
|  |  |  |  |  |  |  |  |  |  |  |
| ASTM Formular |  | PS | $=$ | F/- $\Delta \mathrm{y}$ | $\cdots \cdots$ |  | $r=18.0$ | 037 in. |  |  |
| Actual Formular |  | PS, |  | (Load*1000)/(length*defl) |  |  |  |  |  | , |
|  |  | SF | $=$ | $0.149^{*}{ }^{\wedge} 3 * P S$ |  |  |  |  |  |  |
| vert defil |  |  |  |  |  |  |  |  |  |  |
| Head Rate | Test \# | vert defl1 (inch) |  | vert Load (Kips) |  | Length | PS | (psi) | SF (Ib | -in ${ }^{2}$ ) |
| in. per minute |  | 5\% | 10\% | 5\% | 10\% | (inch) | 5\% | 10\% | 5\% | 10\% |
| 0.5 | 1 | -1.8036 | -3.6048 | 2.18632 | 2.75175 | 36 | -33.67 | -21.20 | -29441.82 | -18540.65 |
| 0.5 | 2 | -1.8012 | -3.6073 | 2.07324 | 2.97792 | 36 | -31.97 | -22.93 | -27957.17 | -20050:85 |
| 0.5 | 3 | -1.8012 | -3.6048 | 2.18632 | 3.05331 | 36 | -33.72 | -23.53 | -29482.03 | -20572.49 |
|  |  |  |  |  |  |  |  |  |  |  |

Table 4.9 - Measured PS and SF of Aluminum 36" pipes

| Pipe Stiffness (Parallel Plate Test) |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | ALUM 36 |  |  |  |  |  |  |  |  |  |
| Pipe |  |  |  |  |  |  |  |  |  |  |
| ASTM Formula |  | PS | $=$ | F/ $\Delta \mathrm{y}$ |  |  | $r=18.248$ |  |  |  |
| Actual Formula |  | PS | $=\quad 1$ | (Load*1000)/(length*defl) |  |  |  |  |  |  |
|  |  | SF | $=$ | $0.149{ }^{*}{ }^{\wedge}{ }^{\text {a }}$ | *PS |  |  |  |  |  |
|  |  |  |  |  |  |  |  |  |  |  |
|  |  |  |  |  |  |  |  |  |  |  |
| vert defl1 (LVD |  |  |  |  |  |  |  |  |  |  |
| Head Rate | Test \# | vert defl | 11 (inch) | vert Load | d (Kips) | Length | PS | (psi) | SF (li | b-in ${ }^{2}$ ) |
| in. per minute |  | 5\% | 10\% | 5\% | 10\% | (inch) | 5\% | 10\% | 5\% | 10\% |
| 0.5 | 1 | -1.8036 | -3.6048 | 0.94238 | 1.43242 | 36 | -14.51 | -14.22 | -13140.39 | -12877.55 |
| 0.5 | 2 | -1.8086 | -3.6024 | 0.94238 | 1.43242 | 36 | -14.47 | -11.05 | -13104.57 | -10000.32 |
| 0.5 | 3 | -1.8676 | -3.6024 | 0.94238 | 1.39472 | 36 | -14.51 | -10.75 | -13173.10 | -9737.12 |
|  |  |  |  |  |  |  |  |  |  |  |



Fig. 4.1 - LVDT Experimental Set-up


Fig. 4.2 - Test Specimen


Fig. 4.3 - Views of ADS 48 Pipe Specimen During Parallel Plate Test


Fig. 4.4 - Views of ADS 36 Pipe Specimen During Parallel Plate Test


Fig. 4.5 - Views of Hancor 36 Pipe Specimen During Parallel Plate Test

(a) PVC 36 under Parallel Plate Test

(b) Deformed Shape of PVC 36"

Fig. 4.6 - Views of PVC Pipe Specimen During Parallel Plate Test

(a) Steel 36 under Parallel Plate Test

(b) Deformed Shape of Steel $36^{\prime \prime}$

Fig. 4.7 - Views of Steel Pipe Specimen During Parallel Plate Test

(a) Aluminum 36 under Parallel Plate Test

(b) Deformed Shape of Aluminum 36"

Fig. 4.8 - Views of Aluminum Pipe Specimen During Parallel Plate Test


$\begin{array}{ccccccc}0 & 1 & 2 & 3 & 4 & 5 & 6\end{array}$

Fig. 4.10 - Load versus Vertical and Horizontal Deflections for ADS 36 Under Parallel Plate Test




Fig. 4.12 - Load versus Vertical and Horizontal Deflections for PVC 36, Steel and Aluminum Under Parallel Plate Test


Fig. 4.13 - Comparison of Measured PS Values at 5\% Vertical Deflection for HDPE Pipes at Different Loading Rates


Fig. 4.14 - Comparison of Measured PS Values at 5\% Vertical Deflection for 36" PVC and Metal Pipes at a Loading Rate of $0.5 \mathrm{in} . / \mathrm{min}$.


Fig. 4.15 - Comparison of Measured PS Values at $10 \%$ Vertical Deflection for HDPE Pipes at Different Loading Rates


Fig. 4.16 - Comparison of Measured PS Values at 10\% Vertical Deflection for 36" PVC and Metal Pipes at a Loading Rate of $0.5 \mathrm{in} . / \mathrm{min}$.

## Chapter 5: Flattening Test

### 5.1 Objectives

The objective of this test is to evaluate pipe performance when subjected to flattening between parallel plates until the pipe's inside diameter is reduced by a certain predetermined percent of its original diameter. The specimen is considered to have, passed the test, if no splitting, cracking, breaking, or separation of ribs or seams, or both has occurred. These phenomena should be observed under normal light with unaided eyes.

### 5.2 Apparatus, Test Specimens and Procedure

The hydraulic jack used in the testing has the capability of constant-rate-crosshead movement. The rate of the head approach can be varied and was in the range of 0.05 to 150 in . per minute. The flattening tests were performed in conjunction with the parallel plate tests. Therefore, the apparatus, test specimens and procedure are identical to those pertaining to the parallel plate tests (see chapter 4 for details). For the flattening tests, no continuous load-deflection readings were recorded. However, for each of the flattened position considered, observations were made to identify splitting, cracking, breaking, or separation of ribs or seams, or both.

### 5.3 Observations on Behavior of Pipes Flattened According to Standards

AASHTO M294 (for HDPE) and ASTM F949-00 (for PVC) require that HDPE and PVC pipes be flattened between parallel plates until the inside diameter is reduced by:

- HDPE: 20\%
- PVC: [100-3.43 ID/(OD - ID)], that is approximately $62 \%^{\circ}$ for the PVC pipe under investigation.
- Note that no flattening test is required for metal pipes.

The following observations were made within the vertical deflection ranges outlined above:
(a) HDPE Pipes (AASHTO M294)

- Less than $15 \%$ vertical deflection

No wall buckling and other unsymmetrical deformations were observed for all the HDPE pipes tested. All the pipes deformed in an elliptical shape.

- At $15 \%$ vertical deflection

At a vertical deflection of $15 \%$ o of the diameter, wall buckling was observed around the springline in the outside wall in the case of HDPE Hancor pipe (Fig. 5.1).

- At $20 \%$ vertical deflection
- ADS 48: Scattered local wall buckling was observed around the pipe's springline (Fig. 5.2).
- ADS 36: Scattered local wall buckling was observed in only certain areas of pipe's springline.
- Hancor 36: Wall buckling, which was observed at a vertical deflection of $15 \%$ diameter on the exterior surface, became more noticeable.


## (b) PVC Pipe (ASTM F949-00)

- Up to $20 \%$ vertical deflection
- No wall buckling and other unsymmetrical deformations were observed for all
- PVC pipes. All the pipes deformed in an elliptical shape (Fig. 5.3).
- At $30 \%$ vertical deflection
- Most PVC specimens tested at loading rates of $0.05,0.5$ and $150 \mathrm{in} . / \mathrm{min}$ exhibited wall rupture either at the invert or at the crown at vertical deflections ranging from 30 to $36 \%$. The pipe failed suddenly with a loud noise as a result of the wall rupture (see Figs. 5.4a and b).
- At 36\% vertical deflection
- Fig. 5.5a to c present some of the views of the pipe at a vertical deflection of $36 \%$ of the diameter. From these figures, reverse curvature at the crown and at the invert, as well as inside wall buckling were clearly observed.
- At $60 \%$ vertical deflection
- All PVC specimens tested at a load rate of $10 \mathrm{in} . / \mathrm{min}$. could be flattened up to $60 \%$ deflection without rupture (see Fig. 5.22d).


### 5.4 Observations on Behavior of Pipes Flattened Up to 60\%

### 5.4.1 HDPE ADS 48" Pipe

Flat invert/crown was attained at a deflection of approximately $30 \%$ (Fig. 5.6b). No reverse curvature was observed until 42\% deflection (Fig. 5.6c and d).

With increasing vertical deflection to $30 \%$ of the pipe's diameter, the extent of wall buckling gradually increased and developed along the specimen length (Figs. 5.2b and 5.7). A crack was observed on the inside wall as seen in Fig. 5.7. Fig. 5.8 shows a bulge in a portion of the wall. Wall buckling occurred primarily in the region of the pipe's springline and at a vertical deflection of $42 \%$ of the diameter (Fig. 5.9). No buckling was observed on the outside of the pipe wall.

### 5.4.2 HDPE ADS 36" Pipe

Flat invert/crown prior to reverse curvature was attained at a deflection of approximately $30 \%$ (Fig. 5.10b). Reverse curvature initiated at $36 \%$ deflection (Fig. 5.10c). As the vertical deflection increased to $30 \%$ of the diameter, the area of wall buckling gradually increased. Excessive wall buckling was observed mainly on the springline of the left and right inside surfaces (Figs. 5.11 to 5.14 ). The test pipe specimen was then compressed to a vertical deflection of $59 \%$ of the diameter. The distance between some corrugations became longer, while the others shortened (Fig. 5.15). The crown region of the test pipe went into reverse curvature prior to reaching the vertical deflection of $59 \%$ of the diameter (Fig. 5.10d). However, the invert region of the specimen maintained almost a flat surface except at the center portion of the supporting steel plate (Fig. 5.16).

### 5.4.3 HDPE HANCOR 36" Pipe

Flat invert/crown prior to reverse curvature was attained at a deflection of approximately $45 \%$ (Fig. 5.17a). Reverse curvature initiated at $20 \%$ deflection (Fig. 5.17b) and was clearly apparent as the. deflection attained $30 \%$ (Figs. 5.17 c). When the vertical deflection value increased to $59 \%$ of the diameter, both the pipe region in the invert as well as at the crown
exhibited almost identical reverse curvature shapes (Fig. 5.17d). In general, pipe properties of HDPE RANCOR 36" pipe are similar to HDPE ADS 36" pipe. The deformation behavior of HDPE RANCOR 36" pipes are displayed in Figs. 5.17 to 5.21.

In the case of HDPE RANCOR 36" pipes, wall buckling on the exterior surface became noticeable even at a vertical deflection of $15 \%$ o of the diameter. Moreover, a break in the rib was evident on the exterior surface of the pipe wall (Fig. 5.21).

### 5.4.4 PVC 36" Pipe

Flat invert/crown prior to reverse curve was attained at a deflection of approximately $30 \%$ (Fig. 5.22 b ). All PVC specimens tested at a load rate of $10 \mathrm{in} . / \mathrm{min}$. could be flattened up to $60 \%$ deflection without rupture (Fig. 5.22d). However, as outlined earlier, most PVC specimens tested at loading rates of $0.05,0.5$ and $150 \mathrm{in} . / \mathrm{min}$. exhibited wall, rupture either at the invert or at the crown at a vertical deflection ranging from 30 to $36 \%$.

### 5.4.5 Steel and Aluminum 36" Pipes

Both the aluminum and steel pipes did not exhibit reverse curvature as clearly as the HDPE pipes. Fig. 5.23a) to d) shows views of the behavior of the aluminum, pipe under parallel plates for different deflection levels. Note the highly unsymmetrical deflected shapes of the aluminum pipe specimen. Similarly, Fig 5.24a) to d) shows that the behavior of the steel 36 " pipe is similar to that of the aluminum 36" pipe.

### 5.5 Conclusions

The following conclusions are of interest with regard to the flattening tests:
(a) All the HDPE pipes passed the flattening test since no splitting, cracking, breaking, or separation of ribs or seams, or both, were observed under normal light with unaided eyes.
(b) The PVC pipes tested at $0.05,0.5$ and $1.50 \mathrm{in} . / \mathrm{min}$. ruptured either at the crown or at the invert after the occurrence of reverse curvature, at a vertical deflection ranging from 30\% to $36 \%$. However, only the PVC pipe specimens tested at $10 \mathrm{in} . / \mathrm{min}$. could be flattened up to $60 \%$ vertical deflection without failure. These pipe specimens did not experience any splitting, cracking, breaking, or separation of ribs or seams, or both, and therefore passed the flattening test.

Also, results indicated the following:
(c) For HDPE pipes, up to $15 \%$ vertical deflection, no wall buckling or unsymmetrical deformed shapes were observed.
(d) At $15 \%$ vertical deflection, wall buckling initiated at the area of the springline in the outside wall of Hancor 36.
(e) At $20 \%$ vertical deflection, scattered local wall buckling initiated in the area of the springline in the inside wall for all HDPE pipes.
(f) PVC pipe performed well up to $20 \%$ vertical deflection. No wall buckling or unsymmetrical deformed shapes were observed. Between 20 to $30 \%$ vertical deflection, inside wall buckling was noticeable.


Fig. 5.1 - Wall Buckling on the Outside of Hancor 36 Pipe at Vertical Deflection of $\mathbf{1 5 \%}$


Fig. 5.2a - Deformed Shape of HDPE ADS 48" Pipe at Vertical Deflection of 20\% Diameter


Fig. 5.2b - Scattered Wall local Buckling at Vertical Deflection of $\mathbf{2 0 \%}$ Diameter


Fig. 5.3-Deformed Elliptical Shape of PVC 36" Pipe at the Vertical Deflection of $20 \%$


Fig. 5.4a - Failure of PVC Due to Wall Rupture in the Invert Region with Reverse Curvature at the Vertical Deflection of 30\% Diameter


Fig. 5.4b - Close up View of invert Rupture of PVC 36" Pipe at the Vertical Deflection of $\mathbf{3 0 \%}$ Diameter


Fig. 5.5a - Deformed Elliptical Shape of PVC 36" Pipe at the Vertical Deflection of 36\%


Fig. 5.5c - Wall Buckling at Springline of PVC 36" Pipe at the Vertical Deflection of 36\%


Fig. 5.6-Deformed Shapes of HDPE ADS 48" Pipe Under Parallel Plates at Different Vertical Deflections


Fig. 5.7 - Extensive Inside Wall Local Buckling and One Wall Cracking at Vertical Deflection of 30\% Diameter ADS48


Fig. 5.9 - Wide-Spread Wall Buckling at Vertical Deflection of $\mathbf{4 2 \%}$ Diameter-ADS48


Fig. 5.10 - Deformed Shapes of HDPE ADS 36" Pipe Under Parallel Plates at Different Vertical Deflections


Fig. 5.11 - Inside Wall Deformation at the Vertical Deflection of 20\% Diameter - ADS36


Fig. 5.13 - Outside Wall Buckling at the Vertical Deflection of 30\% Diameter - ADS36


Fig. 5.12 - Wide Spread Wall Buckling Forming a Line in the Area of Springline at the Vertical Deflection of 30\% Diameter - ADS36


Fig. 5.14 - Wide Spread Inside Wall Buckling at the Vertical Deflection of 36\% Diameter - ADS36


Fig. 5.15 - Deformation of the Outside Surface at Pipe Springline at the Vertical Deflection of $59 \%$ Diameter ADS36


Fig. 5.16 - Lightly Reversed Curvature of Invert Region at the Vertical Deflection of 59\% DiameterADS36


Fig. 5.17 - Deformed Shapes of HDPE Hancor 36" Pipe Under Parallel Plates at Different Deflections


Fig. 5.18 - Inside Wall Buckling at the Springline of HDPE Hancor $36^{\prime}$ Pipe at the Vertical Deflection of 30\% Diameter


Fig. 5.20 - Extensive Wall Buckling on the Exterior Surface of HDPE Hancor 36" Pipe at the

Vertical Deflection of $59 \%$ Diameter


Fig. 5.19 - Deformation of the Invert Region of HDPE Hancor 36" Pipe at the Vertical Deflection of $\mathbf{3 6 \%}$ Diameter


Fig. 5.21 - Breaking of Rib of HDPE Hancor 36" Pipe at the Vertical Deflection of 59\% Diameter


Fig. 5.22 - Deformed Shapes of PVC 36" Pipe Under Parallel Plates at Different Vertical Deflections


Fig. 5.23 - Deformed Shapes of Aluminum 36" Pipe Under Parallel Plates at Different Vertical Deflections


Fig. 5.24 - Deformed Shapes of Steel 36" Pipe Under Parallel Plates at Different Vertical Deflections

## Chapter 6: Curved Beam Stiffness Test

### 6.1 Scope and Objectives

Pipe stiffness (PS) obtained from the parallel-plate loading test (ASTM D2412), is widely used in the modified Spangler equation to obtain an approximate pipe deformation. In the field, reacting forces in response to all external forces are shared by the pipe and the soil element of the pipe-soil composite structure, but in the ASTM D2412 test, the only restraint is in the vertical direction (Gabriel \& Goddard, 1999). In the parallel-plate loading test, wall bending is considered as the most dominant effect while ring compression is the least. An alternative measure of pipe stiffness has been proposed by Gabriel \& Goddard (1999). In their method, a curved specimen, subtending an arc of $90^{\circ}$, cut from a production run pipe, is loaded at both end (pined-pined constraints) with external compressive forces. At the same magnitude of loading, the curved beam is believed to have less bending moment in the walls at the springline than the parallel plate test. Thus, a greater proportion of the wall's compression and a lesser proportion of the wall's bending moment makes up the response of the curved beam than that of the parallel plate test. Hence, it is claimed that the curved beam stiffness test approximates more closely the field condition of the buried pipe.

The objective of this section is to find the pipe stiffness by using the above curved beam approach. Investigating the pipe behavior (Load vs. deflection, Deflection vs. strains, etc.) under the curved beam conditions for different loading rates is another objective pursued in this part of the study.

### 6.2 Experimental Program Apparatus

A hydraulic jack with a varying rate of crosshead movement is used to apply external forces for the tests. A load cell is used to continuously record, this external compressive force with time before and during the periods of loading. The reaction frames were made from 3/8 I-beam steel structure and 3/8 steel plates. A special device is fabricated and welded to the testing frame to hold the thermoplastic specimen. Typical setups for thermoplastic and metal specimens, can be seen in Figs. 6.4 to 6.9. Two deflectometers (LVDT) are used to continuously measure both vertical and horizontal displacements of the test specimens, Fig.
6.1. Strain gages were also installed on one specimen (load rate of $0.5 \mathrm{in} . / \mathrm{min}$.) of each series to monitor the strains on the concave wall (inner wall) and the convex wall (outer wall) of the pipe, as shown in Fig. 6.2.

## Test specimen

The test specimens are cut from randomly selected sections. The longitudinal length of the test specimens cut from ADS $48^{\prime \prime}$ pipes is 40 inches. The length of 36 inches was chosen for the test specimens cut from $36-\mathrm{in}$. diameter HDPE, PVC, aluminum and steel pipes. The longitudinal edges of the test specimen were made to have a smooth plane, free of jagged edges and burrs. Fig. 6.3 illustrates the test specimens cut from a $40-\mathrm{in}$. (48-in. diameter pipe) and 36-in. (36-in. diameter pipe) width ring. Table 6.1 presents the geometric properties of the specimens.

## Test Procedure

The average of three measurements of the longitudinal length at mid- and quarter-points of the arc of the curved specimen is first determined. The test is conducted by applying a nearly instantaneous load to the longitudinally cut edges of the $90^{\circ}$ section of the specimen until $10 \%$ shortening of the chord connecting the longitudinal edges is attained. Four different rates of the crosshead $0.05,0.5,10$, and $150 \mathrm{in} . / \mathrm{min}$. were used during the loading of each pipe. Load and displacement readings were continuously recorded. The load versus displacement for various rates of the cross-head movement was plotted for different types of specimens.

## Test Program

Details of the curved-beam test program carried out in this study are presented in Table 6.2.

### 6.3 Time-Independent Pipe Stiffness, K(0)

The time-independent pipe stiffness, $\mathrm{K}(0)$ in pounds per square inch is calculated using the following procedure:
i) Percent displacement (\% displacement) is calculated by dividing the change in vertical displacement of the chord length of the specimen, by its original chord length, and then multiplying by 100 .
ii) "The time-dependent pipe stiffness, $\mathrm{K}(\mathrm{t})$ is calculated from the load-vertical displacement data at five points from $2 \%$ to $4 \%$ displacements as given by the equation below:

$$
\begin{equation*}
\mathrm{K}(\mathrm{t})=\mathrm{F} /(\mathrm{L} \cdot \Delta \mathrm{y}) \tag{6.1}
\end{equation*}
$$

Where, $\mathrm{F}=$ measured load at the specified deflection on the full length of the curved beam specimen in pounds (lbs.), $\Delta \mathrm{y}=$ the specified displacement for each percent deflection in inches (in),. $\mathrm{L}=$ the length of test curved beam test specimen in inches (in.).
iii) The time-dependent stiffness values, $K(t)$ versus the $\%$ displacements are plotted.
iv) A linear least squares curve is fitted through the points between $2 \%$ and $4 \%$ displacements.
v) The straight line is extrapolated linearly to the $y$-axis intercept giving the time-dependent $\mathrm{K}(0)$.

### 6.4 Presentation and Discussions of Results

## Overall Behavior

Views of specimens ready for testing (i.e., initial state of deformation), during testing (i.e., deformed state), and at failure, are respectively presented in Figs. 6.4 to 6.9. for ADS 48", ADS 36", Hancor 36 ", PVC 36 ", Steel $36^{\prime \prime}$ and Aluminum 36 ". Table 6.3 summarizes the characteristic values corresponding to $5 \%$ and $10 \%$ vertical ring deflection. It is observed that the vertical/horizontal deflection ratio $\left(\Delta_{\mathrm{V}} / \Delta_{\mathrm{x}}\right)$ is higher for $5 \%$ vertical deflection than for $10 \%$ vertical deflection. Also, the ratio $\left(\Delta_{\mathrm{v}} / \Delta_{\mathrm{x}}\right)$ did not vary as the loading rate was varied. For $5 \%$ vertical deflection, the average value of $\left(\Delta_{v} / \Delta_{x}\right)$ was 1.09 for ADS 48", 1.08 for ADS 36", 1.20 for Hancor 36, 1.14 for PVC 36", 1.04 for Steel $36^{\prime \prime}$, and 1.11 for Aluminum 36".

## Load versus Deflections

The curves representing the applied load versus the vertical and horizontal deflections are shown in Figs. 6.10 to 6.15 for ADS 48", ADS 36", Hancor 36", PVC 36 ", Steel $36^{\prime \prime}$ and Aluminum 36", respectively.

## Load versus Strains and Vertical Deflection versus Strains

The curves representing the applied load versus strain readings on the one hand and the vertical deflection versus the strain readings on the other, are shown in Figs. 6.16 to 6.21 for respectively ADS 48", ADS 36", Hancor 36", PVC 36", Steel 36" and Aluminum 36". Table 6.4 summarizes the values of the strains recorded in the different pipes for $5 \%$ and $10 \%$ vertical deflections and the corresponding applied loads. From this table, the following observations can be formulated: (a) At a vertical deflection of $5 \%$ of the diameter, the tensile strains at the outer surface were similar for all the pipes and varied between $15,292 \mu \varepsilon$ and $18,026 \mu \varepsilon$; (b) The compressive strains varied considerably from one type of pipe to another. At the vertical deflection of $5 \%$ of the diameter, it was equal to $17 \mu \varepsilon$ for ADS 48 ", $6,627 \mu \varepsilon$ for ADS $36^{\prime \prime}, 14,078 \mu \varepsilon$ for Hancor $36^{\prime \prime}, 12,169 \mu \varepsilon$ for PVC 36 ", $3,710 \mu \varepsilon$ for Steel $36^{\prime \prime}$ and $1,395 \mu \varepsilon$ for Aluminum 36".

## Time-Independent Pipe Stiffness

A linear least square fit of time dependent stiffness, $K(t)$, versus the vertical displacement in percentage of the original chord length, is presented in Figs. 6.22a to 6.22 f for ADS 48", ADS 36", Hancor 36", PVC 36", Steel 36" and Aluminum 36", respectively.
The time-independent pipe stiffness, $K(0)$ corresponding to the $y$-axis intercepts of the curves was determined for each of the twelve tests of the program, and the results are summarized in Table 6.5. The average pipe stiffness (PS) values obtained from the parallel plate tests (see chapter 4) are also given between parentheses for comparison purposes. From this table, the following observations can be made: (a) for HDPE pipes, the time-independent pip e stiffness, $K(0)$ increased as the loading rate increased; (b) for PVC, no noticeable variation of $K(0)$ with the loading rate was observed; (c) the $K(0)$ values are 2 to 3 times greater than the PS values obtained from the parallel plate tests.

### 6.5 Conclusions

The following conclusions can be drawn:
(a) The $\mathrm{K}(0)$ values increase with the loading rate. They are 2 to 3 times greater than the PS values determined by the parallel plate test.
(b) For a vertical deflection of $5 \%$ of the diameter, the tensile strain in the outer wall was approximately equal to $18,000 \mu \varepsilon$ (i.e., $1,980 \mathrm{psi}$ ) for all $\mathrm{HDPE}, 17,000 \mathrm{gs}$ (i.e., 6,800 psi ) for PVC and $16,000 \mu \varepsilon$ (i.e., 60 ksi for steel and 21 ksi for aluminum) for metal pipes. The compressive strain in the inner wall ranged from 0 to $14,000 \mathrm{gs}$ for plastic pipes and from $1,400 \mu \varepsilon$ to $3,700 \mu \varepsilon$ for metal pipes.

Table 6.1 - Geometric Properties of Arch Specimens

| Pipe Type | DD | OD | Chord Length |
| :--- | :---: | :---: | :---: |
|  | (in.) | (in.) | (OD $/ \sqrt{2}$ ) |
| ADS 48 | 47.02 | 52.32 | (in.) |
| ADS 36 | 36.00 | 41.57 | 40.00 |
| HANCOR 36 | 35.71 | 41.48 | 29.39 |
| PVC 36 | 35.51 | 38.77 | 29.33 |
| STEEL 36 | 35.84 | 36.31 | 27.41 |
| ALUMINUM 36 | 35.85 | 37.28 | 25.68 |

Table 6.2 - Curved-Beam Test Program

| Type of Pipe | Load Rate <br> (in/min.) | Number of Tests |
| :--- | :---: | :---: |
| ADS 48 | 0.05 | 2 |
|  | 0.5 | 2 |
|  | 10 | 2 |
|  | 150 | 2 |
| ADS 36 | 0.05 | 2 |
|  | 0.5 | 2 |
|  | 10 | 2 |
|  | 150 | 2 |
| Hancor 36 | 0.05 | 2 |
|  | 0.5 | 2 |
|  | 10 | 2 |
| PVC 36 | 150 | 2 |
| Steel 36 | 0.5 | 2 |
| Aluminum 36 | 0.5 | 2 |

Table 6.3-Summary of Experimental Results

| Series | Load Rate/Specimen \# | 5\% Vertical Deflection |  |  |  | 10\% Vertical Deflection |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Load | $\Delta_{v}$ | $\Delta_{\mathrm{x}}$ | $\Delta_{\mathrm{v}} / \Delta_{\mathrm{x}}$ | Load | $\Delta_{v}$ | $\Delta_{\mathrm{x}}$ | $\Delta_{V} / \Delta_{x}$ |
|  | (in./min.) | (Lbs/in.) | (in.) | (in.) | , | (Lbs/in.) | (in.) | (in.) | - |
| ADS 48 | $0.05-1$ | 84.26 | 2.402 | 2.24 | 1.07 | 88.39 | -4.800 | 4.05 | 1.19 |
|  | 0.05-2 | 70.00 | 2.401 | 2.34 | $1.03(1.05)^{(\mathrm{a})}$ | 73.32 | 4.800 | 4.06 | 1.18(1.18) |
|  | 0.5-1 | 104.34 | 2.408 | 2.15 | 1.12 | 110.31 | 4.365 | 3.75 | 1.16 |
|  | 0.5-2 | 94.26 | 2.406 | 2.28 | 1.06(1.09) | 97.77 | 3.258 | 2.95 | 1.10 (1.13) |
|  | 10-1 | 144.80 | 2.399 | 2.02 | 1.19 | 151.37 | 4.761 | 3.88 | 1.23 |
|  | 10-2 | 140.08 | 2.390 | 2.14 | $1.12(1.15)$ | 142.15 | 4.776 | 3.91 | 1.22 (1.22) |
|  | 150-1 | 189.65 | 2.343 | 2.185 | 1.07 | 197.38 | 4.791 | 4.07 | 1.18 |
|  | 150-2 | 180.20 | 2.355 | 2.228 | - 1.06 (1.06) | 178.48 | 4.825 | 4.20 | 1.15 (1.16) |
| ADS 36 |  | 82.03 | 1.804 | 1.77 | 1.02 | 82.25 | 3.605 | 3.20 | 1.13 |
|  |  | 77.04 | 1.801 | 1.70. | 1.06 (1.04) | 77.17 | 3.602 | 3.01 | 1.20 (1.16) |
|  | 0.5-1 | 107.60 | 1.806 | 1.59 | 1.14 | 110.33 | 3.607 | 3.04 | 1.19 |
|  | 0.5-2 | 112.63 | 1.801 | 1.76 | 1.02 (1.08) | 109.45 | 3.60 | 3.19 | 1.13 (1.16) |
|  | 10-1 | 142.80 | 1.804 | 1.58 | $1.14$ | 139.45 | 3.66 | 3.00 | $1.22$ |
|  | 10-2 | 148.48 | 1.779 | 1.38 | 1.29 (1.22) | 145.44 | 3.64 | 2.91 | 1.25 (1.24) |
|  | 150-1 | 148.48 | 1.779 | 1.383 | $1.29$ | 146.31 | 4.791 | 4.07 | 1.17 |
|  | 150-2 | 146.66 | 1.897 | 1.708 | 1.11 (1.20) | 138.54 | 4.825 | 4.20 | 1.15 (1.16) |
| HANCOR 36 | 0.05-1 | 59.94 | 1.803 | 1.66 | 1.09 | 57.42 | 3.605 | 2.98 | 1.21 |
|  | 0.05-2 | 61.50 | 1.801 | 1.58 | $1.14(1.11)$ | 62.19 | 3.602 | 2.90 | 1.24 (1.2) |
|  | 0.5-1 | 73.35 | 1.801 | 1.51 | $1.20$ | 72.66 | 3.605 | $\begin{array}{r} \hline 2.84 \\ 2.82 \\ \hline \end{array}$ | $1.27$ |
|  | 0.5-2 | 76.47 | 1.801 | 1.52 | 1.20 (1.20) | 80.16 | 3.602 |  | 1.28 (1.27) |
|  | 10-1 | 81.11 | 1.784 | 1.37 | 1.30 | 77.34 | 3.629 | $\begin{array}{r} 2.71 \\ 2.85 \\ \hline \end{array}$ | 1.34 |
|  | 10-2 | 84.76 | 1.767 | 1.51 | 1.17 (1.24) | 76.30 | 3.585 |  | 1.26 (1.30) |
|  | 150-1 | 111.24 | 1.695 | 1.54 | 1.10 | 87.85 | 3.15 | 2.67 | 1.18 |
|  | 150-2 | 114.11 | 1.730 | 1.45 | 1.19 (1.15) | 107.20 | 3.69 | 2.95 | 1.25 (1.22) |
| PVC 36 | 0.05-1 | 85.11 | 1.801 | 1.74 | \% 1.04 | 99.57 | 3.602 | $\begin{aligned} & 3.22 \\ & 2.23 \\ & \hline \end{aligned}$ | 1.12 |
|  | 0.05-2 | 126.43 | 1.801 | 1.64 | 1.10 (1.07) | 119.14 | 2.53 |  | 1.13 (1.13) |
|  | $0.5-1$ | 132.07 | 1.801 | 1.58 | 1.14 | 129.77 | 3.607 | 2.93 | 1.23 |
|  | 0.5-2 | 151.00 | 1.804 | 1.58 | 1.14 (1.14) | 152.17 | 3.604 | 2.92 | 1.23 (1.23) |
|  | 10-1 | 164.97 | 1.813 | 1.62 | 1.12 | 188.15 | 3.597 | $\begin{array}{r} 2.93 \\ 2.89 \\ \hline \end{array}$ | $\begin{aligned} & 1.23 \\ & 1.25(1.24) \\ & \hline \end{aligned}$ |
|  | 10-2 | 158.12 | 1.828 | 1.58 | 1.16 (1.14) | 175.22 | 3.620 |  |  |
|  | 150-1 | 162.54 | 1.767 | 1.52 | 1.16 | 205.77 | 3.718 | 2.92 | 1.27 |
|  | 150-2 | 162.85 | 1.887 | 1.62 | 1.16 (1.16) | 191.54 | 3.607 | 2.84 | 1.27 (1.27) |
| Steel 36 | 0.5-1 | 60.50 | 1.801 | 1.77 . | 1.02 | 42.32 | 3.602 | $\begin{aligned} & 3.18 \\ & 3.05 \\ & \hline \end{aligned}$ | $1.13$ |
|  | 0.5-2 | 49.31 | 1.806 | 1.71 | 1.06 (1.04) | 32.68 | 3.605 |  | 1.18 (1.16) |
| Aluminum 36 |  | 25.56 | 1.806 | 1.64 | 1.10 | 20.40 | 3.605 | $\begin{aligned} & 3.02 \\ & 2.95 \\ & \hline \end{aligned}$ | $1.19$ |
|  |  | 21.57 | 1.804 | 1.62 | 1.11 (1.11) | 15.45 | 3.602 |  | 1.22 (1.21) |

Note: ${ }^{(a)}$ values between parentheses are average values for the load rate considered.

Table 6.4-Strain Readings for 5\% and 10\% Vertical Deflection (Loading Rate $=0.5 \mathrm{in} . / \mathrm{min}$ )
$\left.\begin{array}{lcccc}\hline \text { Pipe Type } & \begin{array}{c}\text { Vertical } \\ \text { Deflection } \\ (\%)\end{array} & \begin{array}{c}\text { Vertical Deflection } \\ \text { (in.) }\end{array} & \begin{array}{c}\text { Inner SG } \\ (\mu \varepsilon)\end{array} & \begin{array}{c}\text { Outer SG } \\ (\mu \varepsilon)^{2}\end{array} \\ \hline \text { ADS 48 } & 5 & 2.406 & -17 & 18026 \\ & 10 & 3.258^{(\mathrm{a}}\end{array}\right)$

Note: ${ }^{\text {(a) }}$ Maximum/minimum recorded values

Table 6.5 - Time-Independent Pipe Stiffness K (0)

|  | Load Rate | Test No. | K(0)* | $\mathrm{R}^{2+4}$ |
| :---: | :---: | :---: | :---: | :---: |
| (a) ADS 48 | 0.05 | 1 | 62.74 | 0.99 |
|  |  | 2 | 53.04 | 0.99 |
|  |  | Mean | 57.89 (20.75) |  |
|  | 0.5 | 2 | 74.14 | 0.99 |
|  |  | 1 | 76.31 | 0.99 |
|  |  | Mean | 75.22 (26.91) |  |
|  | 10 | 1 | 113.26 | 0.997 |
|  |  | 2 | 109.60 | 0.99 |
|  |  | Mean | 111.43 (35.34) |  |
|  | 150 | 1 | 140.8 | 0.99 |
|  |  | 2 | 144.7 | 0.99 |
|  |  | Mean | 142.8 (46.10) |  |
| (b) ADS 36 | 0.05 | 1 | 93.08 | 0.99 |
|  |  | 2 | 87.46 | 0.99 |
|  |  | Mean | 90.27 (26.85) |  |
|  | 0.5 | 1 | 113.66 | 0.99 |
|  |  | 2 | 132.5 | 0.99 |
|  |  | Mean | 123.08 (37.09) |  |
|  | 10 | 1 | $\therefore 158.27$ | 0.99 |
|  |  | 2 | 153.08 | 1.00 |
|  |  | Mean | 155.68 (49.28) |  |
|  | 150 | 1 | 153.23 | 1.00 |
|  |  | 2 | 163.12 | 0.99 |
|  |  | Mean | 158.18 (57.51) |  |
| (c) Hancor 36 | 0.05 | 1 | 70.80 | 0.99 |
|  |  | 2 | 65.62 | 0.99 |
|  |  | Mean | 68.21 (20.93) |  |
|  | 0.5 | 1 | 79.72 | 1.00 |
|  |  | 2 | 69.00 | 1.00 |
|  |  | Mean | 74.36 (25.53) |  |
|  | 10 | 1 | 82.95 | 1.00 |
|  |  | 2 | 93.2 | 1.00 |
|  |  | Mean | 88.08 (36.47) |  |
|  | 150 | 1 | 136.70 | 0.99 |
|  |  | 2 | 121.26 | 1.00 |
|  |  | Mean | $128.98(-)$ |  |
| (d) PVC 36 | 0.05 | 1 | 83.77 | 0.99 |
|  |  | 2 | 141.36 | 1.00 |
|  |  | Mean | 112.56 (-) |  |
|  | 0.5 | 1 | 117.98 | 1.00 |
|  |  | 2 | 133.94 | 1.00 |
|  |  | Mean | 125.96 (54.62) |  |
|  | 10 | 1 | 137.12 | 1.00 |
|  |  | 2 | 130.70 | 1.00 |
|  |  | Mean | $133.91(-)$ |  |
|  | 150 | 1 | 124.35 | 0.99 |
|  |  | 2 | 130.78 | 1.00 |
|  |  | Mean | 127.56 (-) |  |
| (e) Steel 36 | 0.5 | 1 | 110.0 | 0.98 |
|  |  | $\stackrel{2}{\text { Mean }}$ | 90.8 $1002(33.12)$ | 0.92 |
|  |  | Mean | 100.2 (33.12) |  |
| (f) Aluminum 36 | 0.5 | 1 | 34.65 | 1.00 |
|  |  | 2 | 34.78 | 0.97 |
|  |  | Mean | 34.71 (13.03) |  |

*The average pipe stiffness (PS) values obtained from the parallel plate tests are given between parentheses.
$* *$ Coefficient of Determination


Figure 6.1-Schematic of Locations of Vertical and Horizontal Deflectometers


Figure 6.2 - Schematic of Locations of Strain Gages on the Inner and Outer Pipe Walls

a) $\mathbf{4 8}$ in. Diameter Pipe

b) $\mathbf{3 6}$ in. Diameter Pipe

Figure 6.3-Pipe Ring Cut Into 4 Test Specimens


Fig. 6.4 - Views of ADS 48 Specimen at Different Deformed Shapes During Curved-Beam Tests


Fig. 6.5 - Views of ADS 36 Specimen at Different Deformed Shapes During Curved-Beam Tests

a) Initial shape prior to testing

c) Deformed shape at failure

b) Deformed shape during testing

d) Corrugation buckling at failure

Fig. 6.6-Views of HANCOR 36 Specimen at Different Deformed Shapes During Curved-Beam Tests


Fig. 6.7 - Views of PVC 36 Specimen at Different Deformed Shapes During Curved-Beam Tests


Fig. 6.8 - Views of STEEL 36 Specimen at Different Deformed Shapes
During Curved-Beam Tests


Fig. 6.9 - Views of ALUMINUM 36 Specimen at Different Deformed Shapes During Curved-Beam Tests

6-17






Fig. 6.12 - Load vs Vertical and Horizontal Deflections for Hancor 36
6-19


(u) Noוำตางコ
Fig. 6.13 - Load vs Vertical and Horizontal Deflections for PVC 36


Fig. 6.14-Load vs Vertical and Horizontal Deflections for Steel 36


Fig. 6.15-Load vs Vertical and Horizontal Deflections for Aluminum 36
(a) Load vs Strains

Curved Beam Test \# 2
ADS48-0.5 in./min.

(b) Vertical Deflection vs Strains


Fig. 6.16-Load and Vertical Deflection vs Strains for ADS 48
(a) Load vs Strains

(b) Vertical Deflection vs Strains


Fig. 6.17-Load and Vertical Deflection vs Strains for ADS 36
(a) Load vs Strains

Curved Beam Test \# 2 Hancor36-0.5 in./min.

(b) Vertical Deflection vs Strains


Fig. 6.18 - Load and Vertical Deflection vs Strains for Hancor 36
(a) Load vs Strains

(b) Vertical Deflection vs Strains

Curved Beam Test 2 (PVC 36")/0.5 in./min


Fig. 6.19 - Load and Vertical Deflection vs Strains for PVC 36
(a) Load vs Strains

## Curved Beam Test \# 2

Steel36-0.5 in./min.

(b) Vertical Deflection vs Strains


Fig. 6.20-Load and Vertical Deflection vs Strains for Steel 36

## (a) Load vs Strains


(b) Vertical Deflection vs Strains


Fig. 6.21 - Load and Vertical Deflection vs Strains for Aluminum 36


Fig. 6.22 - Typical Curve Fittings for K(0)

## Chapter 7: Joint Integrity Test

### 7.1 Objectives

The objective of this test is to identify any damage of the joints of the HDPE and PVC pipes at the minimum specified deflection of $20 \%$ of the nominal pipe diameter. The evaluation of PS of jointed specimens under parallel plate is another objective pursued in this part of the study. The maximum radial distances between pipe and fittings, or between bell and spigot are also recorded during the test and after load removal.

### 7.2 Experimental Program Apparatus

The hydraulic jack used in the testing has the capability of constant-rate-crosshead movement. The rate of head approach was 0.5 in . per minute. The load could be applied to the flexible pipe through two parallel flat, smooth, and clean steel bearing plates resting over wooden planks. These wooden planks were positioned between the pipe crown and the steel plate on either side of the joint to enable uniform load application similar to the parallel plate testing. The steel plate at the top is welded to a WF steel beam and the load is applied to the center of the WF beam. The thickness of the plates was about 0.875 in ., so as to minimize bending or deformation of the plate during testing. The plate length was slightly larger than he specimen length, and the plate width was approximately equal to the pipe contact width at maximum pipe deflection plus 6.0 in . The change in inside diameter was measured using LVDTs in three directions: parallel and perpendicular to the direction of loading, and $45^{\circ}$ to the direction of loading. The LVDTs were used to measure to the nearest 0.01 in . Fig. 7.1 shows atypical experimental set-up for the test.

## Test Specimens

The test specimens (Fig. 7.2) included two sizes: $36-\mathrm{in}$. and $48-\mathrm{in}$. diameters. The $36-\mathrm{in}$. diameter pipes consisted of HDPE and PVC pipes. One type of HDPE pipe was of 48 in . diameter. The test specimens had a total length of 36 inches for the $36-\mathrm{in}$. diameter pipes and 48 inches for the $48-\mathrm{in}$. diameter pipe. The ends of specimens were. cut square and free of burrs and jagged edges.

The outside diameter (OD) and the inside diameter (ID) of the test specimens as well as the details of the bell and spigot at the joint for all the pipe types were documented in Chapter 2

## Test Procedure

The pipe sections were positioned with its longitudinal axis parallel to the bearing plates and centered laterally in the test set-up. The LVDTs were installed in place (Fig. 7.1). The load was applied through a hydraulic jack on the center of a WF beam.
The connected pipe and fitting were loaded at rates of 0.5 in. per minute. The load-deflection measurements were recorded continuously and observations were made of the pipe connections.

## Test Program

Table 7.1 presents details of the test program carried out in this study on joint integrity."

### 7.3 Observations and Discussion

Pipe Stiffness
The pipe stiffness values for jointed specimens under parallel plate were calculated using the same procedure as that outlined in chapter 4, and are presented in Table 7.2 for $5 \%$ and $10 \%$ vertical deflections. The average values for specimens with no joints are also provided between parentheses for comparison and discussion.
Results show that for HDPE pipes the PS values of specimens with joints, although slightly smaller than, are very similar to corresponding specimens with no joints. For PVC pipes, however, the PS of specimens with joints is substantially greater than that of corresponding specimens with no joints, that is, the increase of the PS average value due to the presence of the joint is $37 \%$ and $46 \%$ for $5 \%$ and $10 \%$ vertical deflections, respectively.

## Gaps and Openings

Generally, all the specimens with joints behaved satisfactorily for deflections below $10 \%$, where no significant deformations were observed. The maximum openings in this range of deflection was 0.25 inch. As the vertical deflection increased, so did the joint openings and
the radial gaps between the two walls. However, the maximum openings and radial gaps observed were relatively small with a maximum opening of 0.75 in . and a maximum radial gap of 1.5 in . for $30 \%$ vertical deflection.

## HDPE ADS 48 pipes

Before the joint integrity test, the inside wall surface at the joint of the HDPE ADS 48" pipe presented some irregularities in the radial gap. The initial radial gap at the joint ranged from 0.22 in. to 0.68 in., as presented in Table 7.3. During the test and as the vertical load increased, the radial gap at the joint increased as presented in Table 7.3 for $15 \%$ and $30 \%$ vertical deflection. Vertically the maximum radial gap corresponding to $30 \%$ vertical deflection was 0.42 in . (crown/invert), whereas it was 1.25 in . in the horizontal direction (springline). Small gaps were also observed at the haunch and shoulder area. Initial joint openings in the longitudinal direction were also observed (approximately 0.2 in . max.), but did not open significantly wider as the load increased. Wall buckling was observed at approximately $30 \%$ vertical ring deflection (see Fig. 7.10b). No cracks were observed during the test.

Figs 7.7 to 7.10 show views of the behavior of the ADS $48^{\prime \prime}$ pipe joint during the course of the joint integrity test. The diameter recovery was almost complete 24 hours after the end of the test (ID after 24 hour recovery $=45.8$ in., compared to original $\mathrm{ID}=47.0 \mathrm{in}$.).

## HDPE ADS 36" pipes

Before joint integrity test, the inside wall surface at the joint of the HDPE ADS 36" pipe was smooth. During the test, radial gaps first appeared at the springline area of the specimen joint. With increasing vertical deflection, radial gaps widened and spread to the haunch and shoulder area. Table 7.4a presents the radial gaps recorded for $15 \%$ and $30 \%$ vertical deflection. The maximum radial gap observed was approximately 0.6 in . at the springline, for $30 \%$ vertical deflection. In addition, openings in the longitudinal direction were also observed, as presented in Table 7.4 b for $15 \%$ and $30 \%$ vertical deflection. For $30 \%$ vertical deflection, the maximum opening in the longitudinal direction was 0.4 in . at the crown/invert, whereas it was 0.75 in . at the springline. No wall buckling and cracks were observed during the test.

Figs 7.11 to 7.15 show views of the behavior of the ADS 36 " pipe joint during the course of the joint integrity test. The diameter recovery, was almost complete 24 hours after the end of the test (ID after 24 hour recovery $=35 \mathrm{in}$., compared to original ID,= 36.0 in .).

## HDPE HANCOR 36" pipes

Before the joint integrity test, the inside wall surface at the joint of the HDPE HANCOR pipe was smooth. During the load application, the two parts of the specimen did not deform to the same extent, thereby, creating radial gaps.

The recorded radial gaps at the joint are presented in Table 7.5 a for $15 \%$ and $30 \%$ vertical deflection. The maximum gap at $30 \%$ vertical deflection was 0.63 in . at the crown/invert, whereas it was 1.0 in . at the springline. Openings in the longitudinal direction were also observed. The maximum longitudinal opening observed at $30 \%$ vertical deflection was approximately 0.75 in. at the crown/invert. No longitudinal opening was observed at springlines.
At $30 \%$ vertical deflection, wall buckling was observed at both the crown and invert.
Figs 7.15 to 7.18 show views of the behavior of the Hancor $36^{\prime \prime}$ pipe joint during the course of the joint integrity test. The diameter recovery was almost complete 24 hours after the end of the test (ID after 24 hour recovery = 35.0 in ., compared to original ID $=35.85 \mathrm{in}$.).

## PVC 36" pipes

Before the joint integrity test, the inside wall surface at the joint of the PVC $36^{\prime \prime}$ pipe was smooth. Under the load application, the two specimens did not deform to the same extent. However, the radial gap between the pipes at the springline was larger than at other areas. The recorded radial gaps are presented in Table 7.6 for $15 \%$ and $30 \%$ vertical deflection.
The maximum gap for $30 \%$ vertical deflection was 1.5 in. at the springlines. No gap was observed at the crown/invert. No significant joint openings in the longitudinal direction were observed. Prior to the failure, reverse curvature was observed at both the invert and the crown of the pipe.
Figs 7.19 to 7.21 show views of the behavior of the PVC 36 " pipe joint during the course of the joint integrity test. The diameter recovery was almost complete 24 hours after the end of the test (ID after 24 hour recovery $=34.25$ in., compared to original ID $=35.5 \mathrm{in}$.).

### 7.4 Conclusions

The following conclusions can be drawn from the joint integrity results:
(a) Up to $10 \%$ vertical deflection, all the pipes behaved satisfactorily with no signs of cracks or excessive gaps.
(b) The radial gaps and longitudinal openings were small and reached 1.5 in . and 0.75 in ., respectively, for $30 \%$ vertical deflection.
(c) For HDPE ADS $48^{\prime \prime}$ and $36^{\prime \prime}$ diameter pipes, the presence of joints results in a slight reduction ( $10 \%$ maximum for $5 \%$ vertical deflection) of the PS values.
(d) For HDPE Hancor $36^{\prime \prime}$, the presence of joints resulted in an increase ( $23 \%$ for $5 \%$ vertical deflection) of the PS value.
(e) For PVC pipes, the presence of joints resulted in a significant increase ( $37 \%$ for $5 \%$ vertical deflection) of the PS value.

Table 7.1 - Joint Integrity Test Program

| Series | Load Rate <br> (in/min.) | Number of Tests |
| :--- | :---: | :---: |
| ADS 48 | 0.5 | 2 |
| ADS 36 | 0.5 | 2 |
| Hancor 36 | 0.5 | 2 |
| PVC 36 | 0.5 | 2 |
| Steel 36 | 0.5 | 2 |
| Aluminum 36 | 0.5 | 2 |

Table 7.2 - Experimental Pipe Stiffness (Load Rate $=0.5 \mathrm{in} . / \mathrm{min}$.)

| Series | Specimen | 5\% Vertical Deflection |  |  | 10\% Vertical Deflection |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Load (Lbs/in.) | Defl. <br> (in.) | $\begin{gathered} \text { PS } \\ \text { (psi) } \end{gathered}$ | Load (Lbs/in.) | Defl. <br> (in.) | $\begin{gathered} \mathrm{PS} \\ (\mathrm{psi}) \end{gathered}$ |
| (a) ADS 48 | 0.5-1 | 60.47 | 2.406 | 25.13 | 96.59 | 4.806 | 20.10 |
|  | 0.5-2 | 57.32 | 2.404 | 23.84 | 89.52 | 4.800 | 18.65 |
|  | 0.5-mean |  |  | 24.49 |  |  | 19.37 |
|  |  |  |  | (26.91) |  |  | (20.44) |
| (b) ADS 36 | 0.5-1 | 54.45 | 1.806 | 30.15 | 90.05 | 3.600 | 25.01 |
|  | 0.5-2 | 68.05 | 1.813 | 37.54 | 98.42 | 3.602 | 27.33 |
|  | 0.5-mean |  |  | 33.85 |  |  | 26.17 |
|  |  |  |  | (37.09) |  |  | (28.62) |
| (c) HANCOR 36 | 0.5-1 | 59.68 | 1.801 | 33.14 | 86.90 | 3.600 | 24.14 |
|  | 0.5-2 | 53.40 | 1.801 | 29.65 | 86.90 | 3.605 | 24.11 |
|  | 0.5-mean |  |  | 31.40 |  |  | 24.12 |
|  |  |  |  | (25.43) |  |  | (19.89) |
| (d) PVC 36 | 0.5-1 | 139.26 | 1.806 | 77.11 | 258.63 | 3.600 | 71.84 |
|  | 0.5-2 | 129.84 | 1.804 | 72.09 | 254.44 | 3.605 | 70.58 |
|  | 0.5-mean |  |  | $\begin{gathered} 74.60 \\ (54.62) \end{gathered}$ |  |  | $71.20$ |
| Note: ${ }^{(a)}$ Value |  |  | alues of | (54.62) | in,t from Ta | 43 b | (48.64) |

Table 7.3 - Radial Gap at Joint versus Vertical Deflection for ADS 48

| Location | Initial Gap <br> (in.) | Gap at 15\% Vertical <br> Deflection (in.) | Gap at 30\% Vertical <br> Deflection (in.) |
| :--- | :---: | :---: | :---: |
| Crown | 0.4225 | $0.5470(0.1245)^{*}$ | $0.84(0.4175)$ |
| Invert | 0.6810 | $0.8740(0.1930)$ | $1.10(0.4190)$ |
| Springline West | 0.2190 | $0.6400(0.4210)$ | $1.33(1.1110)$ |
| Springline East | 0.3475 | $1.0325(0.6850)$ | $1.60(1.2525)$ |

Note: * Values between parentheses are net values due to ring deflection.

Table 7.4a - Radial Gap at Joint versus Vertical Deflection for ADS 36

| Location | Initial Gap <br> (in.) | Gap at 15\% Vertical <br> Deflection (in.) | Gap at 30\% Vertical <br> Deflection (in.) |
| :--- | :---: | :---: | :---: |
| Crown | 0.0 | 0.0 | 0.0 |
| Invert | 0.0 | 0.0 | 0.0 |
| Springline West | 0.0 | 0.25 | 0.60 |
| Springline East | 0.0 | 0.25 | 0.60 |

Table 7.4b - Joint Opening versus Vertical Deflection for ADS 36

| Location | Initial Opening | Opening at 15\% <br> Vertical Deflection <br> (in.) | Opening at 30\% <br> Vertical Deflection <br> (in.) |
| :--- | :---: | :---: | :---: |
| Crown | 0.0 | 0.40 | 0.40 |
| Invert | 0.0 | 0.40 | 0.40 |
| Springline West | 0.0 | 0.25 | 0.75 |
| Springline East | 0.0 | 0.50 | 0.75 |

Table 7.5a - Radial Gaps at Joint versus Vertical Deflection for Hancor 36

| Location | Initial Gap | Gap at 15\% Vertical <br> Deflection <br> (in.) | Gap at 30\% Vertical <br> Deflection <br> (in.) |
| :--- | :---: | :---: | :---: |
| (in.) | 0.0 | 0.63 |  |
| Invert | 0.0 | 0.0 | 0.63 |
| Springline West | 0.0 | 0.50 | 1.00 |
| Springline East | 0.0 | 0.50 | 0.875 |

Table 7.5b - Joint Opening versus Vertical Deflection for Hancor 36

| Location | Initial Opening | Opening at 15\% <br> Vertical Deflection <br> (in.) | Opening at 30\% <br> Vertical Deflection <br> (in.) |
| :--- | :---: | :---: | :---: |
| Crown | 0.0 | 0.50 | 0.75 |
| Invert | 0.0 | 0.50 | 0.75 |
| Springline West | 0.0 | 0.00 | 0.0 |
| Springline East | 0.0 | 0.00 | 0.0 |

Table 7.6 - Radial Gap at Joint versus Vertical Deflection for PVC 36

| Location | Initial Gap | Gap at 15\% Vertical <br> Deflection (in.) | Gap at 30\% Vertical <br> Deflection (in.) |
| :--- | :---: | :---: | :---: |
| (in.) | 0.00 | 0.0 |  |
| Crown | 0.0 | 0.00 | 0.0 |
| Springline West | 0.0 | 0.60 | 1.50 |
| Springline East | 0.0 | 0.60 | 1.50 |



Fig. 7.1 - LVDT Experimental Set-up


Fig. 7.2 - Test Specimen


Fig. 7.3-Load versus Vertical (Shortening) and Horizontal (Elongation) Deflections for ADS 48


Fig. 7.4 - Load versus Vertical (Shortening) and Horizontal (Elongation) Deflections for ADS 36


Fig. 7.5 - Load versus Vertical (Shortening) and Horizontal (Elongation) Deflections for Hancor 36


Fig. 7.6 - Load versus Vertical (Shortening) and Horizontal (Elongation) Deflections for PVC 36


Fig. 7.7 - Joint Integrity Test Setup for ADS 48" Pipe


Fig. 7.8 - Exterior View of Joint of ADS 48 Specimen Prior to Testing

(a) Deformed Shape of ADS48 at 20\% Defl.

(b) Interior of ADS48 at $15 \%$ to $20 \%$ Defl.

Fig. 7.9 - Behavior of Joint of ADS 48 Specimen at 20\% vertical Deflection

(a) Deformed Shape of ADS48 at 30\% Defl.

(b) Interior of ADS48 at 30\% Defl.

Fig. 7.10 - Behavior of Joint of ADS 48 Specimen at 30\% vertical Deflection


Fig. 7.11 - Joint Integrity Test Setup for HDPE ADS 36" Pipe


Fig. 7.12a-Exterior View of ADS36 Prior to Test


Fig. 7.12b -Interior View of ADS36 Prior 1 Test

(a) Gap at Spingline for ADS36 at 10\% Defl.

(b) Gap at Invert for ADS36 at 20\% Defl.

Fig. 7.13 - Behavior of Joint of ADS 36 Specimen at $\mathbf{1 0 \%}$ and $\mathbf{2 0 \%}$ Vertical Deflection


Fig. 7.14 - Behavior of Joint of ADS 48 Specimen at 30\% Vertical Deflection


Fig. 7.15 - Joint Integrity Test Setup for HDPE
Hancor 36" Pipe


Fig. 7.16a -Exterior View of Hancor36 Prior to Test


Fig. 7.16b -Interior View of Hancor36 Prior to Test


Fig. 7.17 - Behavior of Joint of Hancor 36 Specimen at 15\% and 20\% Vertical Deflection


Fig. 7.18 - Behavior of Joint of Hancor 36 Specimen at $30 \%$ and $33 \%$ vertical Deflection


Fig. 7.19 - Joint Integrity Test Setup for PVC 36" pipe


Fig. 7.20 - Behavior of Joint of PVC 36 Specimen at $30 \%$ vertical Deflection


Fig. 7.21 - Failure of PVC $36 \%$ Pipe at $33 \%$ Vertical Deflection

## Chapter 8:Tensile Tests on Dumbbell-Shaped Speciens

### 8.1 Scope and Objectives

The objective of this test was to determine the tensile properties of an HDPE coupon cut from the ADS D-wall-type pipe in the form of a dumbbell-shaped (dog bone shaped) specimen. The specimens were tested under predetermined cross-head speed and ambient conditions. The tensile properties include the tensile strength, percent elongation, the modulus of elasticity and Poisson's ratio. Most of the test procedure and method of calculating tensile properties follow closely the approach described in ASTM D-638, Standard Test Method for Tensile Properties of Plastics.

### 8.2 Experimental Program

Two types of specimens were used in the test The first type (Type A) is of double wall type since the pipe configuration is of a D-type pipe as described in section 4.1.3 of AASTHO M 294-98, "Standard Specification for Corrugated Polyethylene Pipe 300- to $1200-\mathrm{mm}$ Diameter", and the second type (Type B) has only one wall thickness, cut into half from the first type. Figs. 8.1 and 8.2 show the configurations of the test specimens Type A and Type B, respectively.

Apparatus
The testing device shown in Figure 8.3 was specially fabricated. The specimen was held in place by connecting steel rods at both ends. The rods pass through the infill mortar between the inner and outer walls. A hydraulic jack having a constant rate-of-head movement was used to a pply the tensile force. The two ends of the specimen were free to move into alignment upon load application so that the longitudinal axis of the specimen would coincide with the direction of the applied load. The applied load was measured using the load cell/ gage pressure. The change in length of the specimen, and the axial,. and transverse strains were recorded using LVDTs and strain gages, respectively.

## Test Specimens

The test specimen was first cut from the flexible pipe in the form of longitudinal strips 11.2 in . x 28 in. The coupon was then machined to obtain the shape shown in Fig. 8.4, ensuring one weld at the center of the specimen. The coupon was than machined to obtain the dumbbell-shaped specimen (Dog bone shape), and then instrumented as shown in Fig. 8.5.

## Test Procedure

All strain gages were installed on the specimen and the specimen aligned so as to ensure the longitudinal axis of the specimen to be coincident with the direction of the applied load. The tensile force was applied at a constant rate-of-head speed of 0.5 in . per minute until the specimen failed. The data acquisition system was used to continuously record both transverse strain and axial strain simultaneously. The applied tensile load and the corresponding elongation of the specimen were also continuously recorded.

## Test Program

Table 8.1 presents the details of the testing program.

### 8.3 Calculations

## i) Ultimate tensile strength

Ultimate tensile strength, $\sigma_{\mathrm{u}}$, is calculated by dividing the maximum load at rupture, $\mathrm{F}_{\mathrm{b}}$, in newtons (or pounds-force) by the original cross-sectional area, A , of the specimen m square metres (or square inches).

## ii) Modulus of elasticity

First, a graph of stress versus strain of the specimen is plotted. The initial linear portion of the stressstrain curve is extended, and the modulus of elasticity, E , is given by the slope of this straight line, which is calculated by dividing the difference in stress corresponding to any segment on the straight line by the corresponding difference in strain.

## iii) Poisson's ratio

The axial and transverse strains obtained from the test are plotted against the applied load. Straight lines are drawn through each set of points for both the axial, $\varepsilon_{\mathrm{a}}$, and the transverse, $\varepsilon_{\mathrm{t}}$,
strains. One of any section in the linear portion of the graph is selected and the change in strain is determined. Then, Poisson's ratio, $\mu$, is calculated using Eq. (8.1) shown below:

$$
\begin{equation*}
\mu=-(\text { change in transverse strain }) /-(\text { change in axial strain }) \tag{8.1}
\end{equation*}
$$

### 8.4 Results and Observations

i) Type $A$ specimen

Table 8.2 summarizes the experimental results for tension test Type A for the ADS $48^{\prime \prime}$ pipe. It presents the maximum forces and corresponding stresses, and the recorded strains in the longitudinal and transverse directions. The table also provides the apparent modulus of elasticity and Poisson's ratio for each test which are determined by using the example curve fittings shown in Fig. 8.7. Fig. 8.6 shows the load versus the longitudinal and transverse strains curves for the four Type A tests. Typical views of the specimens at failure are presented in Fig. 8.8.

## ii) Type B specimen

Table 8.3 summarizes the experimental results for tension test Type B for the ADS 48 " pipe. It presents the maximum forces and corresponding stresses, and the recorded strains in the longitudinal and transverse directions. The table also provides the apparent modulus of elasticity and Poisson's ratio for each test, which are determined by using the example fittings shown in Fig. 8.10. Fig. 8.9 shows the load versus the longitudinal and transverse strains curves for the four Type B tests. Typical views of the specimens at failure are presented in Fig. 8.11 .

## iii) Observations

- The single wall specimen (Type B) exhibited apparent tensile properties superior to those of the double wall specimen (Type A).
- The average maximum tensile strength achieved by Type B was 2935 psi compared to 2049 psi for Type A specimen. The average apparent modulus of elasticity was 413 ksi for Type B versus 282.5 ksi for Type A. The average maximum longitudinal strain was similar for both types and attained approximately $1.1 \%$. The maximum transverse strain was substantially lower for Type A (double wall: $1,248 \mu \varepsilon$ ) than for Type B (single wall: 5,596 $\mu \varepsilon$ ).


### 8.5 Conclusions

Results indicate that Type A test (Double wall dumbbell shape) underestimates the tensile strength of the D-wall-type pipes such as the ADS 48". Results also indicate that the seam behavior under tensile stresses is satisfactory in view of the maximum strength achieved.

Table 8.1 - Type A and B Test Program

| Test Type | Description | Number of Tests |
| :---: | :---: | :---: |
| Type A | Double Wall | 4 |
| Type B | Single Wall | 4 |

Table 8.2 - Summary of Results for Tests Type $A$

| Test\# | Maximum Load <br> (Lbs) | Maximum <br> Stress, $\sigma_{u}^{(a)}$ <br> (psi) | Modulus o Elasticity ${ }^{(\text {(a) }}$ $(\mathrm{psi})$ | Max. <br> Long. <br> Strain <br> ( $\mu \varepsilon$ ) | Max. <br> Transv. Strain ( $\mu \mathrm{\varepsilon}$ ) | Poisson's <br> Ratio, $\mu$ | Rupture <br> Mode |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 1132 | 2258 | 301860 | $9591{ }^{(8)}$ | 1528 | 0.146 | Inside wall |
| 2 | 1138 | 2270 | 235503 | $13247^{(1)}$ | 1234 | 0.093 | Inside wall |
| 3 | 1028 | 2051 | 231958 | $16004^{(L)}$ | 1413 | 0.109 | Inside wall |
| 4 | 810 | 1616 | 360622 | $6840{ }^{(12)}$ | 816 | 0.143 | Inside at weld |
| Average | 1027 | 2049 | 282486 | 11420 | 1248 | 0.123 | -- |
| Notes: ${ }^{(a)}$ | Based on a thickness of ( $0.071 \mathrm{in} \times 2$ walls $)$ and an average measured width of 3.53 inches, that is $\mathrm{A}=0.5013 \mathrm{in}^{2}$ <br> $\mathrm{B}, \mathrm{T}, \mathrm{L} 1=$ Bottom, Top, and L1 strain gages |  |  |  |  |  |  |

Table 8.3 - Summary of Results for Tests Type B

| Test\# | Maximum Load (Lbs) | Measured <br> Thickness <br> (in.) $\left(\operatorname{area}\left(-\mathrm{in}^{2}\right)\right)$ | Maximum <br> Stress, $\sigma_{u}{ }^{\left({ }^{(a)}\right.}$ <br> (psi) | Modulus of Elasticity ${ }^{(\text {a })}$ (psi) | Max. Long. Strain $(\mu \varepsilon)$ | Max. <br> Transv. Strain ( $\mu \varepsilon$ ) | Poisson's Ratio, $\mu$ | Rupture <br> Mode |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | 611 | $\begin{aligned} & 3.54 \\ & (0.2513) \end{aligned}$ | 2431 | 745386 | 3670 | 4286 | 0.101 | Seam |
| 2 | 492 | $\begin{aligned} & 3.58 \\ & (0.2542) \end{aligned}$ | 1935 | 238090 | 11337 | 3256 | 0.325 | Wall |
| 3 | 904 | $\begin{aligned} & 3.48 \\ & (0.2471) \end{aligned}$ | 3658 | 313980 | 14636 | 7565 | 0.357 | Wall |
| 4 | 929 | $\begin{aligned} & 3.52 \\ & (0.2499) \end{aligned}$ | 3717 | 353062 | 14293 | 7276 | 0.378 | Wall |
| Average | 734 | $\begin{aligned} & 3.53 \\ & (0.2506) \end{aligned}$ | 2935 | 412629 | 10984 | 5596 | 0.290 | --- |
| Note: (a) Based on a thickness of 0.071 inches and measured widths as indicated. |  |  |  |  |  |  |  |  |

## (a) Front View


(b) Side View


Fig, 8.1 - Double wall type specimen (Type A)


Fig. 8.2 - Single Wall Type Specimen (Type B)


Fig. 8.3 - Typical Test Setup

$28^{\prime \prime}$


Fig. 8.4 - Specimen Cut in the Form of Longitudinal Strip from the Pipe


Fig. 8.5 - Specimen Cut Into Dumbbell-Shape (Dogbone Shape)
(a) Test \# 1

(b) Test \# 2

(c) Test \# 3

(d) Test \# 4


Figure 8.6 - Load vs. Axial and Transverse Strain for Type A Specimen
(a) Modulus of elasticity

(b) Poisson's ratio


Fig. 8.7-Typical Curve Fittings for Modulus and Poisson's Ratio for Type A
(a) General View

(b) Close-up View


Fig. 8.8 - Typical Views of Specimens Type A at Failure

## (a) Test \# 1


(b) Test \# 2

(c) Test \# 3

(d) Test \# 4


Fig. 8.9 - Load vs. Axial and Transverse Strain for Type B Specimen
(a) Modulus of elasticity

Dogbone Type B /ADS48 / Test\#1

(b) Poisson's ratio

> Dogbone Type B /ADS48 / Test\#1


Fig. 8.10 - Typical Curve Fittings for Modulus and Poisson's Ratio for Type B
(a) General View

(b) Lose-up View


Fig. 8.11 - Typical Views of Specimens Type $B$ at Failure

## Chapter 9: Tensile Tests on Full-Ring Specimens

### 9.1 Scope and Objectives

In this test, an apparent hoop tensile strength is determined by utilizing a split disk test fixture. The test specimen, a full-diameter, full-thickness ring cut from the pipe, is tested under a predetermined cross-head speed and ambient conditions. The test procedure and method of calculation follow closely the ASTM D2290, Standard Test Method for Apparent Hoop Tensile Strength of Plastic or Reinforced Plastic Pipe by Split Disk Method.

The apparent tensile strength rather than a true tensile strength is obtained due to the bending moment induced by the change in contour of the ring between the two disk sections as they separate. The tensile strength obtained will provide reasonably accurate information for plastic pipe when employed under conditions approximating those under which the tests are performed. The vertical diametric strain and the modulus of elasticity will also be computed from the results of this test.

### 9.2 Experimental Program

## Apparatus

Two different configurations of the split disk test fixtures, based on two pipe-diameter sizes (48" and 36 " pipe diameters), were specially fabricated. The test fixtures were both made by using $3 / 8$ "smooth rigid steel semi-circular pipes. Steel plates of $3 / 8^{\prime \prime}$ thickness cut in a segmental shape were welded to the-machined steel' pipes to reinforce the fixtures. Figs. 9.1 and 9.2 show the test fixtures for the 36 " and $48^{\prime \prime}$ diameter pipes respectively. These fixtures were then attached to the lifting arms of forklift using steel rods welded to the steel plates. Fig. 9.3 illustrates an overall setup for the 36 " diameter pipe tests. The major difference between the two fixtures is that the test fixture for the 48-in. diameter pipe had two hydraulic jacks in order to have a uniform load application on the test specimens. On the other hand, the test fixture for the 36 -in diameter pipes required only one jack, but with a larger piston diameter for the load application.

## Test Specimen

The length of the HDPE $36^{\prime \prime}$ diameter specimens was chosen as 15.5 in ., whereas the length of the PVC 36" pipes was 8.75 in . to ensure that at least two or three corrugations or spiral ribs were included on the specimens. Figure 9.4 shows the length of the specimen as well as dimensions of the reduced section for the HDPE $36^{\prime \prime}$, diameter pipe. The length of the $48^{\prime \prime}$ diameter specimen was chosen as 40 inches. Fig. 9.5 shows both the length and the dimension of the reduced section for the HDPE $48^{\prime \prime}$ diameter specimens. The reduced cross sections were located at $180^{\circ}$ from each other and machined such that the specimens were free of sharp corners to avoid stress concentrations.

## Test Procedure

Once the test fixture, either for the 36 " or 48 "diameter pipes, was secured to the forklift, the inside surface of the test specimen was lubricated and then mounted on the test fixture. The test specimen was aligned to the center of the split disk specimen holder. The testing was performed at a speed of 0.5 inch per minute. The tensile load was continuously recorded by using the data acquisition system until the specimens completely ruptured. A deflectometer was also used to record the elongation along the direction of the load application (see Fig. 9.3).

## Test Program

Details of the split disk test program are presented in Table 9.1. The reduced lengths as well as the area of the reduced sections for the two walls are also provided in the table.

### 9.3 Calculations

i) Apparent tensile strength

The apparent tensile strength, $\sigma_{a}(\mathrm{psi})$, of the specimen is calculated by dividing the maximum tensile load, $\mathrm{F}_{\mathrm{u}}(\mathrm{lbs})$ by the cross-sectional areas of the reduced sections $\mathrm{A}_{\mathrm{m}}$ (in. ${ }^{2}$ ), as shown below:

$$
\begin{equation*}
\sigma_{\mathrm{a}}=\mathrm{F}_{\mathrm{u}} / 2 \mathrm{~A}_{\mathrm{m}} \tag{9.1}
\end{equation*}
$$

Where,

| $\mathrm{A}_{\mathrm{m}}$ | $=$ minimum cross-sectional area, |
| ---: | :--- |
|  | $=\mathrm{dxb}$, in. ${ }^{2}$ |
| d | $=$ thickness at minimum area; in. (= wall area in. $\left.{ }^{2} / \mathrm{in}.\right)$, and |
| b | $=$ width at minimum area, in. |

## ii) Vertical diametric strain

The vertical diametric strain is calculated by dividing the elongation in the direction of the load application by the original nominal pipe diameter.

### 9.4 Test Results and Observations

Figs 9.6, 9.7, and 9.8 illustrate the test setup and the deformations of the HDPE specimens ADS $48^{\prime \prime}$, ADS 36 ", and Hancor 36 ", respectively. Figs 9.9, 9.10, and 9.11 show the deformations of the PVC specimens and the cracking of the steel and aluminum specimens, respectively. Typical load versus vertical diametric strain for ADS 48", ADS 36", Hancor 36", PVC, steel, and aluminum specimens are presented respectively in Figs. 9.12, 9.13, 9.14, 9.15, 9.16, and 9.17.

The following observations can be made:
(a) All the HDPE pipes achieved a similar maximum apparent tensile strength of approximately 2700 psi (see Table 9.2). However, the maximum radial strain was higher for ADS 48" (19.08\%), compared to ADS 36" (10.45\%) and to Hancor 36" (11.11\%).
(b) All the HDPE pipes also achieved a similar apparent modulus of elasticity (see Table 9.3a) of approximately 70 ksi .
(c) The PVC pipe achieved a maximum apparent tensile stress of 5,258 psi, a maximum tensile radial strain of $2.92 \%$, and an average apparent modulus of elasticity of 271 ksi .
(d) The steel and aluminum pipes achieved respectively a maximum apparent tensile stress of $46,249 \mathrm{psi}$ and $32,063 \mathrm{psi}$, a maximum tensile radial strain of $1.96 \%$ and $1.59 \%$, and an average apparent modulus of elasticity of $4,886 \mathrm{ksi}$ and $2,977 \mathrm{ksi}$.

### 9.5 Conclusions

Results show that apparent tensile strength under split disk tests are lower than those under tensile tests on small dog bone specimens with no welds (see chapter 10). However, they are higher than those achieved on dumbbell shape specimens with welds for ADS 48 (see chapter 8).

Table 9.1 - Experimental Program for Split Disk Tests

| Series | Load Rate $\text { (in. } / \mathrm{min} \text {.) }$ | Number of Tests | Reduced Length of Specimen (one wall) (in.) | Unit Area of Specimen $\left(\mathrm{in}^{2} / \mathrm{in} .\right)$ | Total Area of Reduced Section (in ${ }^{2}$ ) |
| :---: | :---: | :---: | :---: | :---: | :---: |
| ADS 48 | 0.5 | 3 | 34.00 | $0.581{ }^{\text {(b) }}$ | 39.508 |
| ADS 36 | 0.5 | 3 | 14.25 | $0.401^{\text {(b) }}$ | 11.429 |
| HANCOR 36 | 0.5 | 3 | 13.38 | $0.375^{\text {(c) }}$ | 10.035 |
| PVC 36 | 0.5 | 3 | $7.75{ }^{(\mathrm{a})}$ | $0.411^{\text {(d) }}$ | 6.371 |
| Steel 36 | 0.5 | 3 | 23.00 | $0.0593{ }^{(d)}$ | 2.730 |
| Aluminiun 36 | 0.5 | 3 | 23.00 | $0.0474^{(\mathrm{d})}$ | 2.181 |

Notes: ${ }^{(a)}$ Measured ( $\neq$ from written procedure where it was 7.25 in .)
${ }^{(b)}$ From Manufacturer Product Information Sheet
(c) From Hancor, inc., Drainage Hand book
(d) From Contech (Fax of June 28, 2001). Original values given in $\mathrm{in}^{2} / \mathrm{ft}$.
(c) Two wall section

Table 9.2 - Summary of Results for Split Disk Tests

| Series | Test\# | Maximum <br> Load | Total area of <br> Reduced <br> Section <br> (in. ${ }^{2}$ | Maximum <br> Apparent <br> Stress <br> (psi) | Maximum <br> Radial Strain <br> (a) <br> (\%) | Maximum <br> Elongation |
| :--- | :--- | :--- | :---: | :---: | :---: | :---: |
| ADS 48 | 1 | 102.62 | 39.508 |  | 16.23 | (in.) |
|  | 2 | 98.79 | 39.508 |  | 21.52 | 10.79 |
|  | 3 | 106.47 | 39.508 |  | 19.50 | 9.36 |
|  | Average | 102.63 | 39.508 | 2598 | 19.08 | 9.16 |
| ADS 36 | 1 | 32.34 | 11.429 |  | 8.86 | 3.19 |
|  | 2 | 32.57 | 11.429 |  | 12.17 | 4.38 |
|  | 3 | 30.78 | 11.429 |  | 10.33 | 3.72 |
|  | Average | 31.90 | 11.429 | 2791 | 10.45 | 3.76 |
| HANCOR 36 | 1 | 27.22 | 10.035 |  | 10.11 | 3.64 |
|  | 2 | 28.30 | 10.035 |  | 14.31 | 5.15 |
|  | 3 | 26.44 | 10.035 |  | 8.92 | 3.21 |
|  | Average | 27.32 | 10.035 | 2722 | 11.11 | 4.00 |
| PVC 36 | 1 | 32.49 | 6.371 |  | 2.59 | 0.933 |
|  | 2 | 33.96 | 6.371 |  | 3.53 | 1.270 |
|  | 3 | 34.04 | 6.371 |  | 2.64 | 0.950 |
|  | Average | 33.50 | 6.371 | 5258 | 2.92 | 1.051 |
| Steel 36 | 1 | 127.62 | 2.730 |  | 1.88 | 0.677 |
|  | 2 | 121.67 | 2.730 |  | 1.83 | 0.657 |
|  | 3 | 129.50 | 2.730 |  | 2.17 | 0.780 |
|  | Average | 126.26 | 2.730 | 46249 | 1.96 | 0.705 |
| Aluminum 36 | 1 | 66.79 | 2.181 |  | 1.60 | 0.576 |
|  | 2 | 64.28 | 2.181 |  | 1.55 | 0.559 |
|  | 3 | 78.71 | 2.181 |  | 1.62 | 0.583 |
|  | Average | 69.93 | 2.181 | 32063 | 1.59 | 0.573 |

Note: (a) Maximum elongation divided by nominal pipe diameter multiplied by $10^{6}$.

Table 9.3a - Calculations for Apparent Modulus of Elasticity for HDPE Pipes

| Test\# |  | Loads (kips) | Total Area $\left(\mathrm{in}^{2}\right)$ | Stress (psi) | Elongation <br> (in.) | Diameter <br> (in.) | Strain | Apparent Modulus of Elasticity (psi) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| a) ADS 48 |  |  |  |  |  |  |  |  |
|  | R1 | 10.2246 |  |  | 0.598 |  |  |  |
|  | R2 | 25.5408 |  |  | 0.849 |  |  |  |
|  | $\Delta \mathrm{R}$ | 15.3162 | 39.508 | 387.67 | 0.251 | 48 | 0.0052 | 74137 |
| \#2 | R1 | 10.7173 |  |  | 0.317 |  |  |  |
|  | R2 | 26.2390 |  |  | 0.578 |  |  |  |
|  | $\Delta \mathrm{R}$ | 15.5217 | 39.508 | 392.87 | 0.210 | 48 | 0.0054 | 72755 |
| \#3 | R1 | 10.3067 |  |  | 0.512 |  |  | $\cdots$ |
|  | R2 | 25.0071 |  |  | 0.733 |  |  |  |
|  | $\Delta \mathrm{R}$ | 14.7004 | 39.508 | 372.10 | 0.221 | 48 | 0.0046 | 80817 |
| b) $\operatorname{ADS} 36$ |  |  |  |  |  |  | Average | 75903 |
|  |  |  |  |  |  |  |  |  |
| \#1 | R1 | 5.1954 |  |  | 0.522 |  |  |  |
|  | R2 | 15.1209 |  |  | 0.957 |  |  |  |
|  | $\Delta \mathrm{R}$ | 9.9255 | 11.429 | 868.45 | 0.435 | 36 | 0.0121 | 71872 |
| \#2 | R1 | 5.1178 |  |  | 0.677 |  |  |  |
|  | R2 | 15.6637 |  |  | 1.144 |  |  |  |
|  | $\Delta \mathrm{R}$ | 10.5459 | 11.429 | 922.73 | 0.467 | 36 | 0.0130 | 71131 |
| \#3 | R1 | 5.4280 |  |  | 0.743 |  |  |  |
|  | R2 | 15.1209 |  |  | 1.159 |  |  |  |
|  | $\Delta \mathrm{R}$ | 9.6929 | 11.429 | 848.10 | 0.416 | 36 | 0.0116 | 73393 |
| c) HANCOR \#1 |  |  |  |  |  |  | Average | 72132 |
|  | R1 | 5.1178 |  |  | 0.760 |  |  |  |
|  | R2 | 15.2760 |  |  |  |  |  |  |
|  | $\Delta \mathrm{R}$ | 10.1582 | 10.035 | 1012.28 | 0.52 | 36 | 0.0144 | 70081 |
| \#2 | R1 | 5.5055 |  |  | 0.561 |  |  | - |
|  | R2 | 15.276 |  |  | 1.068 |  |  |  |
|  | $\Delta \mathrm{R}$ | 9.7705 | 10.035 | 973.64 | 0.507 | 36 | 0.0141 | 69134 |
| \#3 | R1 | $5.5055$ |  |  | $0.554$ |  |  |  |
|  | R2 | $15.1209$ |  |  | 1.093 |  |  |  |
|  | $\Delta \mathrm{R}$ | 9.6154 | 10.035 | 958.19 | 0.539 | 36 | 0.015 | 63998 |
|  |  |  |  |  |  |  | Average | 67738 |

Table 9.3b - Calculations for PVC and Metal Pipes

| Test\# |  | Loads (kips) | Total Area $\left(\mathrm{in}^{2}\right)$ | Stress <br> (psi) | Elongation <br> (in.) | Diameter <br> (in.) | Strain | Apparent Modulus of Elasticity (psi) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| d) PVC 36 |  |  |  |  |  |  |  |  |
| \#1 | R1 | 5.3505 |  |  | 0.2977 |  |  |  |
|  | R2 | 15.2760 |  |  | 0.5044 |  |  |  |
|  | $\Delta \mathrm{R}$ | 9.9255 | 6.371 | 1557.92 | 0.2067 | 36 | 0.0057 | 271335 |
| \#2 ${ }^{\text {²}}$ | R1. | 5.5055 |  |  | 0.3469 |  |  |  |
|  | R2 | 15.4310 |  |  | 0.5610 |  |  |  |
|  | $\Delta \mathrm{R}$ | 9.9255 | 6.371 | 1557.92 | 0.2141 | 36 | 0.0059 | 261958 |
| \#3 | R1 | 5.5055 |  |  | 0.2190 |  |  |  |
|  | R2 | 15.1209 |  |  | 0.4134 |  |  |  |
|  | $\Delta \mathrm{R}$ | 9.6154 | 6.371 | 1509.25 | 0.1944 | 36 | 0.0054 | 279491 |
|  |  |  |  |  |  |  | Average | 270928 |
| e) Steel 36 |  |  |  |  |  |  |  |  |
| \#1 | R1 | 20.6955 |  |  | 0.06890 |  |  |  |
|  | R2 | 52.366 |  |  | 0.15994 |  |  |  |
|  | $\Delta \mathrm{R}$ | 31.6705 | 2.730 | 11601 | 0.0904 | 36 | 0.0025 | 4640366 |
| \#2 | R1 | 20.0684 |  |  | 0.09596 |  |  |  |
|  | R2 | 50.4846 |  |  | 0.16486 |  |  |  |
|  | $\Delta \mathrm{R}$ | 30.4162 | 2.730 | 11141 | 0.0689 | 36 | 0.0019 | 5863929 |
| \#3 | R1 | 21.0091 |  |  | 0.30266 |  |  |  |
|  | R2 | 50.4846 |  |  | 0.39616 |  |  |  |
|  | $\Delta \mathrm{R}$ | 29.4755 | 2.730 | 10797 | 0.0935 | 36 | 0.0026 | 4152649 |
|  |  |  |  |  |  |  | Average | 4885648 |
| f) Aluminum 36 |  |  |  |  |  |  |  |  |
| \#1 | R1 | 15.3649 |  |  | 0.27805 |  |  |  |
|  | R2 | 29.1619 |  |  | 0.34203 |  |  |  |
|  | $\Delta \mathrm{R}$ | 13.80 | 2.181 | 6326 | 0.0640 | 36 | 0.0018 | 3514443 |
| \#2 | R1 | 15.3649 |  |  | 0.24114 |  |  |  |
|  | R2 | 30.1026 |  |  | 0.32480 |  |  |  |
|  | $\Delta \mathrm{R}$ | 14.74 | 2.181 | 6757 | 0.0837 | 36 | 0.0023 | 2937962 |
| \#3 | R1 | 15.0513 |  |  | 0.1698 |  |  |  |
|  | R2 | 30.7297 |  |  | 0.2756 |  |  |  |
|  | $\Delta \mathrm{R}$ | 15.6784 | 2.181 | 7189 | 0.1058 | 36 | 0.0029 | 2478838 |
| $\cdots$ |  |  |  |  |  |  | Average | 2977081 |



Fig. 9.1-Test fixture for the 36" diameter pipe


Fig. 9.2-Test fixture for the 48" diameter pipe


Fig. 9.3-Overall Configuration of the Test Fixture for Testing the 36" Diameter Pipe


Figure 9.4-HDPE 36" diameter specimen configurations


Figure 9.5-HDPE 48" diameter specimen configurations


Fig. 9.6-Deformation of ADS 48 specimen specimen under tensile load


Fig. 9.7-Deformation of ADS 36 specimen under tensile load


Fig. 9.8 - Deformation of HANCOR 36 specimen under tensile load


Fig. 9.9 - Deformation of PVC specimen under tensile load


Fig. 9.10-External cracking of steel specimen


Fig. 9.11-External cracking of aluminum specimen


Fig. 9.12-Load vs. Vertical Diametral Change for ADS 48 specimen


Fig. 9.13 - Load vs. Vertical Diametral Change for ADS 36 Specimen


Fig. 9.14 -Load vs. Vertical Diametral Change for HANCOR 36 Specimen


Fig. 9.15 - Load vs. Vertical Diametral Change for PVC 36 Specimen


Fig. 9.16 - Load vs. Vertical Diametral Change for Steel 36 Specimen


Fig. 9.17-Load vs. Vertical Diametral Change for Aluminum 36 Specimen

## Chapter 10:Tensile Tests on 10 Inch - Dog Bone Specimens

### 10.1 Objectives

The objective of this test was to determine he tensile properties of an HDPE coupon cut from the pipe specimen in the form of a dog bone shaped specimen. The specimens were tested under predetermined cross-head speed and ambient conditions. The tensile properties include the tensile strength, the percent elongation, the modulus of elasticity and Poisson's ratio. Most of the test procedure and method of calculating tensile properties closely follow the approach described in ASTM D-638, Standard Test Method for Tensile Properties of plastics.

### 10.2 Experimental program Apparatus

A 110-kip ( 500 kN ), servo-hydraulic, tensile testing machine (Type MTS 810, for example see Fig. 10.2) was used for the dog bone tension tests. The machine is equipped with a Testar digital interface and is controlled by a computer program. In addition, the MTS machine was equipped with special grips to hydraulically control the pressure. The tests were performed at a displacement rate ranging from $0.05 \mathrm{in} . / \mathrm{min}$. to $150 \mathrm{in} . / \mathrm{min}$.(see Program, :Table 10.1). The applied load was measured using a load cell. The change in length of the specimen and the axial and transverse strains were recorded using LVDTs and strain gages, respectively.

## Test Specimens

The test specimen was first cut from the flexible pipe in the form of longitudinal strips 1.13 in . $x 9.7 \mathrm{in}$. The coupon was then machined to obtain the dog bone shape shown in Fig. 10.1. No welds or seams were allowed in the specimens, except those specimens designated by Steel 36 - s e a m and Aluminum 36-seam, in which a seam lock was introduced at the middle of the specimens (see Figs. 10.12 and 10.13).

## Test Procedure

Longitudinal and transverse strain gages were installed on the specimen and the specimen was aligned so as to ensure its longitudinal axis to be coincident with the direction of the applied load. For plastic pipe specimens, the tensile force was applied at a constant rate-of-head speed of $0.05,0.510$ and 150 in. per minute until the specimen failed. For metal pipes, the tensile force was applied at a constant rate-of-head speed of 0.5 in . per minute until the specimen failed. The data acquisition system was used to continuously record both transverse strain and axial strain simultaneously. The applied tensile load, as well as the corresponding elongation of the specimen was also continuously recorded.

## Test Program

Table 10.1 presents the details of the testing program including the number of tests and the load rates.

### 10.3 Calculations

i) Ultimate tensile strength

The ultimate tensile strength, $\sigma_{\mathrm{u}}$, is calculated by dividing the maximum load at failure, $\mathrm{F}_{\mathrm{b}}$ in newtons (or pounds-force) by the original cross-sectional area, A of the specimen in square metres (or square inches).

## ii) Modulus of elasticity

First, a graph of stress versus strain of the specimen is plotted. The initial linear portion of the stressstrain curve is extended, and the modulus of elasticity, E is given by the slope of this straight line, which is calculated by dividing the difference in stress corresponding to any segment on the straight line by the corresponding difference in strain.

## iii) Poisson's ratio

The axial and transverse, strains obtained from the test are plotted against the applied load. Straight lines are drawn through each set of points for both the axial, $\varepsilon_{\mathrm{a}}$, and the transverse, $\varepsilon_{\mathrm{t}}$, strains. One of any section in the linear portion of the graph is selected and the change in strain is determined. Then, Poisson's ratio, $\mu$, was calculated using Eq. (10.1) shown below:

$$
\begin{equation*}
\mu=-(\text { change in transverse strain) } / \text { (change in axial strain) } \tag{10.1}
\end{equation*}
$$

### 10.4 Results and Observations

Experimental results are provided in Tables 10.2 a and 10.2 b for plastic and metal pipes, respectively, while, average values are summarized in Tables 10.3a and 10.3b. Photographs of the specimens during testing are presented in Figs. 10.2 to 10.13.

The curves of the stress versus longitudinal and transverse strains for the various loading rates considered are presented in Fig. 10.14 for ADS 48", Fig. 10.15 for ADS 36", 10.16 for Hancor 36" Fig. 10.17 for PVC 36", Fig. 10.18 for Steel 36"and Aluminum 36"and Fig. 10.19 for the seam locks of steel and aluminum pipes.

From these results, the following observations are made:
(a) Results for ADS 48 show scatter and seem inconsistent (see Table 10.3a). This is due to the difficulty of measuring the thickness of the specimen due to the surface irregularities of the pipe wall from which the specimens were cut.
(b) For HDPE pipes, the modulus of elasticity generally increased as the loading rate increased.
(c) The maximum average value of the modulus of elasticity (E) achieved by ADS 48" was 100 ksi, whereas it attained 154 ksi and 147 ksi for ADS $36^{\prime \prime}$ and Hancor 36", respectively. For PVC $36^{\prime \prime}$, the maximum average modulus of elasticity was 451 ksi. For $0.5 \mathrm{in} . / \mathrm{min}$. the average values of E were $69 \mathrm{ksi}, 96 \mathrm{ksi}, 117 \mathrm{ksi}$ and 381 ksi for ADS 48", ADS 36", Hancor 36" and PVC $36^{\prime \prime}$, respectively.
(d) For HDPE pipes, the maximum stress increased as the loading rate increased. It varied between 3.11 ksi and 4.63 ksi for ADS $48^{\prime \prime}$, 2.80 ksi and 4.78 ksi for ADS 36 ", and 2.97 ksi and 4.79 ksi for Hancor 36. For PVC 36" pipe, the average maximum stress did not vary noticeably with the loading rate (between 5.18 and 6.64 ksi ). For a loading rate of $0.5 \mathrm{in} . / \mathrm{min}$, the values of the maximum stress achieved by ADS 48", ADS 36" Hancor $36 "$ and PVC 36 " are, respectively, $3.47 \mathrm{ksi}, 3.48 \mathrm{ksi}, 3.59 \mathrm{ksi}$ and 6.02 ksi .
(e) Steel and Aluminum achieved an average maximum tensile stress of 55.8 ksi and 33.1 ksi, respectively, and an average modulus of elasticity of $25,028 \mathrm{ksi}$ and $9,272 \mathrm{ksi}$, respectively.
(f) The apparent maximum stresses achieved by the Steel and Aluminum seam lock specimens are 8.40 ksi and 4.00 ksi , respectively. The apparent modulus of elasticity is 1,150 ksi for Steel seam lock and 687 ksi for Aluminum seam lock.

### 10.5 Conclusions

The following conclusions can be formulated:
(a) The moduli of elasticity of the different pipes are within the range of values specified by the AASHTO code.
(b) The tensile strengths of the different pipes are in conformity with the AASHTO code.

Table 10.1-Dog Bone Tension Test Type C Program
$\left.\begin{array}{lcc}\hline \text { Type of Pipe } & \begin{array}{c}\text { Load Rate } \\ \text { (in/min.) }\end{array} & \text { Number of Tests } \\ \hline \text { ADS 48 } & 0.05\end{array}\right)$

Table 10.2a - Experimental Results for Plastic Pipes

| Type | Specimen | Width | Thickness in | Cross section in $^{2}$ | Maximum force kips | $\begin{array}{\|c\|} \hline \begin{array}{c} \text { Maximum } \\ \text { stress } \\ \text { ksi } \end{array} \\ \hline \end{array}$ | Strain at max stress | Modulus of elasticity ksi |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| ADS 48 | $\begin{aligned} & 0.05-1 \\ & 0.05-2 \end{aligned}$ | $\begin{aligned} & 0.735 \\ & 0.740 \end{aligned}$ | $\begin{aligned} & 0.339 \\ & 0.339 \end{aligned}$ | $\begin{aligned} & 0.2495 \\ & 0.2509 \end{aligned}$ | $\begin{aligned} & 0.752 \\ & 0.804 \end{aligned}$ | $\begin{array}{r} 3.01 \\ 3.20 \\ \hline \end{array}$ | $\begin{aligned} & 0.121 \\ & 0.142 \end{aligned}$ | $\begin{aligned} & 96 \\ & 69 \end{aligned}$ |
|  | $\begin{aligned} & 0.5-1 \\ & 0.5-2 \end{aligned}$ | $\begin{aligned} & 0.742 \\ & 0.732 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.328 \\ & 0.339 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.2437 \\ & 0.2484 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.837 \\ & 0.870 \end{aligned}$ | $\begin{aligned} & 3.43 \\ & 3.50 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.129 \\ & 0.119 \end{aligned}$ | $\begin{aligned} & 71 \\ & 66 \end{aligned}$ |
|  | $\begin{aligned} & 10-1 \\ & 10-2 \end{aligned}$ | $\begin{aligned} & 0.737 \\ & 0.737 \\ & \hline \end{aligned}$ | $\begin{array}{r} 0.336 \\ 0.337 \\ \hline \end{array}$ | $\begin{aligned} & 0.2475 \\ & 0.2480 \\ & \hline \end{aligned}$ | $\begin{aligned} & 1.056 \\ & 1.103 \\ & \hline \end{aligned}$ | $\begin{aligned} & 4.27 \\ & 4.45 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.113 \\ & 0.105 \end{aligned}$ | $\begin{gathered} 97 \\ 103 \\ \hline \end{gathered}$ |
|  | $\begin{aligned} & 150-1 \\ & 150-2 \end{aligned}$ | $\begin{aligned} & 0.733 \\ & 0.733 \end{aligned}$ | $\begin{aligned} & 0.328 \\ & 0.335 \end{aligned}$ | $\begin{aligned} & 0.2403 \\ & 0.2457 \end{aligned}$ | $\begin{array}{\|l\|} \hline 1.063 \\ \hline 1.189 \\ \hline \end{array}$ | $\begin{aligned} & 4.42 \\ & 4.84 \end{aligned}$ | $\begin{aligned} & 0.100 \\ & 0.090 \end{aligned}$ | $\begin{gathered} 91 \\ 66.00 \end{gathered}$ |
| ADS 36 | $\begin{array}{r} 0.05-1 \\ 0.05-2 \\ \hline \end{array}$ | $\begin{aligned} & 0.742 \\ & 0.747 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.115 \\ & 0.125 \\ & \hline \end{aligned}$ | $\begin{array}{r} 0.0850 \\ 0.0933 \\ \hline \end{array}$ | $\begin{aligned} & 0.250 \\ & 0.248 \\ & \hline \end{aligned}$ | $\begin{aligned} & 2.94 \\ & 2.66 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.073 \\ & 0.091 \\ & \hline \end{aligned}$ | $\begin{aligned} & 77 \\ & 52 \end{aligned}$ |
|  | $\begin{aligned} & 0.5-1 \\ & 0.5-2 \\ & \hline \end{aligned}$ | $\begin{array}{r} 0.751 \\ 0.750 \\ \hline \end{array}$ | $\begin{aligned} & 0.117 \\ & 0.116 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.0875 \\ & 0.0871 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.312 \\ & 0.296 \end{aligned}$ | $\begin{array}{r} 3.56 \\ 3.40 \\ \hline \end{array}$ | $\begin{aligned} & 0.086 \\ & 0.084 \end{aligned}$ | 101 90 |
|  | $\begin{aligned} & 10-1 \\ & 10-2 \end{aligned}$ | $\begin{aligned} & 0.753 \\ & 0.752 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.125 \\ & 0.119 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.0939 \\ & 0.0891 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.353 \\ & 0.354 \end{aligned}$ | $\begin{aligned} & 3.76 \\ & 3.97 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.072 \\ & 0.076 \end{aligned}$ | $\begin{array}{r} 153 \\ 154 \\ \hline \end{array}$ |
|  | $\begin{aligned} & 150-1 \\ & 150-2 \\ & 150-3 \end{aligned}$ | $\begin{aligned} & 0.759 \\ & 0.746 \\ & 0.747 \end{aligned}$ | $\begin{aligned} & 0.111 \\ & 0.117 \\ & 0.110 \end{aligned}$ | $\begin{aligned} & 0.0840 \\ & 0.0876 \\ & 0.0823 \end{aligned}$ | $\begin{aligned} & 0.400 \\ & 0.423 \\ & 0.390 \end{aligned}$ | $\begin{aligned} & 4.76 \\ & 4.83 \\ & 4.74 \end{aligned}$ | $\begin{aligned} & 0.046 \\ & 0.055 \\ & 0.046 \end{aligned}$ | 141 110 126 |
| Hancor 36 | $\begin{aligned} & 0.05-1 \\ & 0.05-2 \\ & \hline \end{aligned}$ | $\begin{array}{r} 0.767 \\ 0.757 \\ \hline \end{array}$ | $\begin{aligned} & 0.093 \\ & 0.095 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.0710 \\ & 0.0719 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.212 \\ & 0.213 \\ & \hline \end{aligned}$ | $\begin{array}{r} 2.99 \\ 2.96 \\ \hline \end{array}$ | $\begin{aligned} & 0.080 \\ & 0.080 \\ & \hline \end{aligned}$ | $\begin{aligned} & 102 \\ & 107 \end{aligned}$ |
|  | $\begin{aligned} & 0.5-1 \\ & 0.5-2 \\ & \hline \end{aligned}$ | $\begin{aligned} & \hline 0.753 \\ & 0.750 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.111 \\ & 0.111 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.0836 \\ & 0.0836 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.292 \\ & 0.308 \\ & \hline \end{aligned}$ | $\begin{array}{r} 3.49 \\ 3.69 \\ \hline \end{array}$ | $\begin{aligned} & 0.089 \\ & 0.102 \\ & \hline \end{aligned}$ | $\begin{aligned} & 109 \\ & 125 \\ & \hline \end{aligned}$ |
|  | $\begin{aligned} & 10-1 \\ & 10-2 \end{aligned}$ | $\begin{array}{r} 0.753 \\ 0.759 \\ \hline \end{array}$ | $\begin{aligned} & 0.108 \\ & 0.126 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.0812 \\ & 0.0953 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.365 \\ & 0.400 \end{aligned}$ | $\begin{array}{r} 4.49 \\ 4.20 \\ \hline \end{array}$ | $\begin{aligned} & 0.060 \\ & 0.073 \end{aligned}$ | $\begin{array}{r} 136 \\ 147 \end{array}$ |
|  | $\begin{aligned} & 150-1 \\ & 150-2 \end{aligned}$ | $\begin{aligned} & 0.749 \\ & 0.753 \end{aligned}$ | $\begin{aligned} & 0.117 \\ & 0.111 \end{aligned}$ | $\begin{aligned} & 0.0879 \\ & 0.0836 \end{aligned}$ | $\begin{aligned} & 0.411 \\ & 0.410 \end{aligned}$ | $\begin{aligned} & 4.68 \\ & 4.90 \end{aligned}$ | $\begin{aligned} & 0.050 \\ & 0.049 \end{aligned}$ | $\begin{aligned} & 154 \\ & 141 \end{aligned}$ |
| PVC 36 | $\begin{aligned} & 0.05-1 \\ & 0.05-2 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.740 \\ & 0.750 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.157 \\ & 0.181 \\ & \hline \end{aligned}$ | $\begin{array}{r} 0.1160 \\ 0.1358 \\ \hline \end{array}$ | $\begin{aligned} & 0.760 \\ & 0.844 \\ & \hline \end{aligned}$ | $\begin{array}{r} 6.55 \\ 6.21 \\ \hline \end{array}$ | $\begin{aligned} & 0.025 \\ & 0.029 \end{aligned}$ | $\begin{array}{r} 398 \\ 379 \\ \hline \end{array}$ |
|  | $\begin{aligned} & 0.5-1 \\ & 0.5-2 \\ & 0.5-3 \\ & \hline \end{aligned}$ | $\begin{array}{r} 0.745 \\ 0.750 \\ 0.747 \\ \hline \end{array}$ | $\begin{array}{r} 0.184 \\ 0.179 \\ 0.187 \\ \hline \end{array}$ | 0.1370 <br> 0.1343 <br> 0.1400 | $\begin{aligned} & 0.768 \\ & 0.739 \\ & 0.974 \end{aligned}$ | $\begin{aligned} & 5.61 \\ & 5.50 \\ & 6.96 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.017 \\ & 0.026 \\ & 0.027 \end{aligned}$ | $\begin{array}{r} 402 \\ 343 \\ 397 \\ \hline \end{array}$ |
|  | $\begin{aligned} & 10-1 \\ & 10-2 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.758 \\ & 0.752 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.189 \\ & 0.191 \\ & \hline \end{aligned}$ | $\begin{aligned} & 0.1429 \\ & 0.1438 \\ & \hline \end{aligned}$ | $\begin{aligned} & 1.111 \\ & 0.792 \\ & \hline \end{aligned}$ | $\begin{array}{r} 7.77 \\ 5.51 \\ \hline \end{array}$ | $\begin{aligned} & 0.029 \\ & 0.017 \end{aligned}$ | $\begin{aligned} & 497 \\ & 406 \end{aligned}$ |
|  | $\begin{aligned} & 150-1 \\ & 150-2 \end{aligned}$ | $\begin{aligned} & 0.748 \\ & 0.752 \end{aligned}$ | $\begin{aligned} & 0.186 \\ & 0.181 \end{aligned}$ | $\begin{aligned} & 0.1391 \\ & 0.1365 \end{aligned}$ | $\begin{aligned} & 0.638 \\ & 0.789 \end{aligned}$ | $\begin{aligned} & 4.59 \\ & 5.78 \end{aligned}$ | $\begin{aligned} & 0.013 \\ & 0.015 \end{aligned}$ | $\begin{aligned} & 367 \\ & 378 \end{aligned}$ |

Table 10.2b - Experimental Results for Metal Pipes

| Type | Specimen | Width <br> in | Thickness in | Cross section $i n^{2}$ | Maximum force kips | Maximum stress ksi | Strain at max stress | Modulus of elasticity ksi |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| STEEL 36 | $\begin{aligned} & 0.5-1 \\ & 0.5-2 \\ & 0.5-3 \end{aligned}$ | $\begin{aligned} & 0.756 \\ & 0.753 \\ & 0.763 \end{aligned}$ | $\begin{aligned} & 0.070 \\ & 0.072 \\ & 0.076 \end{aligned}$ | $\begin{aligned} & 0.0532 \\ & 0.0540 \\ & 0.0577 \end{aligned}$ | $\begin{aligned} & 2.941 \\ & 3.001 \\ & 3.259 \end{aligned}$ | $\begin{aligned} & 55.24 \\ & 55.62 \\ & 56.51 \end{aligned}$ | $\begin{aligned} & 0.037 \\ & 0.074 \\ & 0.064 \end{aligned}$ | $\begin{aligned} & 23,354 \\ & 25,408 \\ & 26,322 \end{aligned}$ |
| ALU 36 | $\begin{aligned} & 0.5-1 \\ & 0.5-2 \\ & 0.5-3 \end{aligned}$ | $\begin{aligned} & 0.747 \\ & 0.753 \\ & 0.753 \end{aligned}$ | $\begin{aligned} & 0.076 \\ & 0.070 \\ & 0.072 \end{aligned}$ | $\begin{aligned} & 0.0570 \\ & 0.0530 \\ & 0.0546 \end{aligned}$ | $\begin{aligned} & 1.778 \\ & 1.819 \\ & 1.841 \end{aligned}$ | $\begin{aligned} & 31.18 \\ & 34.28 \\ & 33.74 \end{aligned}$ | $\begin{aligned} & 0.033 \\ & 0.032 \\ & 0.029 \end{aligned}$ | $\begin{aligned} & 8,641 \\ & 9,913 \\ & 9,262 \end{aligned}$ |
| SEAM STEEL 36 | $\begin{aligned} & 0.5-1 \\ & 0.5-2 \end{aligned}$ | $\begin{aligned} & 1.025 \\ & 1.019 \end{aligned}$ | $\begin{gathered} 0.071 \\ 0.072 \end{gathered}$ | $\begin{aligned} & 0.0731 \\ & 0.0730 \end{aligned}$ | $\begin{aligned} & 0.619 \\ & 0.608 \end{aligned}$ | $\begin{aligned} & 8.48 \\ & 8.32 \end{aligned}$ | $\begin{aligned} & 0.013 \\ & 0.011 \end{aligned}$ | $\begin{gathered} 954 \\ 1,255 \end{gathered}$ |
| $\begin{aligned} & \text { SEAM } \\ & \text { ALU } 36 \end{aligned}$ | $\begin{aligned} & 0.5-1 \\ & 0.5-2 \\ & 0.5-3 \end{aligned}$ | $\begin{aligned} & 1.041 \\ & 1.040 \\ & 1.024 \end{aligned}$ | $\begin{aligned} & 0.073 \\ & 0.072 \\ & 0.074 \end{aligned}$ | $\begin{aligned} & 0.0763 \\ & 0.0753 \\ & 0.0754 \end{aligned}$ | $\begin{gathered} 0.312 \\ 0.297 \\ 0.299^{\text {(b) }} \end{gathered}$ | $\begin{gathered} 4.09 \\ 3.95 \\ 3.97^{(b)} \end{gathered}$ | $\begin{aligned} & 0.024 \\ & 0.013 \\ & 0.010 \end{aligned}$ | $\begin{aligned} & 791 \\ & 562 \\ & 709 \end{aligned}$ |

Note: (b) Correponding to first yield

Table 10.3a - Experimental Average Results for Plastic Pipes

| Make | Specimen | Maximum <br> stress <br> ksi | Strain at <br> max stress <br> - | Modulus <br> of elasticity <br> ksi |
| :---: | :---: | :---: | :---: | :---: |
|  | 0.05 | 3.11 | 0.132 | 82 |
| ADS 48 | 0.5 | 3.47 | 0.124 | 69 |
|  | 10 | 4.36 | 0.109 | 100 |
| ADS 36 | 150 | 4.63 | 0.095 | 78 |
|  | 0.05 | 2.80 | 0.082 | 65 |
|  | 0.5 | 3.48 | 0.085 | 96 |
|  | 10 | 3.87 | 0.074 | 154 |
| Hancor 36 | 0.05 | 4.78 | 0.049 | 126 |
|  | 10 | 4.97 | 0.080 | 104 |
|  | 150 | 4.79 | 0.050 | 147 |
|  | 0.05 | 6.38 | 0.027 | 389 |
|  | 0.5 | 6.02 | 0.023 | 381 |
| PVC 36 | 10 | 6.64 | 0.023 | 451 |
|  | 150 | 5.18 | 0.014 | 372 |

Table 10.3b - Experimental Average Results for Metal Pipes

| Type | Specimen <br> $\therefore$ | Maximum <br> stress <br> ksi | Strain at <br> max stress <br> - | Modulus <br> of elasticity <br> ksi |
| :---: | :---: | :---: | :---: | :---: |
| STEEL 36 | 0.5 | 55.79 | 0.058 | 25,028 |
| ALU 36 | 0.5 | 33.07 | 0.031 | 9,272 |
| SEAM <br> STEEL 36 | 0.5 | 8.40 | 0.012 | 1,105 |
| SEAM <br> ALU 36 | 0.5 | 4.00 | 0.016 | 687 |

(a) Specimen cut in the form of longitudinal strip from the pipe

(b) Specimen cut into dogbone shape

(c) Instrumentation with strain gages


Fig. 10.1 - Details of Specimens


Fig. 10.2 - ADS 48 Specimen During Testing


Fig. 10.3 - ADS 48 Specimen Prior to Failure


Fig. 10.4 - ADS 36 Specimen During Testing


Fig. 10.5 - ADS 36 Specimen at Failure


Fig. 10.6 - Hancor 36 Specimen During Testing


Fig. 10.7 - Hancor 36 Specimen at Failure


Fig. 10.8 - PVC 36 Specimen During Testing


Fig. 10.9 - PVC 36 Specimen at Failure


Fig. 10.10 - Steel 30 Specimen at Failure


Fig. 10.11 - Aluminum Specimen During Testing


Fig. 10.12 - Aluminum Seam Specimen During Testing


Fig. 10.13 - Aluminum Seam Specimen at Failure



Fig.10.14a - Experimental Stress versus Strain Curves for ADS48 at a Loading Rate of : (A) $0.05 \mathrm{in} . / \mathrm{min}$. and (B) $0.5 \mathrm{in} . / \mathrm{min}$.



Strain (\%)

Fig. 10.14b - Experimental Stress versus Strain Curves for ADS48 at a Loading Rate of : (C) $10 \mathrm{in} . / \mathrm{min}$. and (D) $150 \mathrm{in} . / \mathrm{min}$.



Fig.10.15a - Experimental Stress versus Strain Curves ADS36 and a Loading Rate of : (A) $0.05 \mathrm{in} . / \mathrm{min}$. and (B) $0.5 \mathrm{in} . / \mathrm{min}$.


Fig.10-15b - Experimental Stress versus Strain Curves ADS36 at a Loading Rate of : (C) $10 \mathrm{in} . / \mathrm{min}$. and (D) $150 \mathrm{in} . / \mathrm{min}$.


Fig.10-16a - Experimental Stress versus Strain Curves HANCOR 36 at a Loading Rate of : (A) $0.05 \mathrm{in} . / \mathrm{min}$. and (B) $0.5 \mathrm{in} . / \mathrm{min}$.


Fig.10-16b - Experimental Stress versus Strain Curves HANCOR 36 at a Loading Rate of : (C) $10 \mathrm{in} . / \mathrm{min}$. and (D) $150 \mathrm{in} . / \mathrm{min}$.



Fig.10-17a - Experimental Stress versus Strain Curves for PVC36 at a Loading Rate of : (A) $0.05 \mathrm{in} . / \mathrm{min}$. and (B) $0.5 \mathrm{in} . / \mathrm{min}$.


Fig.10-17b - Experimental Stress versus Strain Curves for PVC36 at a Loading Rate of : (C) $10 \mathrm{in} . / \mathrm{min}$. and (D) $150 \mathrm{in} . / \mathrm{min}$.



Fig.10-18 - Experimental Stress versus Strain Curves for a Loading Rate of $0.5 \mathrm{in} . / \mathrm{min}$. for : (A) STEEL36 and (B) ALU36



Fig. 10-19 - Experimental Stress versus Strain Curves
for a Loading Rate of $0.5 \mathrm{in} . / \mathrm{min}$. for : (A) SEAM-STEEL36 and (B) SEAM-ALU36

## Chapter 11: Environmental Stress Cracking Test

### 11.1 Scope and Objective

"Stress-crack" is defined in ASTM D 1693, Standard Test Method for Environmental StressCracking of Ethylene Plastics, as an external and internal rupture in a plastic caused by tensile stresses smaller than its short-time mechanical strength. In the presence of an active environmental agent, cracking may occur under stresses that plastic resins might ordinarily resist indefinitely. This phenomenon is commonly referred to as "environmental stress cracking". Environmental stress cracking is a property that is highly dependent upon the .nature and level of the stresses applied and on the thermal history of the specimen (Decoste, 1951). Environmental stress cracking has been found to occur most readily under high local multi-axial stresses that are developed through the :introduction of a controlled imperfection (Hopkins, et al. 1950).

The objective of this test is to investigate the response of stressed and unstressed HDPE specimens with an imperfection on the specimen surface. The active agent, 100 percent Igepal, CO-630 preheated to $50^{\circ} \mathrm{C} \pm 2^{\circ} \mathrm{C}$, is used as specified by AASHTO M 294, Section 9.4.4.

### 11.2. Experimental Program Apparatus

AASHTO M 294 requires that the specimen used must consist of a 90 -degree arc length of pipe, that it is bent to shorten the inside chord length $20 \pm 1$ percent and retained in this position by a suitable holding device. The external force thus induced from this device is applied to both test conditions. Fig 11.1 shows the configuration of the holding device used to hold the specimen when it is bent. A controlled imperfection (notch) is made on the specimen with a specially designed jig.

Figure 11.2 shows the specimen, which has been exposed to the active agent. A digital caliper with an accuracy of 0.001 inches $(0.02 \mathrm{~mm}$.) is used to measure the propagation of the notch. A measuring tab is used to measure the lengths of each of the specimens.

## Test specimen

Fig. 11.3 shows the configuration of the specimen cut from the ADS $48^{\prime \prime}$ having a length of 20 inches. For the specimens of ADS 36, and Hancor 36" pipes, the length of the specimens was 15.5 inches. The imperfection made on the specimen surface was a notch of $1-\mathrm{in}$. long, and $1 / 8-\mathrm{in}$. deep. Fig. 11.4 shows the location of a notch.

## Test Procedure

The specimens were tested under two conditions: in ambient air and under immersion in the active agent (100 percent Igepal CO-630). In ambient air, the notch size and chord lengths are measured prior to the application of the external force. The specimen was then subjected to the external force and the change in notch and chord lengths were recorded. The external force was maintained for 24 hours and the notch and chord lengths were then measured again at the end of the testing period. The external force was then released and both lengths were recorded immediately. In the second condition, a new specimen was used. Prior to the immersion in the active agent, measurements of the notch and chord lengths were taken before and after external load application. Then, the specimen was immersed completely in the bath of the preheated agent at $50^{\circ} \mathrm{C} \pm 2^{\circ} \mathrm{C}$. This temperature was maintained for 24 hours, then the specimen was removed and the notch and chord lengths were measured. The external force was then released and the measurements were taken again. Note that the measurements taken during the load application were recorded as a bound state; while in the case without load application, the measurements were recorded as an unbound state. Figs. 11.5 and 11.6 show the notch and chord lengths measurements. Figs. 11.7 to 11.19 show the preparation of the specimens, locations of notch, and measurements of the notch lengths.

## Test Program

Table 11.1 gives details of the testing program for the environmental stress cracking test.

### 11.3 Calculations

Environmental stress cracking is evaluated by means of a "relative deformation," which is different from that of ASTM D-1693. "Relative deformation" is defined as the difference
between the deformation of a length based on tests in ambient air and that based on tests under the active agent ( 100 percent Igepal CO-630).
i) Deformation

Deformation (\%) $=($ Length after the test $)-($ Length before the test $) \times 100$
(Length before the test)
ii) Relative Deformation

Relative deformation (\%) = (Deformation in active agent) $-($ Deformation in air $)$

### 11.4 Results and Observations

Any crack in the specimens visible to an observer with normal eyesight should be interpreted as the failure of the entire specimens. Tables 11.2 to 11.4 present the data obtained from the test for the ADS 48", ADS 36", and Hancor $36^{\prime \prime}$ series under both air and active conditions. Table 11.5 presents observations on the specimens during and after the tests.

The following observations are made:
(a) During the 24-hour exposure to air, the lengths of the notch and the chord of the bounded specimens did not vary (Tables 11.2 to 11.4).
(b) After 24 hours in Igepal solution, the notch length variation was negligible for ADS $36^{\prime \prime}$ and Hancor $36^{\prime \prime}$ under bound conditions. However, for ADS 48", the variation was in average $31.7 \%$. This high percentage was due to test \#1 where the notch length variation reached $56 \%$ (see Table 11.2).
(c) Comparing the notch length of specimens in a 24-hour Igepal solution after release of the load, with the original pristine specimen revealed that the change in length varied between $4.93 \%$ and $10.25 \%$ for ADS 36" and Hancor 36" (see Tables 11.3 and 11.4), whereas it reached $17.38 \%$ for ADS 48" (see Table 11.2).
(d) One of the two ADS $48^{\prime \prime}$ specimens showed major cracking after a 24-hour exposure in Igepal.

### 11.5 Conclusions

The following conclusions can be formulated:
(a) The 36-in. diameter HDPE pipes behaved satisfactorily under ESCR tests.
(b) One of the two specimens of the $48-\mathrm{in}$. diameter HDPE pipe failed the ESCR test under the conditions described in this study.

Table 11.1 - ESCR Test Program

| Pipe Type | Environment | Temperature | Number of specimens |
| :--- | :--- | :--- | :---: |
| ADS 48 | Air | Ambient | 2 |
|  | Igepal | $50 \%$ | 2 |
| ADS 36 | Air | Ambient | 2 |
|  | Igepal | $50 \%$ | 2 |
| HANCOR 36 | Air | Ambient | 2 |
|  | Igepal | $50 \%$ | 2 |

Table 11.2 - Environmental Test Measurements for ADS 48

| Environment -. Specimen (Temp.) | Designation | Unbound Condition |  | Bound Condition |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Notch (in.) | Chord (in.) | Notch <br> (in.) | Chord <br> (in.) |
| $\begin{array}{r} \mathrm{AlR}_{-} 1 \\ \left(22.8^{\circ} \mathrm{C}\right) \end{array}$ | Before Test | 1.031 | 33.82 | 1.0090 | 27.04 |
|  | After 24H | $1.044^{(a)}$ | $30.71{ }^{(a)}$ | 1.0235 | 26.97 |
|  | Deformation (\%) | +1.26 | -9.20 | +1.44 | -0.26 |
| ATR-2$\left(22.8^{\circ} \mathrm{C}\right)$ | Before Test | 0.9350 | 33.78 | 0.9425 | 27.01 |
|  | After 24H | $0.9540^{(2)}$ | $30.51{ }^{(a)}$ | 0.9260 | 27.05 |
|  | Deformation (\%) | +2.03 | -9.68 | -1.75 | +0.15 |
| AIR-average | Deformation (\%) | +1.65 | -9.44 | -0.16 | -0.05 |
| $\begin{aligned} & \text { IGEPAL-1 } \\ & \left(50.1^{\circ} \mathrm{C}\right) \end{aligned}$ | Before Test | 0.9470 | 34.13 | 0.941 | 27.32 |
|  | After 24H | $1.2065{ }^{(a)}$ | $31.06^{(\mathrm{a})}$ | 1.4680 | 27.32 |
|  | Deformation (\%) | $+27.40^{(\mathrm{b})}$ | -9.00 | $+56.00^{(b)}$ | 0.00 |
| $\begin{aligned} & \text { IGEPAL-2 } \\ & \left(51.6^{\circ} \mathrm{C}\right) \end{aligned}$ | Before Test | 1.0035 | 33.90 | 0.9815 | 27.01 |
|  | After 24H | $1.0775^{(\mathrm{a})}$ | $30.43^{(a)}$ | 1.0545 | 26.73 |
|  | Deformation (\%) | +7.37 | -10.24 | +7.44 | -3.74 |
| IGEPAL-average | Deformation (\%) | +17.38 | -9.62 | +31.72 | -1.87 |
| Notes: ${ }^{(a)}$ After release at end of test <br> (b) Crack along notch (see Figs. 11 |  | and 11.12) |  |  |  |

Table 11.3 - Environmental Test Length Measurements for ADS 36

| Environment <br> - Specimen \# (Temp.) | Designation | Unbound Condition |  | Bound Condition |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Notch <br> (in.) | Chord <br> (in.) | Notch (in.) | Chord <br> (in.) |
| $\begin{aligned} & \hline \text { AIR-1 } \\ & \left(22.5^{\circ} \mathrm{C}\right) \end{aligned}$ | Before Test | 1.0255 | 25.55 | 0.9415 | 20.470 |
|  | After 24H | 0.9795 | 22.362 | 0.9570 | 20.394 |
|  | Deformation (\%) | -4.49 | -12.48 | 1.65 | -0.37 |
| $\begin{aligned} & \hline \text { AIR-2 } \\ & \left(25.2^{\circ} \mathrm{C}\right) \end{aligned}$ | Before Test | 0.9980 | 24.016 | 0.9580 | 19.134 |
|  | After 24H | 0.9615 | 21.339 | 0.9425 | 19.173 |
|  | Deformation (\%) | -3.66 | -11.15 | -1.62 | 0.20 |
| AIR- average | Deformation (\%) | -4.08 | -11.81 | 0.015 | -0.09 |
| $\begin{aligned} & \text { IGEPAL-1 } \\ & \left(50.1^{\circ} \mathrm{C}\right) \end{aligned}$ | Before Test | 1.0005 | 25.470 | 0.9000 | 20.350 |
|  | After 24H | 0.9470 | 22.165 | 0.8340 | 20.276 |
|  | Deformation (\%) | -5.35 | -12.98 | -7.33 | -0.36 |
| $\begin{aligned} & \text { IGEPAL-2 } \\ & \left(51.6^{\circ} \mathrm{C}\right) \end{aligned}$ | Before Test | 0.9075 | 24.331 | 0.9510 | 19.488 |
|  | After 24H | 1.0455 | 22.087 | 0.9760 | 19.450 |
|  | Deformation (\%) | 15.21 | -9.22 | 2.63 | -1.9 |
| IGEPAL-average | Deformation (\%) | 4.93 | -11.1 | -2.35 | -1.13 |

Table 11.4 - Environmental Test Length Measurements for HANCOR 36

| Environment <br> - Specimen (Temp.) | Designation | Unbound Condition |  | Bound Condition |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Notch (in.) | Chord (in.) | Notch (in.) | Chord <br> (in.) |
| $\begin{aligned} & \text { AIR-1 } \\ & (-) \end{aligned}$ | Before Test | 0.3170 | 25.20 | 0.3245 | 20.31 |
|  | After 24H | $0.3065{ }^{(a)}$ | $22.36{ }^{\text {(a) }}$ | 0.3100 | 20.27 |
|  | Deformation (\%) | -3.31 | -11.27 | -4.47 | -0.20 |
| AIR-2 | Before Test | 0.2655 | 25.59 | ---- | 20.47 |
| (-) | After 24H | 0.2935 | $22.72^{(a)}$ | ---- | 20.47 |
|  | Deformation (\%) | +10.55 | -11.22 | --- | 0.00 |
| AIR- Average | Deformation (\%) | +3.62 | -11.24 | -4.47 | -0.10 |
| IGEPAL-1 | Before Test | 0.3410 | 25.47 | 0.2960 | 20.71 |
| $(-)$ | After 24H | 0.3150 | $22.60{ }^{\text {(a) }}$ | 0.2940 | 20.63 |
|  | Deformation (\%) | -7.62 | -11.27 | -0.68 | -0.39 |
| IGEPAL-2 | Before Test | 0.3225 | 25.20 | 0.2830 | 20.39 |
| (-) | After 24H | 0.2810 | $22.13^{(a)}$ | - ---- | 20.35 |
|  | Deformation (\%) | -12.87 | -12.18 | -- | -0.20 |
| IGEPAL-average | Deformation (\%) | -10.25 | -11.72 | -0.68 | -0.30 |

Notes: (a) After release of load at end of test

Table 11.5-Observations

| Pipe Type. | Environment | Observations | ASTM D1693 |
| :---: | :---: | :---: | :---: |
| ADS 48 | Air | No cracks or crazing | Pass |
|  | Igepal | Long cracks, extension of controlled notch <br> Depressions and cracks on the surface, and through wall open cracks in the 1st of the two tests. (Fig. 11. 12) | Failure for 50\% of tests Time $=24$ hours |
| ADS 36 | Air | No cracks or crazing | Pass |
|  | Igepal | Cracks but extension of controlled imperfection | Pass |
| HANCOR 36 | Air | No cracks or crazing | Pass |
|  | Igepal | Cracks but extension of controlled imperfection | Pass. |



Fig. 11.1 - Configuration of the Holding Device.


Fig. 11.2 - Specimen After Exposure to the Active Agent


Fig. 11.3 - Specimen Configuration and Dimension for ADS 48 Pipe


Fig. 11.4-Location of the Notch on the Specimen Crown.


Fig. 11.5 - Measurement of a Notch Length by the Digital Caliper


Fig. 11.6 - Measurement of Chord Length


Fig. 11.7 - Preparation of Notch for ADS 48 Specimen


Fig. 11.8-Close-up of Notch Before Test for ADS 48 Specimen


Fig. 11.9 - Measurement of Notch Length Before Test


Fig. 11.10 - Preparation of Notch for ADS 48 Specimen
(a) Test \#1

(b) Test \#2


Fig. 11.11 - View of Notch after Test Under Igepal for Bound ADS 48 Specimen


Fig. 11.12 - View of Notch after Test Under Igepal for Unbound ADS 48 Specimen


Fig. 11.13-Close-up of Notch before Test for ADS 48 Specimen


Fig. 11.14 - View of Notch after Test under 24 H Igepal for Bound ADS 36 Specimen


Fig. 11.15 - View of Notch after Test Under 24 H Air for Bound ADS 36 Specimen


Fig. 11.16 - Close-up of Notch before Test for Hancor 36 Specimen


Fig. 11.17 - View of Notch after Test Under 24 H Igepal for Bound Hancor 36 Specimen 2


Fig. 11.18 - View of Notch after Test Under 24 H Igepal for Bound Hancor 36 Specimen 4


Fig. 11.19-Close-up View of Notch after Test Under 24 H Igepal for Bound Hancor 36 Specimen 4

## Chapter 12: Conclusions

This study describes the laboratory work performed and presents results for ten different tests carried out in this investigation. The main objective of the laboratory work was to evaluate and characterize, under laboratory conditions, the performance and properties of the different plastic and metal pipes considered in the study.

The following are the findings of 'the laboratory investigation in this study:
(a) Visual Inspections of the different pipes indicated that HDPE, PVC, and metal pipes generally meet the requirements of AASHTO-M294, ASTM F949, and ASSHTO-T249. However, visible creasing at the surface of inside and outside walls, as well as irregular surface at certain locations around the circumference of the bell and spigot joint, were observed in ADS 48. Also the contact length of the seam lap in the case of aluminum and its distance from the adjacent ribs for both types of metal pipes do not conform to AASHTO T249 requirements. These irregularities, even though they seem not to have an apparent incidence on structural performance, may require improvement.
(b) Beam Test results indicated that for the plastic pipes, the valley longitudinal bending strains were greater than the crown longitudinal bending strains. For the metal pipes, the longitudinal bending strains in the ribs were greater than the longitudinal bending strains in the wall (valley) between the ribs. For a vertical bottom deflection of $1 \%$ of the span length, the longitudinal bending strain ranged from $114 \mu \varepsilon$ (i.e., 12.5 psi ) to $1000 \mu \varepsilon$ (i.e., 110 psi ) for HDPE, it reached $600 \mu \varepsilon$ (i.e., 240 psi ) for PVC and $200 \mu \varepsilon$ (i.e., 5800 psi for steel and 2000 psi for aluminum) for metal pipes.
(c) Parallel Plate Test results indicated that for $5 \%$ vertical deflection and a loading rate of $0.5 \mathrm{in} . / \mathrm{min}$., all the pipes achieved a pipe stiffness, PS, greater than the minimum specified by the Standards. They also revealed no sign of distress or buckling in the pipes for vertical deflections less than $15 \%$. Finally and as expected, the tests confirmed that for a
given vertical deflection, the HDPE pipe stiffness (PS). substantially decreased as the loading rate decreased and vice-versa.
(d) Flattening Test results indicated that all the HDPE pipes passed this test, since no splitting, cracking, breaking, or separation of ribs or seams, or both, were observed under, normal light with unaided eyes. The PVC specimens that could be flattened up to $60 \%$ vertical deflection without failure also passed the flattening test. However, a number of PVC pipe specimens ruptured before reaching the $60 \%$ limit.
(e) Curved Beam Test results indicated that time-independent pipe stiffness $K(0)$ is 2 to 3 times greater than the PS values determined by the parallel plate test for all the pipes and increase with the loading rate for HDPE pipes. For a vertical deflection of $5 \%$ diameter the tensile strain (stress) in the outer wall was approximately equal to $18,000 \mu \varepsilon$ (i.e., $1,980 \mathrm{psi}$ ) for all HDPE, $17,000 \mu \varepsilon$ (i.e., $6,800 \mathrm{psi}$ ) for PVC and $16,000 \mu \varepsilon$ (L e., 60 ksi for steel and 21 ksi for aluminum) for metal pipes.
(f) Joint Integrity Test results indicated that all the pipes behaved satisfactorily with no sign of cracks or excessive gaps up to $10 \%$ vertical deflection. The radial gaps and longitudinal openings were small and reached 1.5 in . and 0.75 in ., respectively, for $30 \%$ vertical deflection. The presence of a joint generally modified the PS of the pipe: it resulted in a $10 \%$ reduction of PS at $5 \%$ vertical deflection for HDPE ADS 48 and 36 inch diameter pipes, and in $23 \%$ and $37 \%$ increase of PS for 5\% vertical deflection for Hancor 36 and PVC, respectively.
(g) Type C tension tests (Small Dog bone with no welds) indicated that the tensile properties of the pipes, the modulus of elasticity, and the tensile strength, are within the range of values specified by the AASHTO code. Type A tension tests (Double Wall Dumbbell Shape), performed on ADS 48 only, underestimated the tensile strength of the D-wall-type pipes, such as ADS 48. Type B tension tests (Single Wall Dumbbell Shape) indicated that the seam behavior of the D-wall-type pipe under tensile stresses is satisfactory given the maximum strength achieved. Type D tension tests (Split Disk. Test) performed on all the
pipes indicated that the apparent tensile properties under split disk tests are lower than those under Type C tension tests on small dog bone specimen with no weld, but greater than those achieved on dumbbell shape specimens with welds for ADS 48.
(h) ESCR Tests performed on HDPE pipes indicated that the 36 inch-diameter HDPE pipes behaved satisfactorily under ESCR tests. For the 48 in-diameter HDPE pipe however, one of the two specimens failed the ESCR test under the conditions described in this study.

## REFERENCES

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M249-93: Standard Method of Test for Helical Lock Seam Corrugated Pipe.
T249-98 : Standard Specifications for Corrugated Polyethylene Pipe, 300- to $1200-\mathrm{mm}$ Diameter.

MP7-97 : Standard Specification for Corrugated Polyethylene Pipe, 1350 and 1500 mm Diameter.

M304-94: Standard Specification for Poly (Vinyl Chloride) (PVC) Profile Wall Drain Pipe and Fittings Based on Controlled Inside Diameter.

## ASTM Standards

F949-00: Standard Specification for Poly (Vinyl Chloride) (PVC) Corrugated Sewer Pipe With a Smooth Interior and Fittings.
F679-00: Specification for Poly (Vinyl Chloride) (PVC) Large Diameter Plastic Gravity Sewer Pipe and Fittings.

D2412 : Test Method for Determination of External: Loading Characteristics of Plastic Pipe by Parallel-Plate Loading.
D618: Practice for Conditioning Plastics and Electrical Insulating Materials for Testing.

D883: Terminology Relating to Plastics.
D2122: Test Method for Determining Dimensions of Thermoplastic Pipe and Fittings.
D2290: Standard Test Method for Apparent Tensile Strength of Ring or Tubular Plastics and Reinforced Plastics by Split Disk Method.

D1693: Environmental Stress Cracking of Ethylene Plastic
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